

FINAL REPORT

# HYDROGEN-OXYGEN CATALYTIC IGNITION AND THRUSTER INVESTIGATION

VOLUME II
HIGH PRESSURE THRUSTER EVALUATIONS

By
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PREPARED FOR

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P. N. HERR, Project Manager

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#### FOREWORD

This report was prepared by the Applied Technology Division of the TRW Systems Group, One Space Park, Redondo Beach, California, under Contract NAS 3-14347. The contract was administered by the Lewis Research Center of the National Aeronautics and Space Administration, Cleveland, Ohio. The NASA Project Manager for the contract was Mr. P. N. Herr of the Liquid Rocket Technology Branch. This is Volume II of the final report on the subject contract and summarizes the high pressure thruster technical effort conducted during the period from March 1971 to December 1971. Volume I describes the catalytic ignition and low chamber pressure thruster evaluations, and Volume II presents the results of the high pressure thruster evaluations begun during March 1971.

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#### **ABSTRACT**

The high pressure thruster effort was conducted with the major objective of demonstrating the TRW duct cooling concept with gaseous  $H_2/O_2$  propellant in a thruster operating at nominally 300 psia (2068 kN/m²) and 1500 lbf (6672N). The high response catalytic igniter concept reported in Volume I of this report was incorporated into the design. The analytical design methods for the duct cooling were proven in a series of tests with both ambient and reduced temperature propellants. Long duration tests as well as pulse mode tests demonstrated the feasibility of the concept. All tests were conducted with a scaling of the raised post triplet injector design previously demonstrated at 900 lbf in NASA MSFC and MSC demonstration firings. A series of environmental conditioned firings were also conducted to determine the effects of thermal soaks, atmospheric air and high humidity. This volume presents the results of the high pressure thruster evaluations. Volume I presented the results of the catalytic igniter and low pressure thruster evaluations, NASA CR-120869.

1. SUMMARY

#### SUMMARY

The experimental results of the high pressure, large thruster efforts of Contract NAS 3-14347 have demonstrated the feasibility of the TRW duct cooled concept as a viable design for gaseous  $\rm H_2/O_2$  attitude control thrusters. In this design cooling is provided by bypassing a percentage of the GH\_2 (20 to 30%) from the injector supply through a free floating cooling channel jacket around the primary combustion zone. This coolant is heated and ejected at high velocity along the nozzle walls at a contraction ratio of  $\sim$  1.5:1. The high velocity film of gas flows over the remainder of the nozzle and provides a thermal protection to the nozzle with no other cooling means. This results in a virtual adiabatic wall nozzle which requires design only for pressure loading. The 1500 lbf (6672 N) flightweight nozzle required wall thickness is on the order of 0.050 inch (0.127 cm). This results in a very lightweight thrust chamber structure

The cooling design approach results in a major reduction in thrust chamber fabrication complexity with only minimal sacrifice in performance. The resulting geometries are simple shapes and lend themselves to virtually text book analytic design with a minimum of required assumptions. As a result the predicted thrust chamber cycle life can be accepted with high confidence.

A detailed materials survey was conducted for the selection of the thrust chamber fabrication materials. A-286 was selected for the nozzle, and Berylco-10 was selected for the duct. The injector face elements were constructed from OFHC copper. All remaining external parts were fabricated from S.S.347.

The high response catalytic igniter developed earlier in the contract effort (described in Volume I) was incorporated into the design for the test firings. This unit has demonstrated the ability to repeatedly and reliably ignite the high pressure thruster in both steady state and pulse mode operation with total response times less than 50 ms.

A raised post triplet injector design was used for the experimental firings. This injector was an extrapolation from a 900 lbf (4000 N) injector developed earlier at TRW. This injector approach has proven the ability to approach 100% combustion performance over a wide range of mixture ratios. As a very high performing injector it also has a high face heat transfer rate due to the rapid, near face mixing of the propellants. The test results indicated that the pressure limit margin for the thrust chamber assembly was slightly in excess of 300 psia (2068.5  $\rm KN/m^2$ ) with the limit being the injector's durability. The limited scope of the program did not allow for detailed investigation of the injector to improve this limit.

The demonstrated altitude delivered performance at  $\varepsilon$  = 40:1 of the thrusters at a mixture ratio of 4:1 was  $\sim$ 432 lbf-sec/lbm (4248 N-sec/kg) with 32% of the total GH<sub>2</sub> flowing through the duct. This is slightly higher than the performance prediction model would give. Although the thruster experiments were not conducted with the duct at its thermal limit because of program limited hardware availability, the test data

strongly support a design point limit performance of 440 lbf-sec/lbm. At this performance level the thermal stress limits indicate a life in excess of  $10^6$  cycles.

The pulse mode results indicated that good pulse response could be achieved with the limiting factor being the catalytic igniter response of  $\sim 25\text{--}30~\text{ms}$ . Actual thruster responses of less than 50 ms were achieved with the use of non-optimum valving.

The resulting thermal test data also confirmed the nozzle wall temperature profile predictions. The peak interior nozzle wall temperature were uniform to within  $50^{\circ}\text{F}$  ( $10^{\circ}\text{C}$ ) at any axial location in the nozzle.

The major limitation in the design approach was found to be with the high performance triplet injector as fabricated from OFHC copper. Two minor erosion problems occurred during the test effort which necessitated repairs of the unit. The high heat fluxes near the face precluded actual experimental data from being generated at pressures greater than 300 psia (2068 kN/M²). The high response catalytic igniter performed quite well. The cold, high humidity environmental tests indicated an icing problem which precluded ignition when the catalyst surfaces were occluded. When the catalyst bed was isolated by low level purging the problem was eliminated.

The firing program as a whole was quite successful. A total of 146 tests were conducted with a number of thermal duration tests.

2. INTRODUCTION

#### 2. INTRODUCTION

The NASA is currently planning and studying the Space Shuttle vehicle as a means of achieving low launch costs for space exploration and utilization. The orbiter vehicle of the Space Shuttle has as one of its many projected uses the direct application as a space laboratory. The direct payload weight payoff function of the orbiter is, in part, determined by its propulsion system requirements. A unit of mass in the propulsion system is essentially a unit of mass less in payload. As a consequence, in order to achieve the full potential of the vehicle it is desirable to consider the use of high energy propellants. This requirement along with the consideration of the use of the vehicle as a laboratory suggests the use of  $\rm H_2/O_2$  for the attitude control system, since the combination provides high energy and is the cleanest burning of all the propellants which may be considered.

The effort reported herein reflects an advanced technology which may be considered for use by such vehicles as the Space Shuttle. The effort utilizes the previously developed catalytic ignition concepts as sponsored by NASA LeRC. The thruster concept was suggested by TRW Systems as a logical concept for such application, particularly to meet long life and high cycle life requirements.

This part of the "Hydrogen-Oxygen Catalytic Ignition and Thruster Investigation" program, NASA LeRC Contract NAS 3-14347, was composed of analytical and experimental evaluations of the duct cooling concept for gaseous  $\rm H_2/O_2$  thrusters operating at high pressure. The high response catalytic igniter activity of the early part of the NAS 3-14347 was extended to this thruster program activity. Whereas the igniter activity began in June 1970, the high pressure thruster activity was added to the program as a limited supplemental effort and commenced in March, 1971. The entire experimental effort terminated in December, 1971.

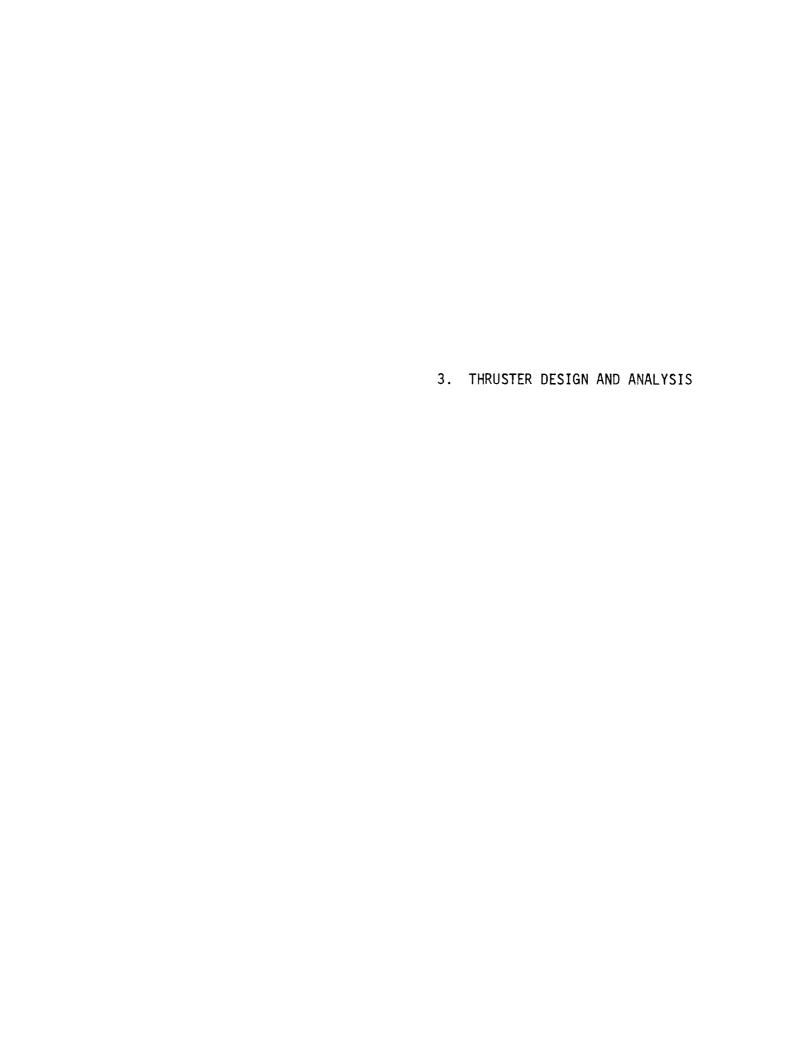
The objectives of the high pressure thruster program effort were as follows:

- Provide a design basis for the use of the duct cooling concept.
- Evaluate the overall thruster performance, operating characteristics and durability of a cooled, near flightweight gaseous  $\rm H_2/O_2$  thruster.
- Determine minimum impulse bit capability of the thruster design.
- Determine the effects of environmental, humid air on the thrust chamber assembly operation.
- Evaluate the potential of the use of the igniter flow only as a minimum impulse bit device.

The following tasks were accomplished to meet the objectives:

- Thruster/Igniter Analysis and Design.
- Thruster Hardware Fabrication (one complete thruster assembly).
- Thruster/System Interaction Analysis.
- Thruster/Igniter Experimental Evaluation.

This report presents results pertaining to the high pressure thruster design. The igniter background is presented in Volume I of this report.



#### 3. THRUSTER DESIGN AND ANALYSIS

The entire emphasis in this part of the contracted effort was on the TRW duct cooled approach. No other designs were considered. The approach is described in detail in the ensuing sections of this report. To orient the reader the overall design concept is presented first.

#### 3.1 DUCT COOLED DESIGN CONCEPT AND FEATURES

There are undoubtedly many approaches which can be taken to the design of gaseous  $H_2/O_2$  thrusters for potential application to the Space Shuttle orbiter vehicle. Some of the more important factors to consider in the design are:

- 1. All propellant lead in plumbing should be integrated into the injector assembly as much as possible. This requirement eliminates external appendages around the thruster assembly which may compound variable installation geometry problems.
- 2. The head end assembly design should accommodate the ability to enter the propellant feed line from: (1) in-line, (2) right angle, and (3) reverse valve installation. The thrust chamber assembly should be unaffected by these installation requirements. This makes possible a common thrust chamber and injector assembly for all installations.
- 3. The thrust chamber cooling concept must not result in specialized nozzle designs for various degrees of scarfing and expansion ratio requirements. This eliminates nozzle regenerative cooling, if a common developed engine is to be adopted to all installations.
- 4. The design goal for thermal and pressure loading margin on the thruster should be an all elastic stress design. If this is achieved high cycle fatigue design approaches can be used.
- 5. The selected materials should be existing, well proven, and readily available materials. This eliminates the need for new materials research to satisfy the design goals.
- 6. The injector design should not couple adversely with the thrust chamber design. If this goal is met the injector can be optimized for performance, and the thrust chamber cooling can be optimally designed without iterative interaction design with the injector.
- 7. For long life the valves should be set-off from the injector by thermal stand-off plumbing and isolated struts. This prevents soak back thermal effects on the valve seats and closures.
- 8. For MIB control the igniter should be designed to provide both ignition functions and small bit propulsion functions. This requires a flow rate adequate for vehicle sensing and an overall

mixture ratio which results in a temperature compatible with the thrust chamber design. It also results in separate valve functions for the igniter which may be met by separate valves or integrated, two step valves.

Figure 1 illustrates the approach taken to partially meet the above goals as well as provide a flexible research tool to meet the objectives of the current contracted effort. Beginning at the head end of the thruster the main valves for the flight engine are in-line poppet valves. (TRW provided Flodyne ball and in-line poppet valves for the testing of the engine.) The main valves are pneumatically actuated by small solenoid pilot valves.

The igniter is fed by separate propellant valves. The GO, enters axially and the  $\mathrm{GH}_2$  enters radially. The igniter operates at a core combustion MR of  $10:1^2$  and an overall MR of 1:1. The igniter is centrally located in the injector. Its combustion chamber is a copper chamber with a reduced throat to provide back pressure in the igniter at startup to enhance ignition.

The injector is a 3-ring raised post triplet. The GH<sub>2</sub> is injected in an axial showerhead manner from the top of the posts. The GH<sub>2</sub> is injected as 2 impinging GO<sub>2</sub> jets onto the center GH<sub>2</sub> jet. The GO<sub>2</sub> orifices are recessed below the surface of the H<sub>2</sub> posts to minimize recirculation effects. The GO<sub>2</sub> manifold feeding is axially through each of the propellant rings. The GH<sub>2</sub> requires a larger manifold, and this requirement is met with the toroidal manifold. The H<sub>2</sub> to O<sub>2</sub> volume ratios are controlled to provide maximum pulse performance. OFHC copper was used for the experimental injector.

Cooling is achieved through the use of a duct in the combustion chamber. This Berylco-10 duct is mechanically anchored on the cold head end only. The remainder of the duct is free to float radially and fore and aft. It is designed to expand to just touch the walls of the nozzle at steady state temperature. The GH2 flows through 90 constant width channels and exits at a contraction ratio of 1.5:1. Heat transfer analyses reveal that the controlling heat transfer resistance is always on the hot gas side. This means that the duct nearly perfectly decouples the cooling from the injector. As a consequence the injector does not have to be tailored to provide a controlled wall temperature environment.

The nozzle is a thin wall configuration. The maximum thickness is at the throat, and a continuous taper from 0.050" (0.127 cm) to 0.020" (.0508 cm) at the exit is used. (An expansion ratio of 40:1 was selected for this program.) The GH<sub>2</sub> film pours along the nozzle wall and provides nearly perfect thermal protection of the nozzle. The peak nozzle temperature is selected near  $1800^{\rm o}{\rm F}$  ( $1000^{\rm o}{\rm K}$ ) and usually occurs at an expansion ratio of  $\sim 8:1$ . From this point on the nozzle temperature actually drops, due to internal radiation. The backwall temperature limits of  $1260^{\rm o}{\rm R}$  ( $700^{\rm o}{\rm K}$ ) are met in two ways. In the combustion chamber section the wall temperature is controlled by the coolant bulk temperature. Beyond the duct exit the surface temperature is controlled by the use of  $\sim 0.125$ " (0.317 cm) of Mini-K insulation type batting.

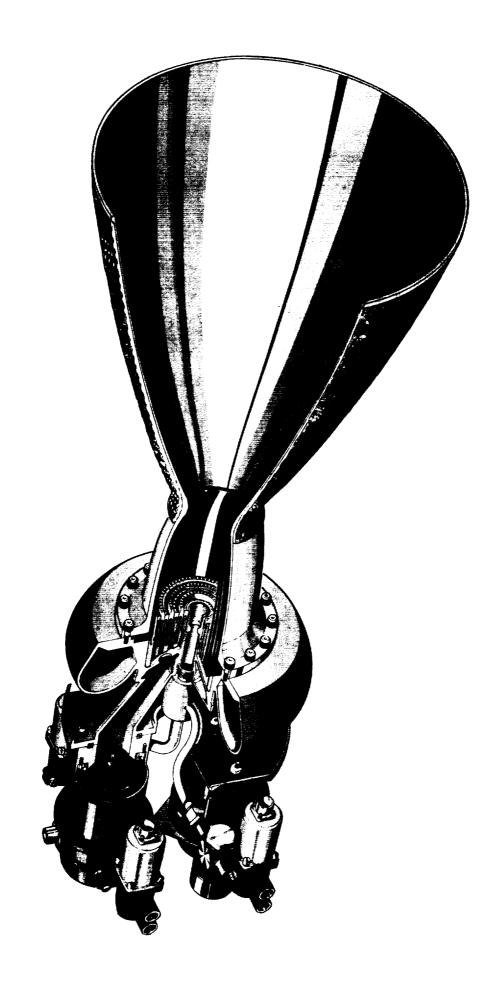


Figure 1. TRW Duct Cooled Engine Concept for  $\mathrm{H}_2\mathrm{O}_2$  Thrusters

For maintainability purposes the thruster is made dissassembleable as shown. The experimental chamber assembly was designed virtually to the above description.

#### 3.2 THRUSTER ANALYSIS AND DESIGN

The analytical design tasks encompassed both high and low pressure thruster designs for  $1500\ lbf$  ( $6672\ N$ ). The nominal design conditions for each case are summarized in Table 1.

Table 1. Thruster Design Conditions

	<u>Case I</u>	Case II
Thrust	1500 lbf (6672N)	1500 lbf (6672N)
Chamber Pressure	15 psia (103 kN/m <sup>2</sup> )	300 psia (2086 kN/m <sup>2</sup> )
Mixture Ratio	2.5	4.0
Nozzle Expansion Ratio	5:1	40:1
Propellant Inlet Temperatures		
0xygen	300°R (167°K)	300°R (167°K)
Hydrogen	200°R (111°K)	200°R (111°K)
Propellant Inlet Pressures		
Oxygen	375 psia (2586 kN/m²)	375 psia (2586 kN/m <sup>2</sup> )
Hydrogen	375 psia (2586 kN/m <sup>2</sup> )	375 psia (2586 kN/m <sup>2</sup> )
C* Efficiency	98%	98%
Specific Impulse	$375 \frac{lbf-sec}{lbm} (3677 \frac{Nsec}{kg})$	435 $\frac{1bf-sec}{1bm}$ (4266 $\frac{Nsec}{kg}$ )
Igniter Propellant Flow Rate	1-5% of the total flow rat	e

Primary emphasis was placed on the high pressure design, although the design approach used is applicable to both the high and low pressure designs.

#### 3.3 THERMAL DESIGN

Detailed thermal analyses were conducted on the design utilizing an approach which had been previously developed by TRW Systems for duct cooled thrusters. The specific thermal design evaluations were as follows:

 Transient and steady state temperature distributions in injector.

- Steady state temperatures in duct and chamber for several duct materials over a range of flow rates.
- For optimum choice of duct material and flow rate the effect of varying duct length was considered.
- Steady state temperatures in duct and chamber flange were determined.
- For optimum choice of duct material and flow rate the transient thermal behavior of the duct was determined.
- Insulation requirements to maintain chamber and nozzle backwall below 800°F (700.056°K) were determined.
- Transient temperature distributions in nozzle were evaluated.

#### 3.3.1 Material Thermal Properties

A summary of the thermosphysical properties of the various candidate materials considered in the thermal analysis are summarized in Tables 2 through 7.

#### 3.4 THERMAL ANALYSIS OF DUCT COOLED CHAMBER AND NOZZLE

#### 3.4.1 <u>Background</u>

The TRW duct/film cooling approach has, experimentally and theoretically, been shown to be an effective method for cooling a wide range of rocket engine designs for  $H_2/F_2$ ,  $N_2O_4/N_2H_4$ ,  $C_3H_8/F_2/O_2$  prior to this program. It essentially consists of fabricating a duct liner which is inserted inside the chamber shell and is cooled by propellant injection into the annular passage at the injector end of the chamber. With suitable duct sizing and flow rates the coolant side conductance will always be much greater than the gas side conductance; consequently the duct is decoupled from the combustion process.

The coolant fluid exits from the duct in the convergent position of the nozzle and will lie as a cool boundary layer close to the wall along the length of the nozzle and will effectively cool the nozzle walls.

Among the advantages of the duct cooling concept are the following:

- Unlike conventional thrusters, duct cooled thrusters are not life limited.
- Fast response and small pressure drops as well as being light and easy to build are immediate advantages of the concept.
- Cooling in throat and nozzle regions is generally more effective than for film cooling alone.
- Structural loads imposed on inner duct wall are small since the pressure differential across wall is small.

THERMOPHYSICAL PROPERTIES OF MATERIALS USED IN THERMAL ANALYSIS

		Pr676 .728 .724 .724 .706 .666 .617 .601
	<sup>p</sup> C <sub>p</sub> (Btu/in <sup>3</sup> - <sup>9</sup> F)	u(Lbm/in-sec) 258x10-6 343x10-6 418x10-6 596x10-6 596x10-6 1086x10-6 1.209x10-6
Table 4 Berylco 10	k(Btu/in-sec- <sup>o</sup> f) 2.48x10 <sup>-3</sup> 2.79x10 <sup>-3</sup> 3.00x10 <sup>-3</sup> 3.10x10 <sup>-3</sup> 3.41x10 <sup>-3</sup> 3.72x10 <sup>-3</sup>	Table 5   Min K Insulation   Rin K Insulation   R
	Temp ( <sup>o</sup> F) -300. 50. 250. 400. 750. 1100.	k = 5.78 PCp = .00 1.512 3.512 3.945 3.763 3.763 3.497 3.481 3.481 3.481
		Temp (°F) -260 -160 -160 -60 500 800 1200 1500
	PC <sub>p</sub> (8tu/in <sup>3</sup> -oF) .0298 .0298 .0305 .0315 .0313	°C <sub>p</sub> (Btu/in <sup>3</sup> -°F) .0241 .0381 .0317 .0321
Table 2 OFHC COPPER	k(Btu/in-sec- <sup>0</sup> F) 5.093c10 <sup>-3</sup> 5.093x10 <sup>-3</sup> 5.002x10 <sup>-3</sup> 4.90x10 <sup>-3</sup> 4.796x10 <sup>-3</sup> 4.731x10 <sup>-3</sup> 4.569x10 <sup>-3</sup>	Table 3 A-286 Steel k(Btu/imsec- <sup>0</sup> F) 1.30x10 <sup>-4</sup> 1.74x10 <sup>-4</sup> 2.18x10 <sup>-4</sup> 2.69x10 <sup>-4</sup> 3.25x10 <sup>-4</sup> 3.76x10 <sup>-4</sup>
	Temp ( <sup>o</sup> F) -300. 50. 300. 600. 900. 1100.	Temp (°F) -300. 70. 440. 800. 1160.

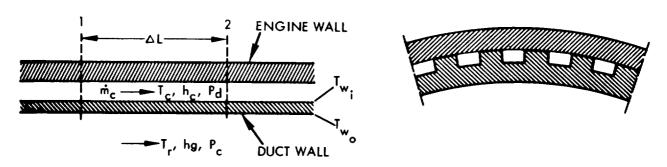
Table 7
Emissivities

inside of nozzle  $\varepsilon=.6$  Outside of insulation  $\varepsilon=.9$ 

#### 3.4.2 Analytical Techniques

#### 3.4.2.1 <u>Duct Cooled Region</u>

For thermal analysis purposes the duct is depicted as below:



<u>Combustion Gas Side Conditions</u>. One dimensional Mach numbers are computed at any axial station on the basis of the following relationship

$$\frac{R_{i}}{R^{*}} = \frac{1}{M_{i}} \left[ \left( \frac{2}{\gamma+1} \right) \left( 1 + \frac{\gamma-1}{2} M_{i}^{2} \right) \frac{\frac{\gamma+1}{2(\gamma-1)}}{2(\gamma-1)} \right]$$
(1)

where suffix i corresponds to some axial station

R = radius

R\* = throat radius

M = Mach number

 $\gamma$  = average gas specific heat ratio

The core mixture ratio (neglecting igniter flow) is given by

$$(0/F)_{C} = \frac{(0/F)_{O}}{1 - \frac{WHD}{WHO}}$$
 (2)

where suffixes c,o refer to core and overall conditions, respectively

(0/F) = mixture ratio

 $\dot{W}_{HD}$  = hydrogen flow in duct

W<sub>HO</sub> = overall hydrogen flow

The core mixture ratio determines the core combustion gas temperature. The combustion temperature is then used to calculate the core adiabatic wall temperature

$$T_{aw} = T_{c} \frac{1 + R^{\left(\frac{\gamma - 1}{2}\right)} M^{2}}{1 + \frac{\gamma - 1}{2} M^{2}}$$
 (3)

where R = recovery factor =  $P_r^{1/3}$ 

P<sub>r</sub> = core gas Prandtl number

The film coefficient for heat transfer between the hot gas and the wall was evaluated using the Bartz simplified technique (Ref. 1) with properties evaluated at the average between the wall gas environment and the wall temperature. The Bartz equation is given by

$$h_g = .026 \quad \left[ \frac{1}{D^{\star \cdot 2}} \quad \left( \frac{\mu^{\cdot 2} c_p}{P_r \cdot 6} \right) \left( \frac{P_c g}{C^{\star}} \right)^{\cdot 8} \right] \quad \left( \frac{A^{\star}}{A} \right)^{\cdot 9} \sigma \quad (4)$$

where  $D^*$  = throat diameter

 $^{\mu,c}_{p}$ , $^{p}_{r}$  = viscosity, specific heat and Prandtl number at stagnation conditions

 $P_c = chamber pressure$ 

C\* = gas characteristic velocity

 $\sigma$  is a correction term which accounts for property variations across the boundary layer and is given by

$$\sigma = \left[ \left( \frac{1}{2} \frac{T_{W}}{T_{C}} - \left( 1 + \frac{Y-1}{2} M^{2} \right) + \frac{1}{2} \right) 0.8 - \frac{\omega}{5} - \left( 1 + \frac{Y-1}{2} M^{2} \right) \frac{\omega}{5} \right]^{-1}$$
 (5)

where  $T_w = wall$  temperature

 $\omega$  = temperature exponent of viscosity, usually given a value of 0.6

Hydrogen Coolant - Side Conditions. The coolant - side convective heat transfer established by McCarthy and Wolf (Reference 2).

$$N_{u_b} = .025 R_{e_b}^{0.8} P_{r_b}^{0.4} (T_w/T_b)^{-.55}$$
 (6)

where  $N_{u_b} = Nusselt number = \frac{h_c k_b}{D_h}$ 

k<sub>b</sub> = bulk thermal conductivity

 $D_h$  = hydraulic diameter in channel

h<sub>c</sub> = coolant side coefficient

 $R_{e_b}$  = bulk Reynolds number

Prb = bulk Prandtl number

 $T_{w}$  = inside wall temperature

 $T_h = coolant bulk temperature$ 

For the type of configurations under investigation, entrace, roughness and curvature effects upon heat transfer coefficients are small enough to be neglected. Any effects would only improve the cooling predictions.

A heat balance is conducted at each lateral station in the duct to evaluate temperature rise of coolant along each station.

$$\dot{q} = \frac{T_{r} - T_{b}}{\frac{1}{h_{g}A_{h}} + \frac{t_{w}}{k_{w}} + \frac{1}{h_{c}A_{c}}}$$
(7)

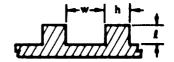
 $A_h$  = heated area per station

A<sub>c</sub> = cooled area per station (including 'fin' effect where applicable)

 $t_w = duct wall thickness$ 

 $k_{W}$  = wall material conductivity

For channels as shown the coolant surface consists of the width 'w' plus some fractions of the channel height 'l' to account for this 'fin effect'. The degree of this fin effect depend upon the duct material and the ratio of l to w. Analysis has shown that it is undesirable to have l/w>l since the extra part of 'l' gives no additional cooling.



Temperature rise of fluid is given by

$$\Delta T = \dot{q}/\dot{m}_c C_{p_b}$$

$$\dot{m}_c = \text{coolant flow rate}$$

$$c_{p_b} = \text{bulk specific heat}$$
(8)

Pressure drops in the duct include both momentum and friction loss and is based upon the fundamental relationship

$$-dP = \frac{\rho V}{g_c} dV + \frac{F_d^{\rho} V^2}{2g_c D_h} dL$$
 (9)

 $\rho$  = density

V = velocity

 $F_d$  = friction factor

From station to station, area changes are small and velocity and density changes are linear then pressure drop per station is approximated by

$$\Delta P = \frac{\overline{\rho} (V_2^2 - V_1^2)}{2g_c} + \frac{F_0 \overline{\rho} \overline{V}^2}{Wg_c D_r} \Delta L$$
where 
$$\overline{\rho} = \frac{\rho_1 + \rho_2}{2}$$

$$\overline{V} = \frac{V_1 + V_2}{2}$$

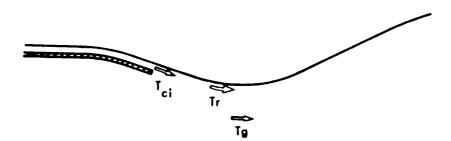
$$\overline{V}^2 = \frac{V_1^2 + V_2^2}{2}$$
(10)

The friction factor is given by the standard relationships for smooth tubes

$$F_D = \frac{64}{R_{ed}}$$
 for  $R_{ed} < 1600$   
 $F_D = .0397$  for  $1600 \le R_{ed} < 400$   
 $F_D = .316$  for  $R_{ed} \le 4000$   $R_{ed} < 100000$ 

# 3.4.2.2 Film Cooled Region

This region extends from the duct exit to the nozzle exit as shown.



In the film cooled region the temperature acting as a driving potential to the wall is dependent upon the temperature at which the coolant exits from the duct, the hot gas temperature, and the degree to which the hot gas mixes with the cool boundary layer. The latter is described by the film coolant effectiveness n.

$$T_{r_f} = T_g - \eta \left( T_g - T_{c_i} \right)$$
 (11)

 $T_{r_f}$  = film recovery temperature

 $T_q$  = gas temperature

T<sub>c</sub> = duct coolant exit temperatures

Knowledge of the film effectiveness, n, as a function of axial distance from the injection point is necessary to establish the local driving temperature at the wall.

Several boundary layer mixing models are available which consider the mixing of the combustion gases with the coolant along the wall. It has been found at TRW that the most effective is the approach of Carlson and Talmor (Reference 3).

Carlson and Talmor assume that the boundary layer growth downstream of the injection is unaffected by the coolant and write the boundary layer flow equation

$$M_{g} = au_{e}\rho_{e} L (\delta - \delta *)$$
 (12)

where

 $M_g$  = hot gas flow rate in boundary layer

a = mixing coefficient

 $u_{\rho} \& \rho_{\rho}$  = velocity and density at edge of boundary layer

 $\delta$  = boundary layer thickness

 $\delta^*$  = momentum thickness

L = axial length

 $\delta$  and  $\delta^*$  are calculated assuming a one-seventh power velocity profile by solving the integral momentum equation for an accelerating, compressible, turbulent boundary layer on an infinite wall. This yields

$$m_{g} = .329 \text{ aL}(R_{eg}, x) \frac{4/5}{(\rho_{e}/\rho_{d})^{1/5}} \left(\frac{\nu_{o}}{\nu_{e}}\right)^{1/5} \left(\frac{T_{e}}{T_{o}}\right)^{8/7} \frac{\nu_{e}}{x^{4/5} \nu_{e}^{108/35}} \left[\int_{0}^{x} \frac{27/7}{T_{e}} \left(\frac{T_{o}}{T_{e}}\right)^{10/7} \frac{\rho_{r} \nu_{r}}{\rho_{o} \nu_{o}} dx\right]^{4/5}$$

 $R_{eq}$ ,x = Reynolds number is core based upon axial distance from coolant outlet

 $\mu, \rho, T$  = viscosity, density, absolute temperature

Suffixes: e = edge of boundary layer

o = stagnation conditions

r = reference temperature

A heat balance relating the boundary layer flow and core gas flow was similar to that adopted by Librizzi and Cresci (Reference 4) and yields

$$m_g = m_c \frac{c_{p_c}}{c_{p_g}} \frac{(T_c - T_{c_i})}{(T_g - T_c)}$$
 (14)

Now

$$\frac{\mathsf{T}_{\mathsf{c}} - \mathsf{T}_{\mathsf{c}}}{\mathsf{T}_{\mathsf{g}} - \mathsf{T}_{\mathsf{c}}} = \frac{\mathsf{1} - \mathsf{n}}{\mathsf{n}} \tag{15}$$

and is therefore the film ineffectiveness to effectiveness ratio denoted by Y.

Combining the boundary layer momentum thickness equation with the heat balance equation, simplifying the integral by taking  $\rho_r$   $U_r \cong \rho_0 U_0$ , and expressing local velocities in terms of local Mach numbers and local density and viscosity in terms of local pressure ratios we arrive at the final form

$$Y = .329aX_1$$
 (16)

where

$$X_{1} = X \left[ \left( \frac{P_{e}}{P_{o}} \right)^{1/4 - \left( \frac{1+\omega}{4} \right) \left( \frac{\gamma-1}{\gamma} \right)} \underbrace{\left[ \frac{1 + \frac{\gamma-1}{2} M^{2}}{x M^{27/7}} \right]}_{x M^{27/7}} \int_{0}^{x} \underbrace{\left[ \left( \frac{1 + \frac{\gamma-1}{2} M^{2}}{2} \right) \right]^{1/2} dx}_{1/2} \right]^{0.8}$$
(17)

where P = pressure

 $\omega$  = temperature exponent of viscosity (nominally 0.6)

M = local Mach number

 $\ensuremath{\mathsf{X}}$  represents the correlation for flat plate incompressible flow with no accelerations

$$\chi = \frac{R_{e_{g_1} x}^{R_{e_{g_1} x}}}{R_{e_{c,s'}}} \left(\frac{\mu_e}{\mu_c}\right) \left(\frac{C_{p_e}}{C_{p_c}}\right)$$
(18)

where  $R_{e_{c,s}}$  = Reynolds number of coolant at channel exit

The assumption of boundary layer growth being unaffected by coolant injection has been examined by Chapman (Reference 5) and found to be valid when coolant flow is less than 50% of the core flow. All cases under examination are well within this criterion and this suggests that the above approach is acceptable.

A pressure gradient correction extending flat plate incompressible correlations to accelerating flow, was also proposed by Stollery and El-Ehmany (Reference 6). The correlation substitutes

$$\left[\frac{1+\frac{\gamma-1}{2}M^2}{M}\right]^4 \int_0^x \left[\frac{M}{1+\frac{\gamma-1}{2}M^2}\right]^4 dx$$
 (19)

for x in the X grouping.

Experimental results obtained at TRW between 1968 and 1971 showed that the experimental results were best described by a combination of these two correlations. This yields as the correlating parameter for compressible accelerating flow.

$$x_1 = x \left\{ \frac{1}{x} \qquad \left[ \frac{1 + \frac{\gamma - 1}{2} M^2}{M} \right] \int_0^x \left[ \frac{M}{1 + \frac{\gamma - 1}{2} M^2} \right] dx \right\}^{8}$$
 (20)

In addition, the mixing coefficient, based upon experiments at TRW was shown to be the form

$$a = A + \frac{B}{P_C} \int_{\substack{duct \\ exit}}^{x} \frac{\partial P}{\partial x} dx$$
 (21)

It is noted here that this simplified form of the boundary mixing coefficient has the effect of density (or pressure level) on it as well as velocity gradient (implicit in the  $\partial P/\partial X$  gradient). This is sufficient to obtain the gross detail of the film mixing coefficients. Furthermore, it takes a most complex phenomena and reduces it to an easily managed engineering solution. In a large number of TRW experiments encompassing film coolants with molecular weight of 2 to 32 it has been found that the subsonic transonic, and supersonic mixing for thrusters at 300 psia (2086 kN/m²) is adequately described for engineering purposes with the emperical constants as

$$A \simeq 0.59$$
,  $B \simeq 0.34$ 

The viscosity of the gas at the edge of the boundary layer is found by the expression given by Bartz (Reference 1)

$$\mu_e = (46.6 \times 10^{-10}) (MW)^{1/2} (T)^{0.6} Lb/In-Sec ( ) (22)$$

MW = molecular weight

Equation (11) now becomes

$$T_{r_f} = T_g - \frac{1}{1 + .329aX_1} (T_g - T_{c_i})$$

This expression is used to evaluate the driving temperature in the film. The heat transfer coefficient between the film and the wall is given by the Bartz relationship.

A heat balance on the wall under steady state conditions yields

$$h_{g_f}(T_{r_f} - T_{w_i}) = \sigma \varepsilon_i F_i(T_{w_i} - T_o^4) + \sigma \varepsilon_o F_o(T_{w_o} - T_o^4)$$
 (23)

subscripts i and o correspond to inside and outside of nozzle wall respectively.

 $T_{t}$  = wall temperature

σ = Stefan/Bottzman constant

 $\varepsilon$  = emissivity

F = shape factor

 $T_{o}$  = ambient environment temperature

### 3.4.3 Additional Duct Design Rationale

From past experience at TRW it has been found that the most desirable and effective design for a coolant duct is one using straight channels with a fairly large number of channels. The latter gives a high degree of cooling and also decouples the duct behavior from any injector flow maldistributions However, it will also raise duct pressure drop and so requires optimization. The choice of number of channels and preliminary dimension estimates are presented below in the design rationale.

### 3.4.3.1 Limiting Parameters

The key design parameters are as follows:

- 1) For any given duct material there is a limiting maximum temperature above which the material strength drops to too small of a value.
- 2) The exit velocity from the duct should equal the full stream velocity. If there is too great a disparity between the two velocities excessive turbulence will be induced in the film which results in increased mixing between the film and the core gas flow and, hence, higher downstream temperatures.

hence 
$$\frac{\dot{W}}{(hw)_{\Theta} \rho w} = V \infty$$
 (25)

where W = coolant flow

 $(hw)_e$  = (height x width) of each channel of duct exit

 $\rho$  = coolant density at exit

n = number of channels

$$(hw)_e = \frac{W}{\rho nVoo}$$

- 3. Since the duct exit pressure must equal the free stream pressure and there is a limit on the maximum duct inlet pressure this will put a  $\Delta P$  limitation on the duct (nominally around 60 psi (413.7 KN/m<sup>2</sup>)).
- 4. The land width between channels and the hot gas wall thickness at the duct exit should be as small as manufacturing tolerances will allow since these 'lands' will cause a 'wake' to be formed in the film. It has been found that with wide exit lands the wake can be sufficient to cause burnout of the thruster walls.

# 3.4.3.2 Design Approach

From limiting design parameters 2) and 4), if it is initially assumed that the exit land width were zero, then

$$W_{e} = \frac{2\pi R_{e}}{n} \tag{26}$$

where

$$R_e$$
 = chamber radius at exit  
 $h_e = \frac{\dot{W}}{\rho V_{00} 2\pi R_e}$  (27)

An estimate is then made of the heat load to the duct wall and this will equal the heat absorbed by the coolant, and the coolant thermal properties at the duct exit are then determined.

Performing a heat balance at the duct exit region

$$h_c^A c (T_{W_i} - T_b) = h_g^A g (T_r - T_{W_0})$$
 (28)

where

h<sub>c</sub> = coolant side coefficient (McCarthy & (Wolf)

 $A_c = cooled area$ 

 $T_{W_i}$  = inside wall temperature

 $T_h = coolant exit temperature$ 

 $h_g = gas side coefficient (Bartz)$ 

 $A_{q}$  = heated area

 $T_n = recovery temperature$ 

 $T_{W_0}$  outside wall temperature

Thus the required value of  $h_cA_c$  to maintain a desired duct wall temperature at the exit is known.

For high conductivity material

$$A_{Ce} = n \left(2h_e + W_e\right)$$
 (Fin effect) (29a)

For low conductivity material

$$A_{C_e} = nW_e$$
 (no fin effect) (29b)

Since  $h_e$  is already previously determined

$$A_{c_e} = \frac{2\pi R_e}{W_e} (2h_e + W_e)$$
 (30a)  
 $A_{c_e} = 2\pi R_e$  (30b)

$$A_{C_{\mathbf{P}}} = 2\pi R_{\mathbf{P}} \tag{30b}$$

Since  $h_c = (h_e, W_e)$  only

$$h_{C} = (n) \text{ only}$$
 (31)

thus 
$$h_c A_c = (n)$$
 only (32)

Therefore the required number of channels to meet the exit heat balance criteria can be determined. From the point of view of manufacturing criteria it is desirable to have the number of channels divisable into 360. Thus the number of channels is rounded off to the nearest number divisible into 360. Another manufacturing criteria is the desirability of maintaining a constant channel width along the length of the duct, ie. The value of We determined will be a constant value along the length of the duct. For high conductivity materials this is a practical criteria since heat striking the 'lands' will be easily conducted into the coolant, even when the lands are fairly wide the conductivity will be sufficient to avoid excessively hot spots. With low conductivity material heat will not be conducted away

so well from the lands and if these are wide enough may get hot enough to burn out and so for low conductivity materials the requirement for constant channel width may not be practical.

Based upon a specified maximum desirable wall temperature the analysis will provide a channel height distribution along the length of the duct to maintain this wall temperature. In general it will not be possible to exactly meet all the geometric, pressure drop and wall temperature requirements and so compromises are made to allow for a practical design. In general, the pressure drop and geometric restrictions are given the first consideration since these are exactly fixed beforehand. There is some latitude in duct temperature and it requires further structural analysis to adequately define the temperature limitations.

## 3.4.4 Verification of the Thermal Model

Prior to the start of this program a series of tests was conducted in the altitude facility at NASA MSFC on a 900 lbf (4003.2N)  $\rm H_2/O_2$  thruster, 40:1 area ratio thruster, operating at 300 psia (2068.5 KN/m²) to verify the above described thermal model. In these tests the percent duct coolant was varied from 20 to 30% of the total  $\rm H_2$  flow. The tests ranged in length from 20 seconds to 120 seconds in duration.

The primary area of thermal interest in these tests was the correlation of behavior in the film cooled region, which was the primary region of thermal uncertainty. Therefore, the region downstream of the duct exit was thermally well instrumented. Since the duct cooled region in the combustion zone was one where standard analytical techniques apply, a much greater level of confidence in the analysis of this region was to be expected. In this thruster the duct extended to an area ratio of 1.5:1 where the coolant hydrogen was injected to become a film.

Typical temperature distribution results are shown in Figures 2 and 3 for high and low mixture ratios. It is seen that the model actually overpredicts the peak temperature and predicts this peak to occur at a location in the nozzle upstream of where it actually occurred.

The "cusp" effect is seen in all the data. The near zero gradient dT/dX in the cusp zones means the real mixing coefficient nearby went to zero in this region. To prove that this in fact is the case the NASA sponsored Mass Addition Boundary Layer (MABL) Program developed at Dynamic Sciences was used to trace the history of the "rigorous" mixing coefficient in the MABL program. The results are shown in Figure 4. This result is in consonance with the TRW results. Since the TRW program provides an excellent rapid means of design solution and so well predicts the nozzle behavior as borne out by the above results it was used exclusively to thermally design the thruster for this program.

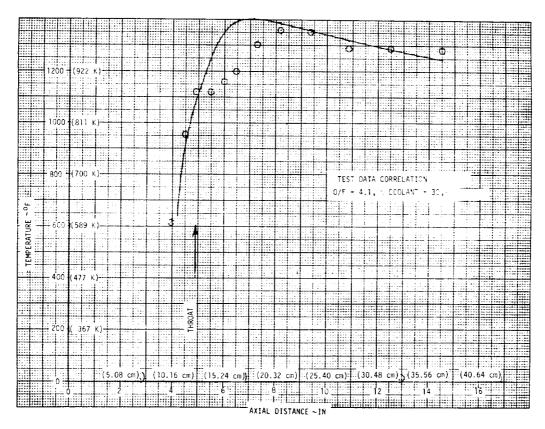


Figure 2. Thermal Model Verification Using MSFC 900 1bf (4003.2N) Results, MR = 4.1:1

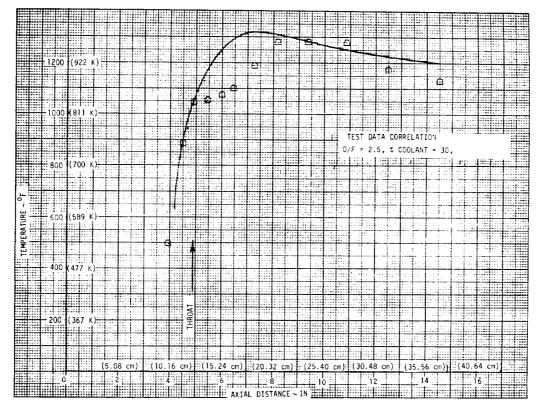


Figure 3. Thermal Model Verification Using MSFC 900 1bf (4003.2N) Results, MR = 2.5:1

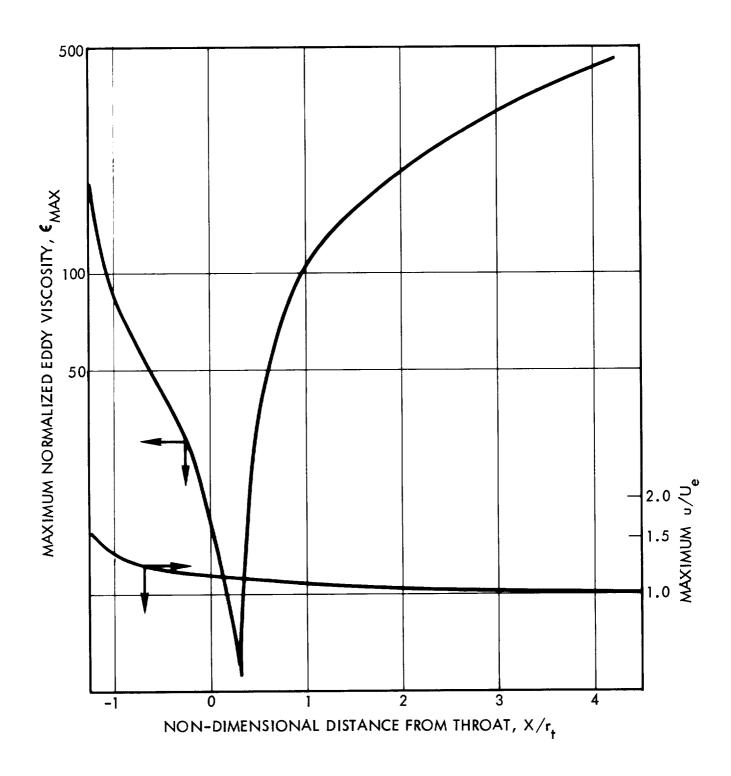


Figure 4. MABL Program Results for 900 lbf (4003.2N) MSFC Data for Mixing Effect and Velocity Ratio

### 3.4.5 Thermal Design Results

The thermal design results for the various duct configurations which could seriously be considered for this program are summarized below. The channel analysis showed the optimum number of channels to be  $\sim$  90. With this number determined the other important parameters were optimized. For the analysis the chamber and nozzle was always kept as A-286 steel.

# 3.4.5.1 Duct Steady State Analytical Results

Only a limited number of analyses were conducted for the ambient temperature condition and the results of this analysis using the duct cooling programs is presented in Figures 5 and 6.

Figure 5 shows the duct design parameters for a duct manufactured from OFHC copper or silver copper, with a limiting  $\Delta P$  of 60 psi. The data show that for reasonable nozzle and duct temperatures the coolant flow should be around 25% of the total hydrogen flow. With a constant channel width of .076" (0.193 cm) the inlet duct height should be .46" (0.117 cm) and the exit duct height .026" (0.066 cm). The resulting temperature distribution in the duct and nozzle are presented in Figure 6.

The results of the analysis for the cold propellant condition are presented in Figure 7 through 13.

Figure 7 shows the typical results of a run conducted using the on line computer program and shows temperature distributions in duct, coolant, film and nozzle wall.

Figures 8 and 9 present the parametric design curves for OFHC, silver/copper, Berylco 10 alloy and A-286 steel. The results for the first three materials are sufficiently close to enable one design chart to be used for all three. In the case of A-286 steel the comparatively low conductivity of this material requires small land widths along the length of the duct and this results in the variable channel width as indicated.

From these design charts it was determined that the optimum coolant flow rate to maintain adequate duct and nozzle temperatures is approximately 20% of the overall hydrogen flow. Figures 10, 11 and 12 present the temperature profiles in the duct and nozzle for OFHC/silver Copper, Berylco 10 and A-286 steel respectively.

The effect of duct length is graphically presented in Figure 13 where the maximum nozzle and duct temperatures and pressure drop are presented for a Berylco 10 duct as a function of duct exit area ratio with a constant coolant flow rate in the duct using duct geometry optimized for a duct exit area ratio of 1.5:1.

Figure 14 summarizes the heat transfer coefficient at the wall along the length of the thruster for a nominal case with a hydrogen inlet temperature of  $200^{\circ}R$  (111 $^{\circ}K$ ), a coolant flow rate of 20% of the total H<sub>2</sub> flow and with the duct taken to an area ratio of 1.5:1.

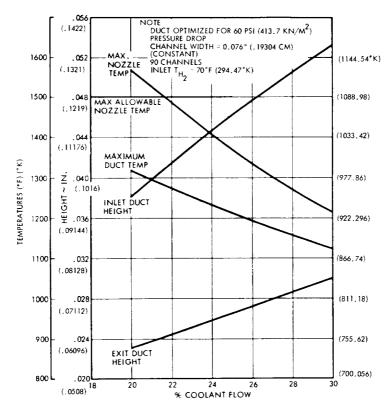
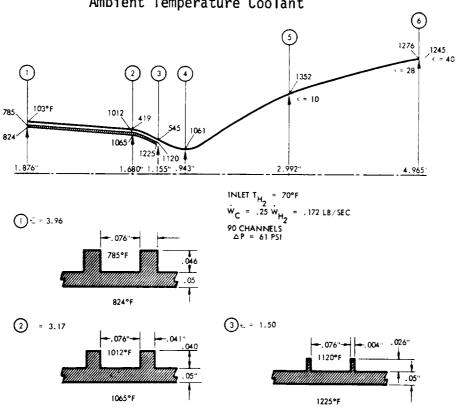


Figure 5. Duct Design Parameters for OFHC or Silver Copper, Ambient Temperature Coolant



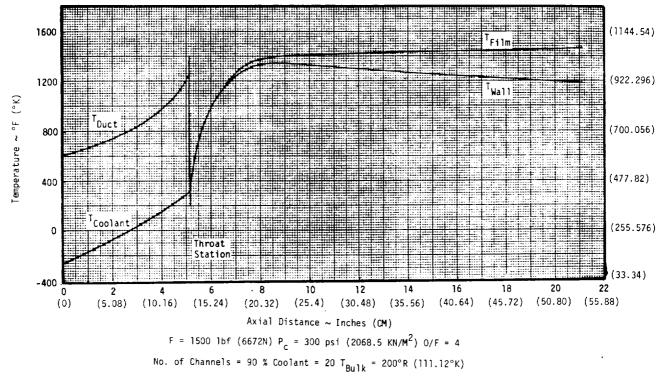


Figure 7. Duct Cooled Thruster Thermal Analysis Design Results With Reduced Temperature Propellants

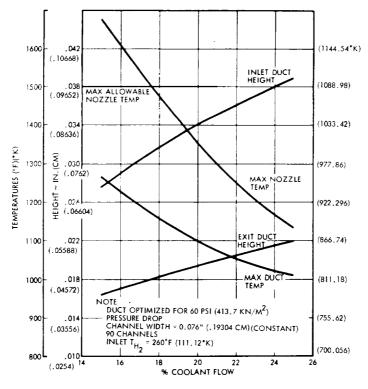


Figure 8. Duct Design Parameters for Copper, Reduced Temperature Coolant

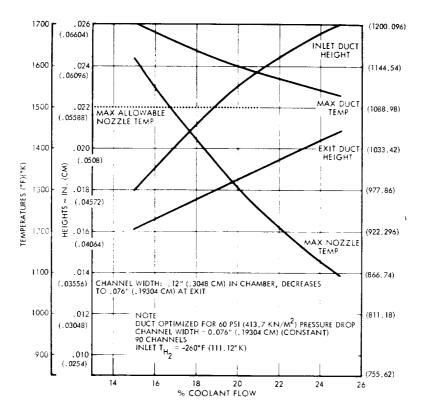


Figure 9. Duct Design Parameters for A-286 Duct

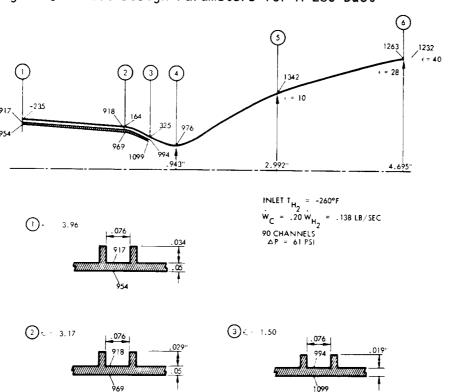


Figure 10. Thruster Thermal Design With OFHC Copper Duct, Reduced Coolant Temperature

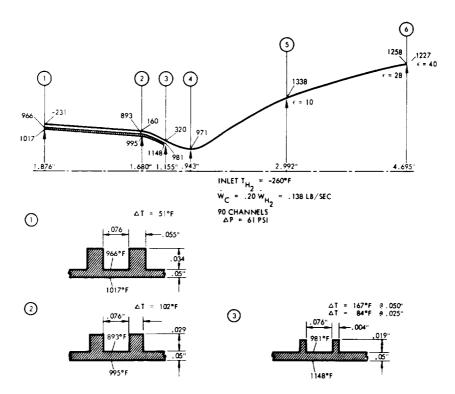


Figure 11. Thruster Thermal Design With Berylco 10 Duct, Reduced Temperature Coolant

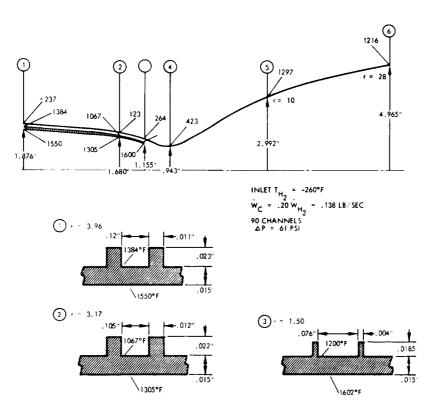


Figure 12. Thruster Thermal Design With A-286 Steel Duct, Reduced Temperatures

29

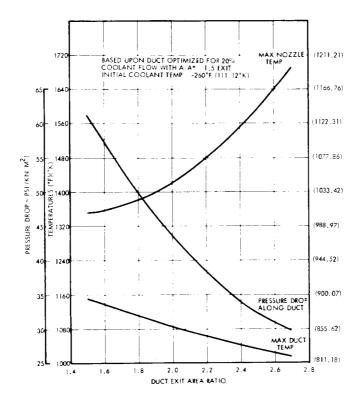


Figure 13. Effect of Changing Exit Area Ratio of Berylco 10 Duct

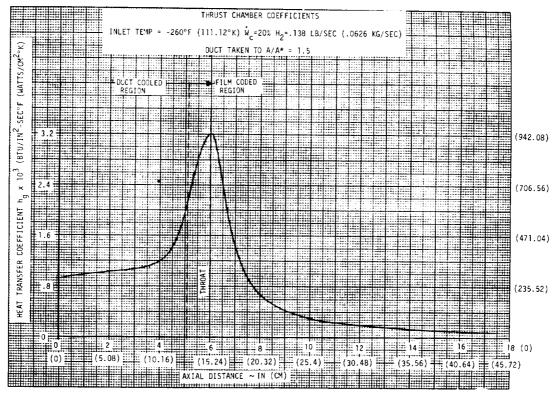


Figure 14. Nominal Gas Side Film Coefficient Distribution Used in the Thermal Analysis

# 3.4.6 Thermal Transient Duct and Nozzle Responses

The duct and chamber were modeled for solution on TAP. The nodal arrangement used in the combustion chamber is shown in Figure 15 with a simplified flange nodal selection. The flange zone was modelled separately as shown in Figure 16. The boundary conditions for the transient runs were derived from the previous data as input to this analysis.

Figure 17 shows the steady state duct results and Figure 18 shows the transient results for the Berylco-10 duct. As is seen the duct will operate at a temperature of approximately 1460 R (811K). It also has a startup thermal time constant of approximately two seconds. This large time constant results in a large margin of safety in the thruster for initial starting condition control. Berylco-10 was selected for the duct material.

The thermal response transients for the nozzle are shown in Figure 19. Here is it seen that the throat has a response time on the order of 6 seconds. While the exit of the nozzle takes nearly 60 seconds to respond. Such response again is indication of the thermal margin in the duct cooled design.

# 3.4.7 Nozzle Insulation Requirements

The nozzle beyond the duct exit must be insulated to meet thruster backwall temperature limitations. For the purpose of sizing an insulation for the thruster the previously derived temperature and heat transfer data were used. Radiation from the backwall was assumed to take place from a surface with emissivity of 0.9 to a sink at  $0^{\circ}R$  ( $0^{\circ}K$ ). The insulation chosen was Min K. Figure 20 presents the steady state backwall temperature at different nozzle stations for a range of insulation thickness. A thickness on the order of 0.125" (0.317 cm) will keep the wall at  $800^{\circ}F$  ( $700^{\circ}K$ ) in steady state.

# 3.4.8 Duct and Flange End Temperature Distributions

Other thermal data of interest to the design are the  $\Delta T$  across the duct and the gradients along the duct and nozzle near the head end. These results are shown in Figure 21 for the various candidate duct assemblies. As is seen relatively small  $\Delta T$  differences occur across the duct itself. The entire head end assembly of the unit can be expected to remain essentially at the H<sub>2</sub> coolant supply temperature.

### 3.4.9 Injector Thermal Modelling

A thermal model of the outer hydrogen and oxygen injector rings was formulated for solution on TRW's Thermal Analyzer Program (TAP). A sketch of this thermal model is presented in Figure 22 indicating the location of the nodes used in the analysis. Each node is assigned a capacitance and is connected to adjacent nodes via resistance paths as indicated in the sketch. Propellant enters the oxygen ring at  $300^{\circ}R$  ( $166.68^{\circ}K$ ) and the hydrogen ring at  $200^{\circ}R$  ( $111.12^{\circ}K$ ).

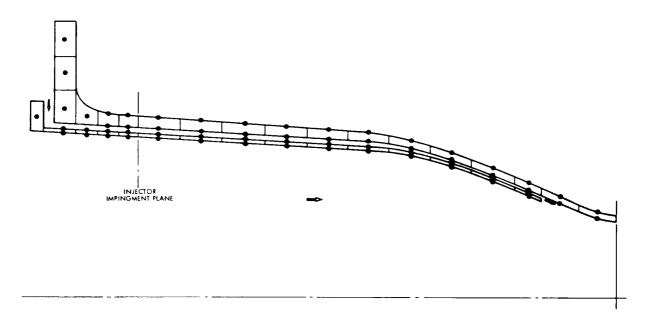


Figure 15. Duct and Chamber Thermal Nodal Model

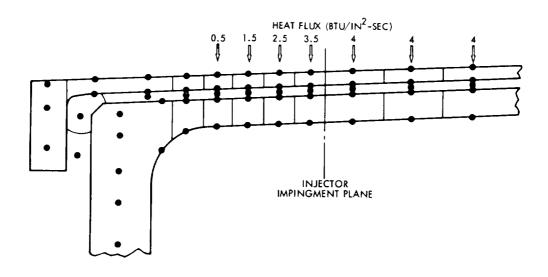


Figure 16. Flange End Nodal Model and Heat Flux Boundary Conditions.

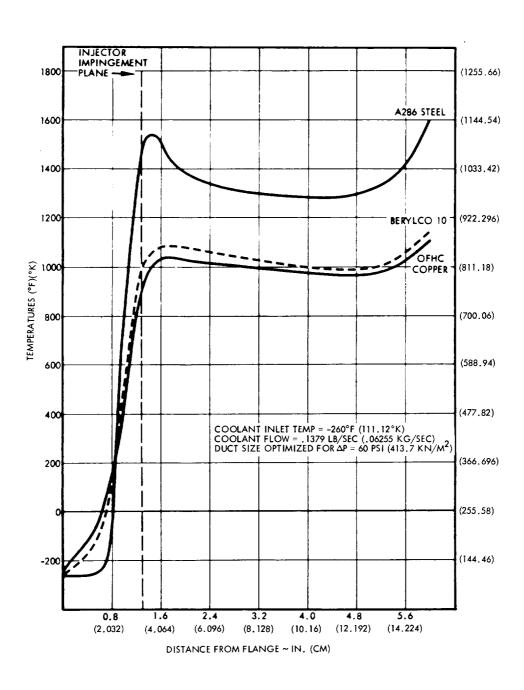


Figure 17. Steady State Duct Temperature Profile

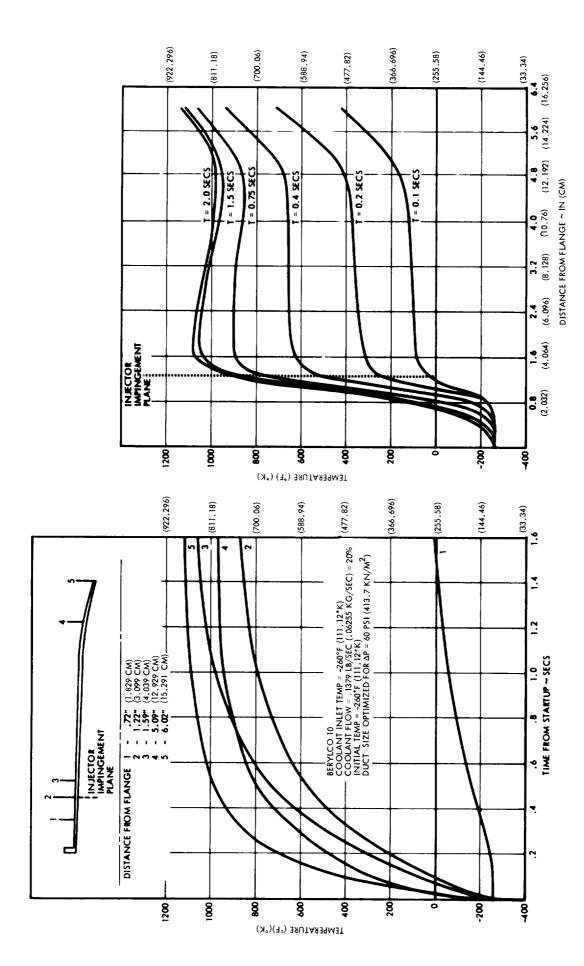


Figure 18. Duct Thermal Transients on Startup

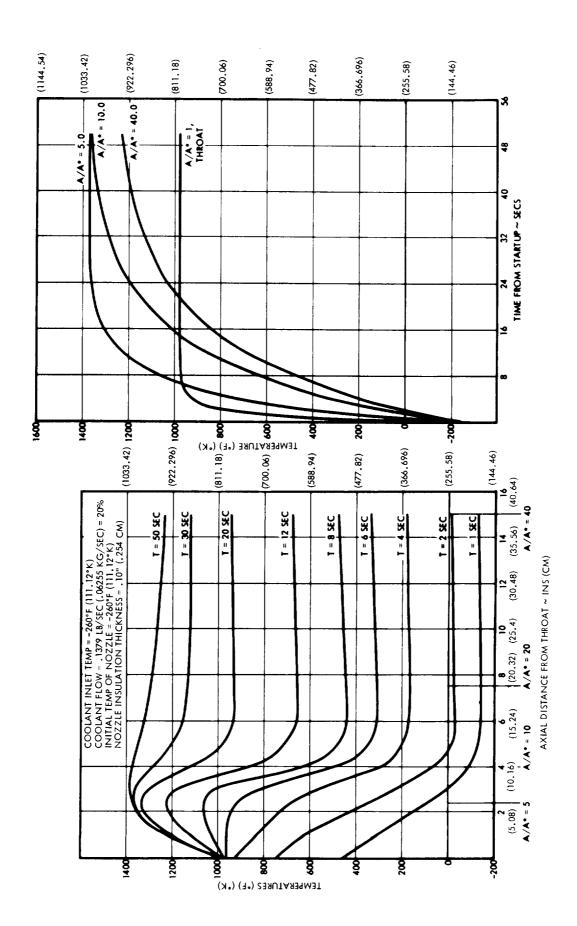


Figure 19. Nozzle Inner Wall Thermal Transients A-286 Steel

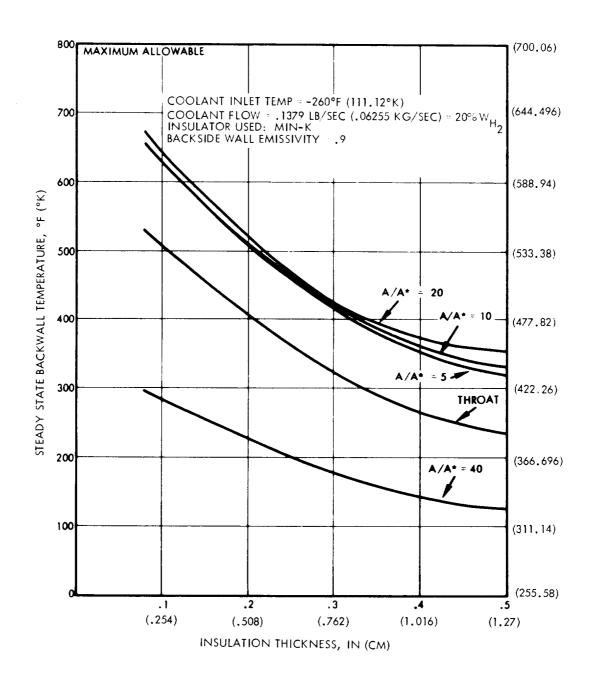
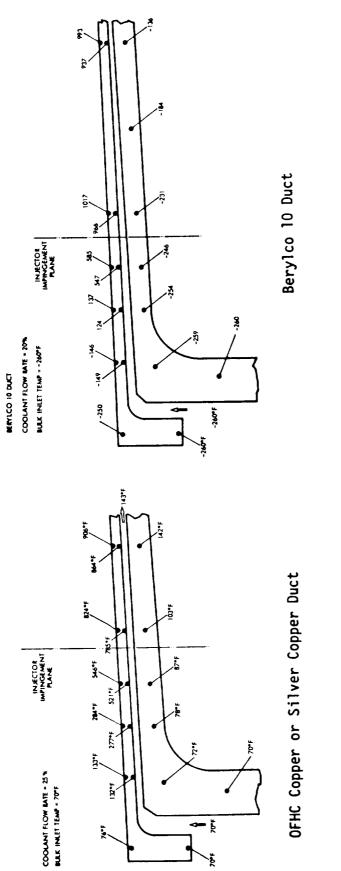


Figure 20. Backwall Insulation Requirements



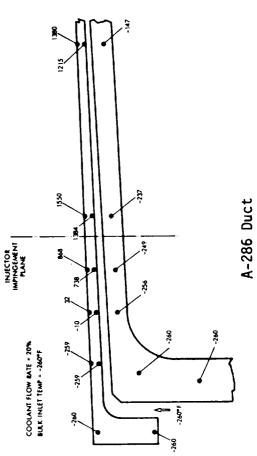


Figure 21. Duct and Chamber Flange End Temperatures

Heat Input to the injector surface could only be approximated as the complex recirculating flow pattern in the immediate vicinity of the injector, does not allow for accurate evaluation of this quantity. However, based upon previous experimental results and from the geometry of the injector the following assumptions were made for the thermal loading on the injector.

Hydrogen ring  $\dot{q}/A = 3$  BTU/in<sup>2</sup>-sec(4.902 MWatt/m<sup>2</sup>) over the projected exposed area Oxygen ring  $\dot{q}/A = 2$  BTU/in<sup>2</sup>-sec(3.268 MWatt/m<sup>2</sup>) over the projected exposed area

The average heat transfer coefficient to the cool gas in the injector orifice is estimated by using the relationship for orifice flow with a non-fully developed boundary layer. (Reference 7)

$$\overline{N}_{u_1} = .116 \left[ R_{ed}^{2/3} - 123 \right] P_r^{1/3} \left[ 1 + (d/x)^{2/3} \right] \left[ \frac{\mu_B}{\mu_W} \right]^{.14} -$$
(33)

where

 $\overline{N}_{u_1}$  = average Nusselt Number in orifice

R<sub>e</sub> = Reynolds number in orifice

P<sub>r</sub> = Prandtl number

d = diameter of orifice

x = length of orifice

 $\mu_{R}$  = viscosity of propellant at bulk temperatures

 $\mu_W$  = viscosity of propellant at wall temperature

In the annular manifold the coolant side coefficient is given by the relation for flow in an annulus

$$h_2 = \frac{.021 \left(1 + 2.3 \, d_{h/L}\right) \, c_p \, \rho \, V}{R_e^{.2} \, P_\mu^{.2/3}}$$
(34)

d<sub>h</sub> = hydraulic diameter of annulus

1 = length of annulus

 $C_n$  = bulk specific heat

ρ = bulk density

v = fluid velocity

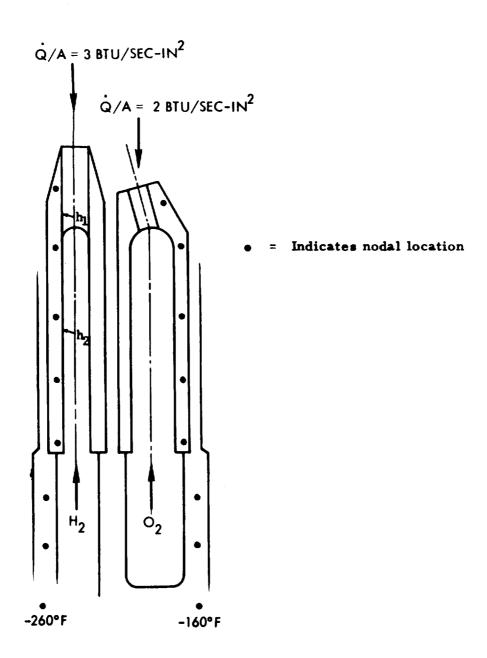


Figure 22. Injector Thermal Model

the results of the analysis are shown in Figure 23 for the 2 rings. The transient response results show the injector to have a response time of  $\sim$  9 seconds.

# 3.4.10 Stress and Life Analysis

Both the thrust chamber duct and the nozzle lend themselves quite nicely to rigorous analytical stress analysis. Thin shell theory is readily applicable and has been used exclusively in the design efforts. Final analyses were conducted using the Shell Computer Program developed in NASA Contract NAS 9-4552.

The Rhome and Hass program was used for the injector. The rudiments of the design approach are summarized here. For the purposes of the stress analysis the following was assumed.

- PRELIMINARY CHAMBER GEOMETRY SELECTION
  - . L\* = 18 inches (45.72 cm)
  - . Contraction Ratio = 4.0 (with tapered inner wall contour)
  - . Nozzle: 80% Bell
  - Duct Exit Plane Location:  $\epsilon_c = 1.5$  to 2.5
- CHAMBER WALL
  - Thin-walled Outer Shell/Nozzle: A286
  - Duct Liner: OFHC Copper/Copper Alloy
- DUCT DESIGN APPROACH
  - . Straight Channels
  - . Internally Fed from Injector  $H_2$  Manifold

Of primary interest here is the expected fatigue life. The basic starting point in the analysis is the Manson universal slope equation to find the total strain range.

$$\Delta \epsilon_{t} = \frac{3.5 \sigma_{u}}{E} N_{f}^{-0.12} + D^{0.6} N_{f}^{-0.6}$$
 (35)

where

 $\Delta \epsilon_{+}$  = total strain range

 $\sigma_{\mu}$  = ultimate tensile strength

E = modulus of elasticity

 $D = \ln \left( \frac{100}{100-RA} \right)$ 

RA = reduction in area

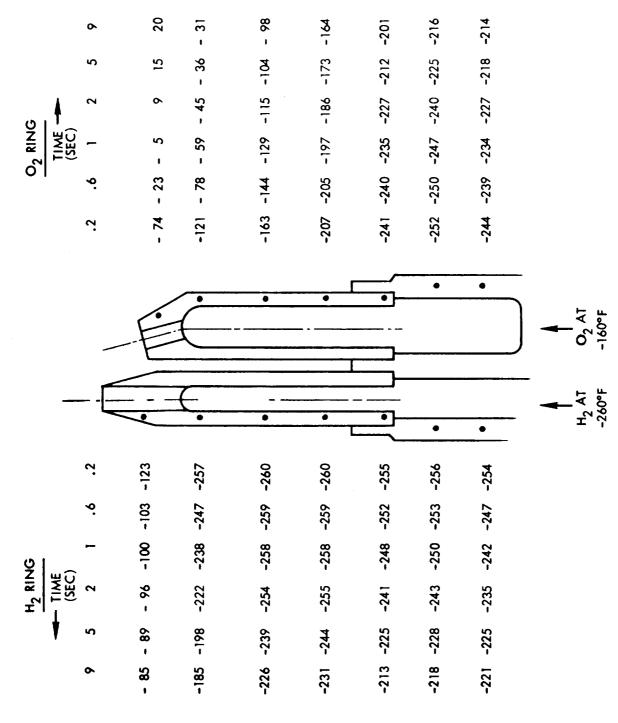


Figure 23. Injector Transient Temperature Profiles

The stress analysis considered the effects of:

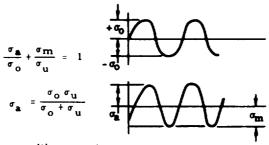
- Pressure Loading
- Thermal Stress
- Combined Fatigue and creep damage
- Dynamic Stress

As indicated above the duct concept lends itself to text book formulation. Prior to using the thin shell program the duct and nozzle were analytically described as illustrated in Figure 24 for the axial and meridional stresses introduced into a cylindrical element by the loads on an element. From this analysis the basic understanding of the thruster was derived and the geometry was programmed into the Shell Computer Program, as schematically illustrated in Figure 25.

At elevated temperatures where intercrystalline cracking may occur due to the effects of creep, the above rule was modified by the use of the "10% Rule." In the use of this rule the fatigue life is taken as 10% of that predicted by the universal slope method. Therefore, for 106 cycle life, the designer uses  $N_{\rm f}=10^7$  in the universal slope equation and the permissible total strain range is determined. The calculation fatigue stress range with mean stress  $\sigma_{\rm m}=0$  is found from

$$\sigma_0' = \pm \frac{1}{2} \Delta \varepsilon_t \cdot E \tag{36}$$

From this result and available test data for the candidate materials. The fatigue stress range  $\sigma$  was obtained by proportioning the above theoretical results. Modified Goodman diagrams were used to calculate the fatigue stress  $\sigma_a$  as follows:



- $\sigma_{o}$  = fatigue stress range with mean stress  $\sigma_{m}$  = 0
- $\sigma_{a}$  = fatigue stress range with mean stress  $\sigma_{m}$  =  $\sigma_{a}$
- σ = material ultimate strength

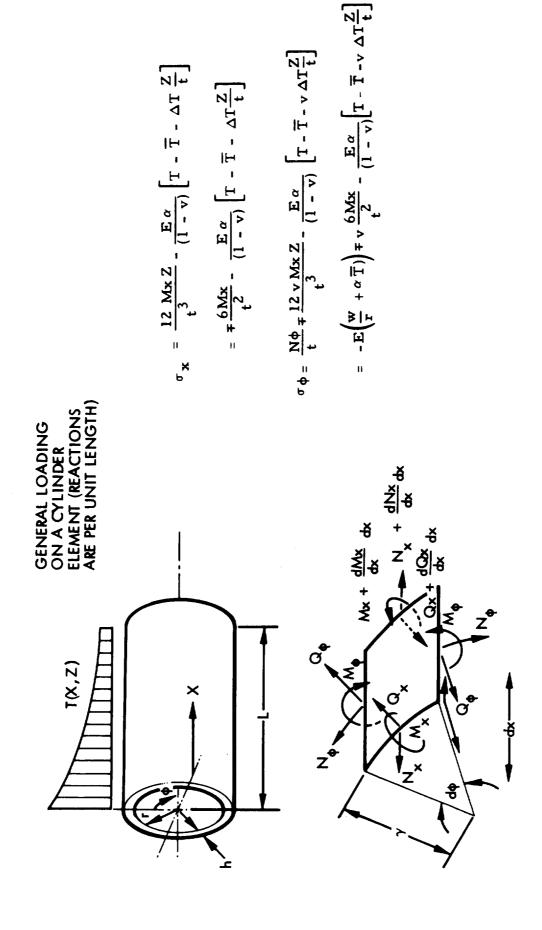
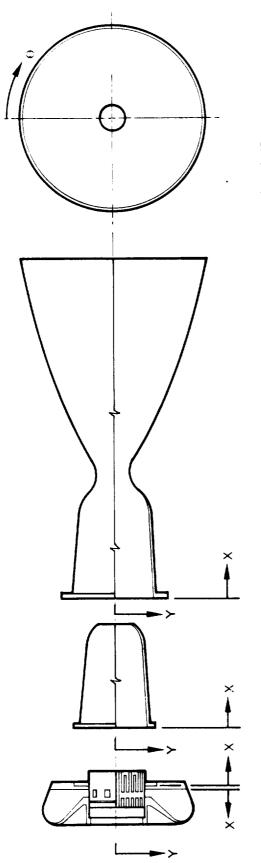


Figure 24. Thermal Stress Analysis Model for Duct Cooled Thruster



STRESS-DEFORMATION ANALYSIS OF SHELL OF REVOLUTION SUBJECT TO ARBITRARY LOADS AND TEMPERATURE DISTRIBUTIONS =

$$P = f(X, Y, \Theta)$$

$$T = f(X, Y, \Theta)$$

# PROGRAM BASIS

- LINEAR ELASTIC THIN SHELL THEORY
- FOURIER SERIES EXPANSION TECHNIQUE APPLIED
- PROGRAM WRITTEN IN FORTRAN IV
- SOLUTIONS YIELD SEPARATELY BENDING MOMENTS AND MEMBRANE FORCES SO THAT SECONDARY AND PRIMARY STRESSES COULD BE EVALUATED.

Figure 25. Shell Computer Program NASA Contract NA-S9-4552

The total fatigue stress range is given by  $2\sigma_a$ .

To include the combined effects of fatigue and creep the three approaches illustrated in Figure 26 were considered. The third method was selected, since it is felt to be most conservative and makes direct use of experimental data illustrated in Figure 26.

To facilitate the use of this analytical approach fatigue-temperature diagrams for the candidate materials are prepared from which one can directly determine the design stress allowables when the temperature distributions are known. Typical results are given in Figures 27 through 30.

The computer results for the A-286 nozzle are given in Figure 31 for the  $10^6$  cycle use without creep, 10 hrs and 50 hrs creep. The immediate observation is that the thin wall nozzle has a very large fatigue-creep margin, and no concern should be expressed over its life capabilities.

The initial results for the duct are shown in Figure 32. Here it is seen that both the analytical model and the computer model predicted an excessive stress at the beginning of the duct. To solve this potential problem a small amount of  $GH_2$  film cooling (5%) was added to the hot gas side. This removed the duct temperature discontinuity effect, and Figure 33 shows the resultant stresses, indicating that the  $10^6$  fatigue cycle limit can now be met.

There also exists the possibility of destructive startup transient effects in the nozzle wall at the throat. Because of the thin wall and reduced driving temperature with the duct coolant, this problem is reduced to negligible proportions with the A-286. The results are shown in Figure 34.

The more complex triplet injector geometry was analyzed strictly by computer program. The results are given in Figures 35 and 36 for both ambient and reduced temperature propellants.

A thruster dynamic stress analysis was also conducted for the model shown in Figure 37. The analysis resulted in the conclusion that there was no problem here.

The low pressure thruster was analyzed in exactly the same manner as above. The entire results for the low pressure thruster are summarized in Figure 38.

The stress analysis conclusions were as follows:

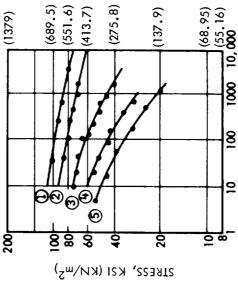
- Fundamental theories of beams on elastic foundation, theory of elasticity, theory of plates and shells (Hetenyl and Timoshenko) are directly applicable.
- All components have margin of safety based on 10<sup>6</sup> cycle fatigue and 50 hours creep damage.

• 
$$(2\sigma_a)^i = (2\sigma_a) \frac{\sigma_a \text{ (creep rupture)}}{\sigma_u}$$

= ultimate strength of material at temperature

 $(2\sigma_a)$  = total fatigue stress range

For combined fatigue and creep damage



= effective fraction of each cycle for which material is subjected to maximum stress (use value of 0.3)

N<sub>f</sub> = fatigue cyclic life

يد

 $1 + \frac{k}{AF} (N_f) \frac{m + 0.12}{m}$ 

Z

This equation applies if  $N_f' < 10\% N_f$  and if  $N_f' < 10^5$  cycles.

m = slope of creep rupture line

 $\phi_{\mathrm{f}}$  = damage fraction caused by fatigue = damage fraction caused by creep

**•**∪

φ<sub>f</sub> + φ<sub>c</sub> ≤ 1 (Rocketdyne)

= time to intercept of creep rupture curve

∢

= frequency of application, cpm

Ŀ

2

8 CREEP RUPTURE A-286 TIME - HR 8

1000°F (811.18°K) (866.74°K) 1100°F (922.296°K) 1200°F (977.86°K) 1300°F 1350°F (1005.64°K)

00000

Combined Fatigue and Creep Analysis Approaches Figure 26.

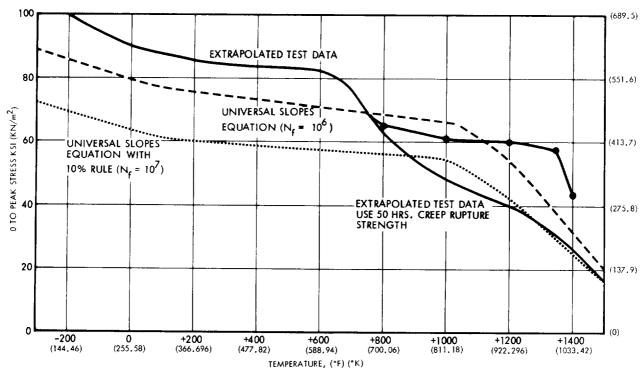


Figure 27. Unnotched Allowable Maximum Fatigue Stress for A-286 (Aged) at 106 Cycles

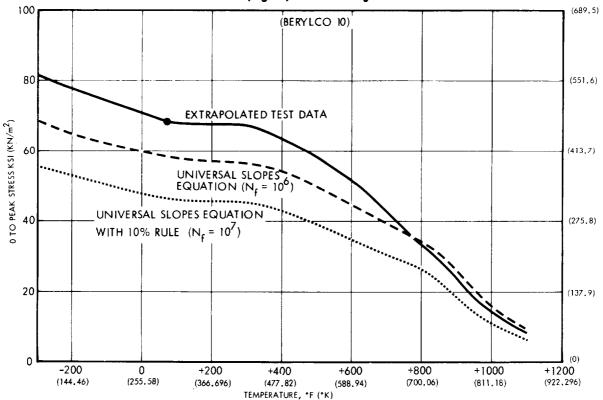


Figure 28. Unnotched Allowable Maximum Fatigue Stress for Be Cu Alloy No. 175 (Berylco 10) at 106 Cycles

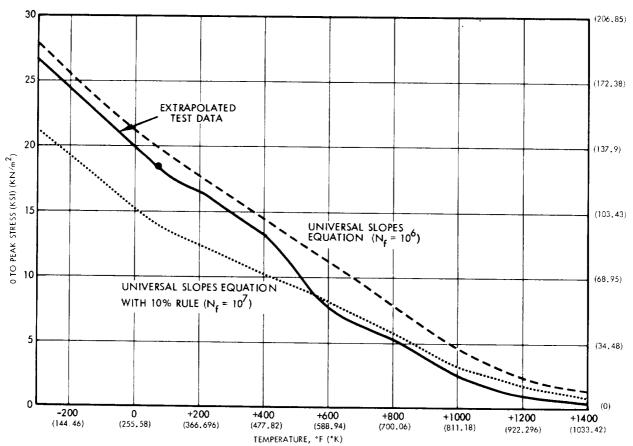


Figure 29. Unnotched Allowable Maximum Fatigue Stress for OFHC Copper (Annealed) at 10<sup>6</sup> Cycles

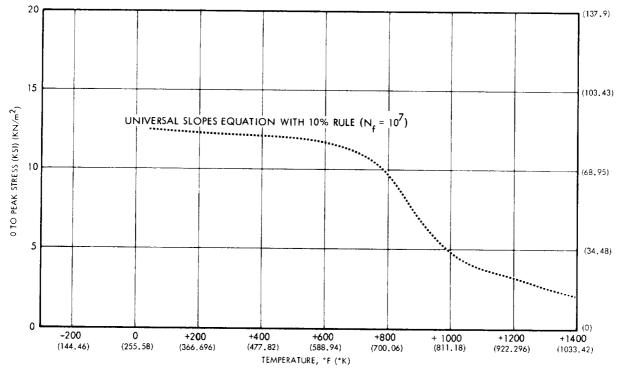


Figure 30. Unnotched Allowable Maximum Fatigue Stress for Narloy Casting Alloy at 10<sup>6</sup> Cycles

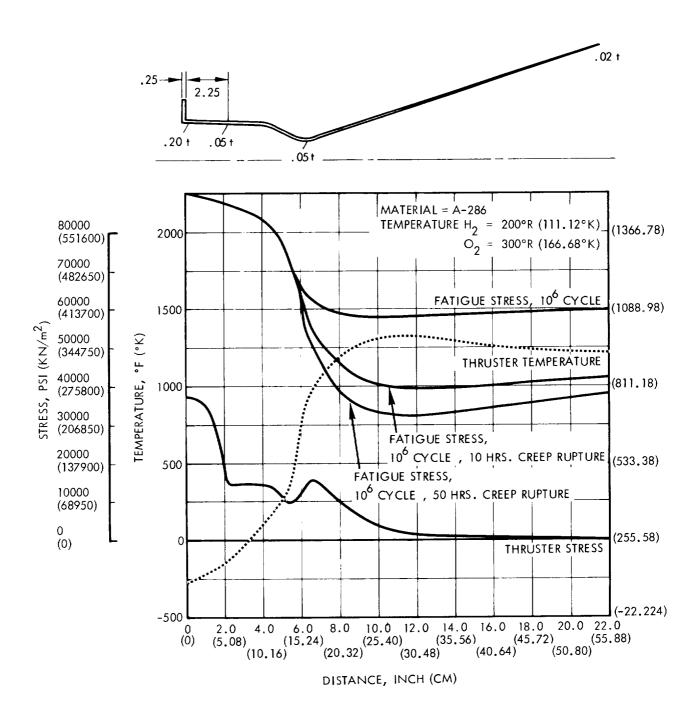
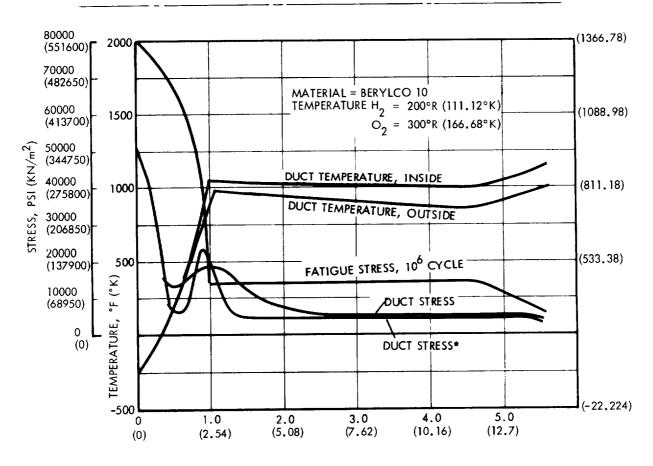


Figure 31. Thrust Chamber Stress





DISTANCE, INCH (CM)

\*FROM COMPUTER PROGRAM

Figure 32. Duct Stress

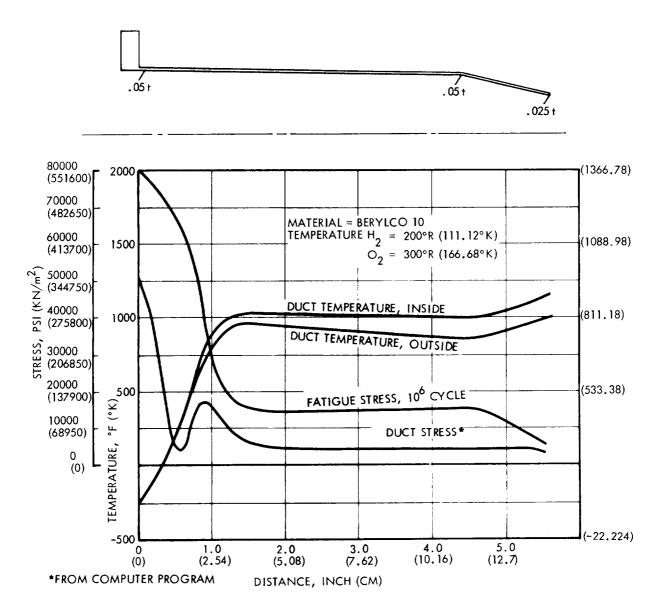
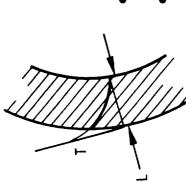


Figure 33. Duct Stress - Film Cooling Added (5%)

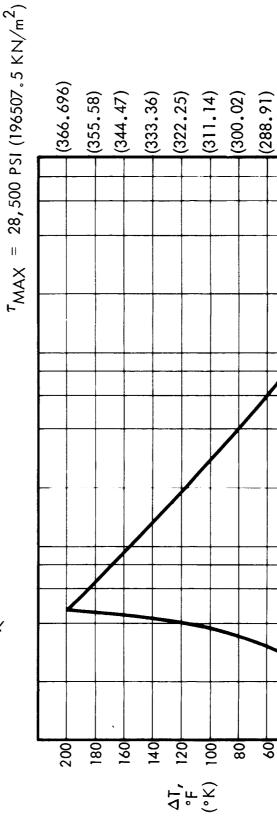
IN DUCT COOLED ENGINE WORST START UP ZONE IMMEDIATELY DOWNSTREAM OF THROAT



$$\frac{T_o}{T_a d} = \frac{\frac{1}{2} \frac{h}{k} \sqrt{6 \alpha_T t}}{1 + \frac{1}{2} \frac{h}{k} \sqrt{6 \alpha_T t}}$$

• MAXIMUM  $\Delta I A I t = L^2 / 6 \alpha_I$ 

 $\tau = \pm \frac{E\alpha_2 \ \Delta T}{2 \ (1 - \Delta)}$ 



TIME FROM ENGINE STARTUP, SECONDS

Start Up Transient Stresses

Figure 34.

(277.80)

(266.69)

10.0

.000

0.100

40

52

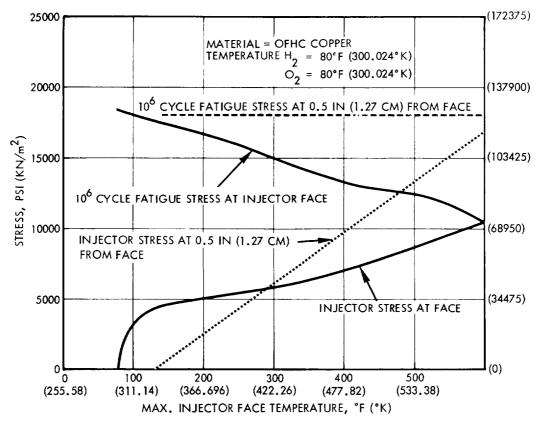


Figure 35. Maximum Injector Stresses With Ambient Propellants

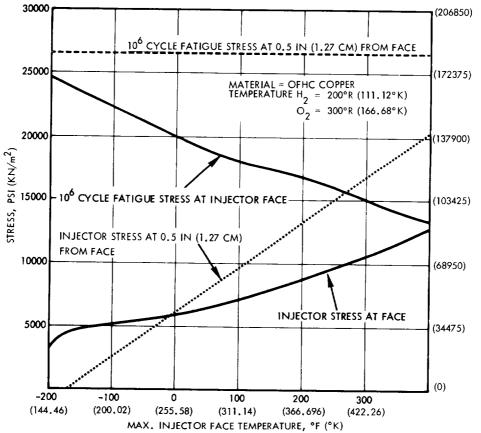


Figure 36. Maximum Injector Stresses With Reduced Temperature Propellants

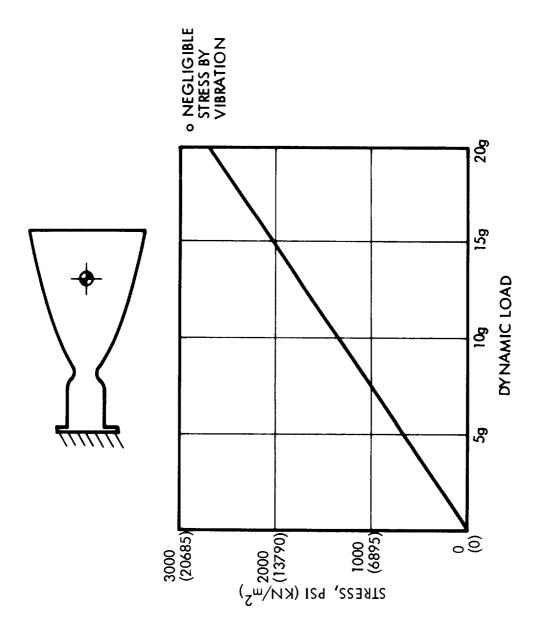


Figure 37. Dynamic Stress Magnitude at Nozzle Throat

Summary of Thermal Stress Analysis for 15 psia (103, 43  $\rm KN/m^2)$  Ducted Thrust Chamber Design

J.

Location:	Chamber Duct	Nozzle
Material:	A-286	A-286
Thickness (in.) (cm)	0.05 (.127)	0.05*
Allowable Stress		
Ultimate (psi) (KN/m <sup>2</sup> )	138,000 (951510)	64,000 (441280)
Yield (psi) (KN/ $\mathrm{m}^2$ )	93,000 (641235)	62,000 (427490)
10 <sup>6</sup> Cycle Fatigue (psi) (KN/m <sup>2</sup> )	42,000 (289590)	21,000 (144795)
Temperature $(^{O}F)$ $(^{O}K)$	800 (700.06)	1,400 (1033.42)
Maximum Calculated Stress (psi) $\left(\mathrm{KN/m}^2 ight)$	20,900 (144105.5)	(0) **0
Margin of Safety	1.01	High

\*0.05 inch (.127 cm) at throat tapering to 0.02 (.0508 cm) at exit.

STRESS, PPSI (KXN/m²).

STRESS, PPSI (KXN/m²).

STRESS, PPSI (KXN/m²).

STRESS, PPSI (KXN/m²).

STRESS, PPSI (KNN/m²).

STRESS, PPSI (KNN/m²).

STRESS, PPSI (KNN/m²).

STRESS, PPSI (MN/m²).

STRESS, PPSI (M

Duct-Cooled Thruster Thermal/ Structural Material Data

Figure 38. Thruster Thermal Stress Analysis Summary

 Large margin of safety for re-entry induced stress at maximum circumferential gradient of 2000°F (1366.78°K) to 1200°F (922.296°K) and maximum axial gradient of 2000°F (1366.78°K) to 1200°F (922.296°K) over initial 3 inches (7.62 cm).

#### 3.4.11 Performance Prediction Approach

The theoretical behavior of  $\rm H_2/O_2$  propellants is current propulsion state-of-the-art. Consequently, this type of data is not presented here. The injector design approach from a performance point of view is of interest, and the engineering method used for gaseous propellant injectors at TRW Systems is summarized below.

Starting from the premise that combustion efficiency is governed by turbulent mixing of the two propellants after some initial mixing, an analytical expression is formulated for the combustion efficiencies as a function of the hydrogen jet radius and the distance traveled by the propellants. The axial mixing is governed by Equation (33) below (Reference (7):

$$Y_0 = 1 - e^{-\left[\frac{1}{k \times \ell_e^{1/2} - 0.70}\right]}$$
 (33)

where  $Y_0$  = centerline concentration

This expression covers the case of a gas jet in a continuum of another gas specie and relates the axial concentration (or mixing) variation. It is postulated that the combustion efficiency for this case would be related to  $1-Y_0$  or:

$$\eta_{c}^{\star} = e^{-\left[\frac{1}{kx\rho_{e}^{1/2}} - 0.70\right]}$$
(34)

where: k = constant (determined experimentally)

 $\overline{x} = x/r_j$ 

 $\Sigma$  = axial distance

r<sub>j</sub> = jet radius

 $\rho_e = \rho_e/\rho_j$ 

 $\rho_e$  = density of continuum gas

 $\rho_i$  = density of jet gas

In order to more closely simulate an actual rocket engine injector, Equation (35) is modified to account forinteraction between adjacent jets caused by small inter-element spacing.

$$\eta_{c^*} = e^{-\left[\frac{c}{k\bar{x}\bar{\rho}_e^{1/2}} - 0.70\right]}$$
(35)

where: c = interaction parameter

This expression has been used to correlate experimental data from various sources for different injector geometries as shown in Figure 39. The constants k and c are found to have limits within which Equation (35) can be adequately used to describe the axial combustion efficiency variation. As seen the constant k varies between 0.03 and 0.10 and is related to the degree of initial mixing. Thus, for a showerhead injectors with very little initial mixing k is 0.03 with resulting greater lengths required for high performance, whereas a triplet injector has a great deal of initial mixing with corresponding shorter lengths required for good combustion efficiency. The interaction parameter, c, is a measure of the interaction between adjacent jets which can result in enhanced mixing. This constant is thus a function of the pattern fineness or inter-element spacing.

\_To determine the effects of other engine parameters, the expression for  $\bar{x}$  can be generated as follows:

$$\overline{x} = \frac{x}{r_j} \tag{36}$$

where 
$$x = \frac{L^*}{\varepsilon_c} - \frac{1}{3_{\varepsilon_c}} \left( \frac{F}{\pi C_F P_c} \right)^{1/2} \cot \beta \left( \varepsilon_c^{1.5} - 1 \right)$$
 (37)

L\* = characteristic chamber length

 $\varepsilon_c$  = chamber contraction ratio

F = thrust

 $C_{r}$  = thrust coefficient

P<sub>c</sub> = chamber pressure

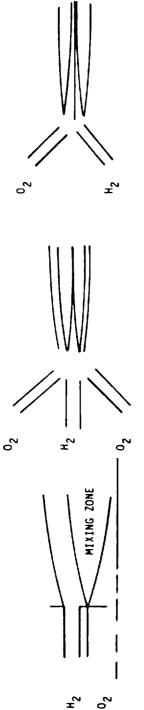
 $\rho$  = chamber contraction angle

and

$$r_{j} = \left[\frac{A_{j}}{n_{j}\pi}\right]^{1/2} \tag{38}$$

 $A_{j}$  = total flow area of jet

n; = number of jet orifices



NOTE: H2 VELOCITY ALWAYS -5-8 02 VELOCITY ASSUMING AXISYMMETRIC SYMMETRY, CENTER LINE CONCENTRATIONS FOLLOW FOR:

$$Y_0 = 1 - Exp \left[ \frac{K_0}{K_1 \times R_0} \right]$$

COMBUSTION PERFORMANCE POSTULATED TO BE:

$$\frac{x}{\rho_{c}} = \frac{x}{r} / \frac{r}{J}$$

$$\frac{x}{\rho_{c}} = \frac{z}{\rho_{c} / \rho_{c} J}$$

$$\frac{x}{\rho_{c}} = \frac{1}{\rho_{c} / \rho_{c} J}$$

$$\frac{y}{\chi} = \frac{1}{\rho_{c} / \rho_{c} J}$$

$$\frac{y}{\chi} = c \frac{L}{\epsilon_{c}} (\frac{p}{\rho_{c}})^{1/2} - \frac{1}{\rho_{c}} (\frac{\epsilon_{c}}{\rho_{c}})^{1/2}$$

$$\frac{y}{\chi} = c \frac{L}{\epsilon_{c}} (\frac{p}{\rho_{c}})^{1/2} - \frac{1}{\rho_{c}} (\frac{\epsilon_{c}}{\rho_{c}})^{1/2}$$

EXAMINED DATA FOR TRIPLET, DOUBLET, SHOWERHEAD, COAXIAL INJECTORS. RANGE OF PARAMETERS: 
$$0.5 < K_0 < 3.0\\ 0.03 < K_1 < 0.1\\ 0.03 < K_1^2 < 0.1$$
 IMPINGING JETS 
$$K_3 = 0.70$$

Figure 39. Generalized Mixing Model

Equation (37) can be written as follows for  $\beta$  = 40<sup>0</sup> and C<sub>F</sub> = 1.70 since these parameters will not vary too much for different cases.

$$x = \frac{L^*}{\mathcal{E}_c} - .13 \left(\frac{F}{P_c}\right)^{1/2} \left[\frac{\mathcal{E}_c^{1.5} - 1}{\mathcal{E}_c}\right]$$
 (39)

Equation (38) can be transformed by using the expression for compressible gas flow and by taking the case of: 0/F = 4.0 with  $M_{inj} = .6$ ,  $C_D = .75$ ,  $T_o = 530^{\circ}F$  (550.04°K),  $\gamma = 1.41$ , and M = 2.016.

This results in the following:

$$r_{j} = .18 \left(\frac{F}{n_{j}P_{c}}\right)^{1/2} \tag{40}$$

Combining Equations (39) and (40) results in the following expression for the nondimensional axial coordinate:

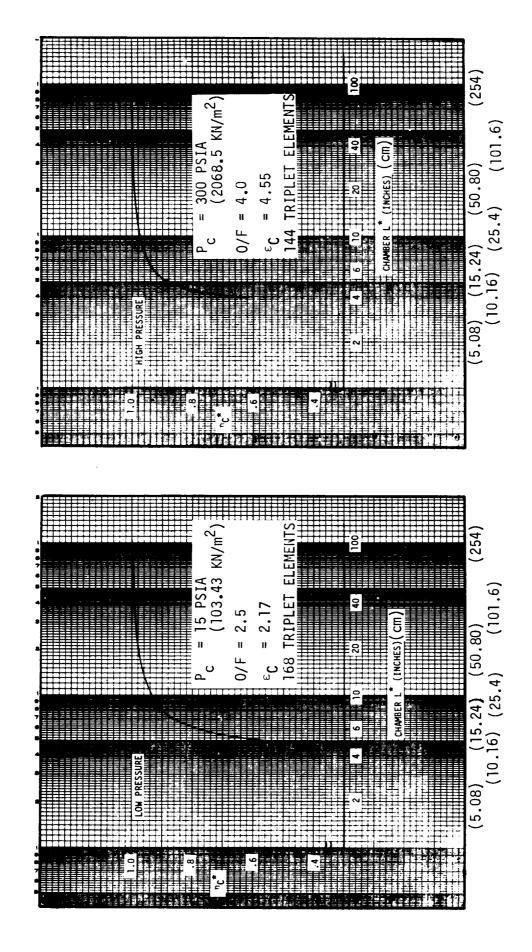
$$\bar{x} = \frac{x}{r_j} = 5.6 \frac{L^*}{\xi_c} \left( n_j \frac{P_c}{F} \right)^{1/2} - (n_j)^{1/2} \left( \frac{\xi_c^{1.5} - 1}{\xi_c} \right)$$
 (41)

Examination of Equations (35) and (41) results in the following conclusions concerning the effects of the various engine parameters on combustion efficiencies:

- 1) a very fine injection pattern is desired
- 2) strong initial mixing is desired
- 3) long chamber length is desired
- 4) low chamber contraction ratio is desired and
  - 5) low ratio of thrust to chamber pressure is desired.

Substituting the appropriate engine parameters for the nominal high and low chamber pressure thruster designs into Equations (35) and (41) results in the combustion efficiency - L\* curves shown in Figure 40. The required characteristic chamber length for the high chamber pressure design to achieve a combustion efficiency of 98% is about 14 inches (35.56 cm). This compares to an L\* of 25 inches (63.50 cm) to achieve  $\eta_{\text{C*}} = 98\%$  in the low chamber pressure design.

The basic results are then modified slightly to provide for uniform flow across the face of the injector. This results in a slight decrease in thrust/element for the inner ring elements. At no time is the thrust/element allowed to exceed the theoretically determined values. As is seen in Figure 40, the 144 element triplet for the high pressure engine is predicted to achieve 99+% combustion performance.



Predicted Gaseous  $\rm H_2/\rm O_2$  Thruster Performance Trends - Thrust 1500 lbf (Thrust = 1500 lbf, 6672 N) Figure 40.

Both a rigorous and an engineering performance model were utilized to predict the performance of the TRW duct cooled concept. In the ready estimating procedure, the procedures outlined in CPIA 178 were followed. The JANAF ODK reference computer program and the CPIA 178 data were used to account for all but the cooling effects. Since the former are standard industry methods, they are not repeated here. The cooling interaction method is of interest, however.

With cooling the  $I_{\mbox{sp}}$  of the core is degraded as follows:

$$I_{Sp}'(core) = I_{Sp}(core) \left[\frac{H_{c} - H_{e} - \Delta h}{H_{c} - H_{e}}\right]^{1/2}$$
where:  $H_{c} = \text{chamber enthalpy at core 0/F}$ 

$$h_{e} = \text{exit enthalpy at core 0/F}$$

$$\Delta h = \text{energy transferred from core}$$

$$to coolant$$

$$X = \frac{m}{cool} \frac{1}{M_{z}}$$

$$Y = \frac{m}{igniter/m_{T}}$$

$$(0/F)_{CORE} = \frac{(0/F)(1 - Y/2) - Y/2}{1 - X - Y(1 + 0/F)/2}$$

Enthalpy in the core is transferred to the film coolant. Treating the problem as a two-zone problem, the coolant  ${\bf I}_{\rm sp}$  contribution is:

$$I_{sp}'(coolant) = I_{sp}(coolant) \left[ \frac{H_c - H_e + \Delta h}{H_c - H_e} \right]_{coolant}$$

Of course, the coolant loses mass and the core gains mass as given by the previously described mixing coefficient ralations. The  $\rm I_{sp}$  of the chamber is computed from:

$$I_{sp}$$
 (TCA) =  $\frac{1}{\dot{w}_T}$   $\left[I_{sp}' \text{ (core)} \cdot \dot{w}_{core} + I_{sp} \text{ (coolant)} \cdot \dot{w}_{coolant} + I_{sp} \text{ (Ig)} \dot{w}_{Iq}\right]$ 

The accuracy of this simplified approach with mixing properly accounted for has been verified in MSFC tests of the duct coolant concept. This correlation was then used to prepare the predicted performance map as shown in Figure 41.

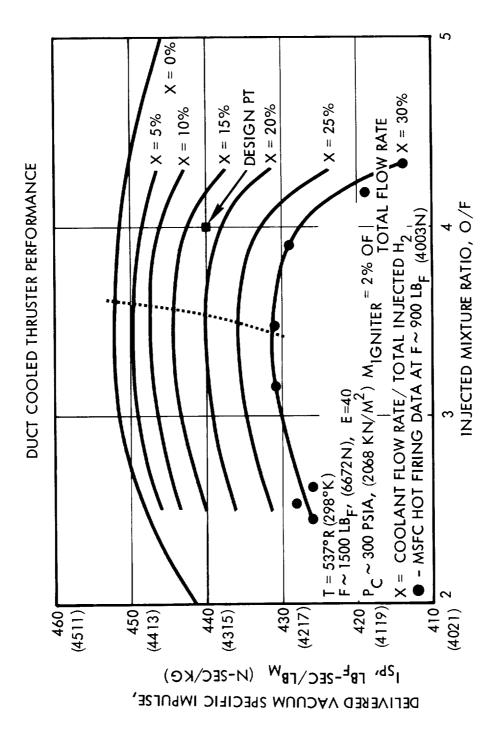


Figure 41. High Pressure Thruster Predicted Performance Map

## 3.5 THRUSTER MECHANICAL DESIGN AND FABRICATION

This section discribes the mechanical design of the thruster and its fabrication. The overall assembly cross-section is given in Figure 42. As seen in the cross-section the valves are shown on a stand off plate for thermal isolation. The catalytic igniter (described in detail in Volume I) is shown in the center of the triplet injector. The igniter nominal flow rate is  $\sim 2\%$  of the total flow, and the flow is controlled by two separate flow control valves. The duct coolant flow and the interior film coolant flow split is controlled by a replaceable flow control ring for the experimental engine. The duct is mechanically clamped in place by the nozzle flange being bolted onto the head end assembly. As shown the duct itself floats freely inside the chamber.

### 3.5.1 <u>Injector</u> Hydraulics

The triplet injector is designed with three rings of  $0_2$ -H<sub>2</sub>- $0_2$  triplet elements. Forty-eight H<sub>2</sub> orifices and 96  $0_2$  orifices make up each of the three rings of triplets. The orifice diameters are varied for each ring to provide a relatively uniform mass distribution over the cross-sectional area. The injection geometry is presented in Figures 43 and 44 for the cold propellant design and for the ambient temperature design, respectively. The internal manifolding is designed for low Mach numbers to reduce circumferential flow variations. For cold propellants, the maximum Mach number at rated flow is 0.06 for the H<sub>2</sub> and 0.09 for the 0<sub>2</sub>, (Figure 45). With ambient temperature propellants, the maximum internal Mach numbers are 0.12 (Figure 46). The injection Mach numbers are 0.47 for the H<sub>2</sub> and 0.41 for the 0<sub>2</sub>, which corresponds to a pressure drop of approximately 50 psi (344.75 kN/m²) at nominal chamber pressure. The internal manifolding is designed for a volume ratio of 4 to reduce mixture ratio transients during pulse mode operation. The dynamic response of the engine is limited by valve response rather than manifold volumes, as indicated in Figure 47.

# 3.5.2 <u>Injector Design and Fabrication</u>

As discussed earlier the injector selection was a raised post triplet with 144 elements. The design detail is given in Figures 48, 49, and 50 for the hole pattern, body, and back body. The injector rings for the experimental injector were OFHC. For the flight thrusters they would be Berylco-10.

The injector inner body assembly is a brazed assembly as shown in Figure 51. The rings are first brazed into the center body. The center body in turn is brazed into the S.S. 304L body.

The back side of the injector assembly is an all welded body, Figure 52. All of these parts were S.S. 304L.

The hole pattern is EDM into the assembly after its final braze and pressure check. The final injector assembly is shown in Figure 53.

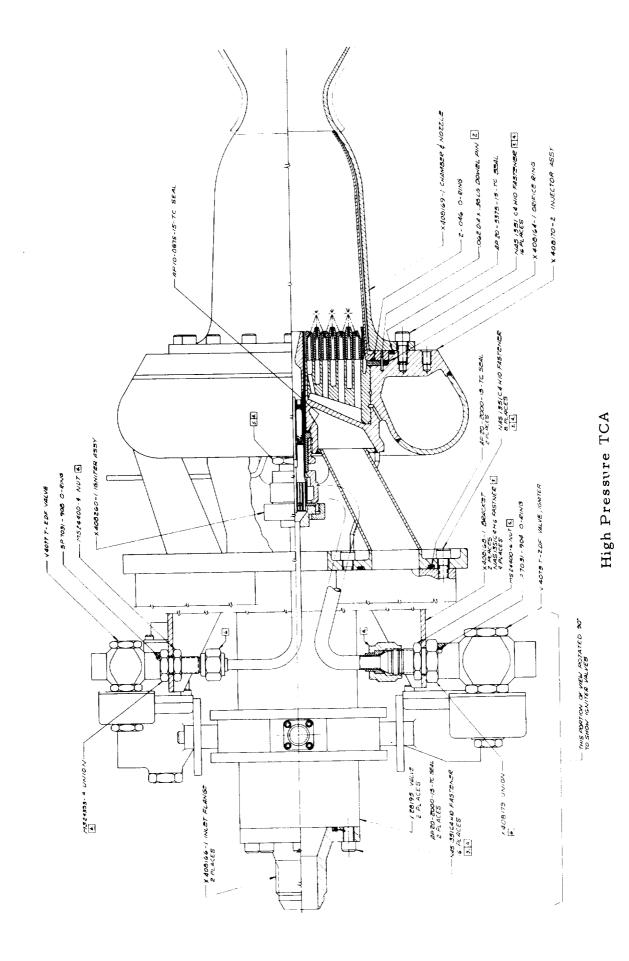


Figure 42. Duct Cooled Thruster Assembly Cross-Section

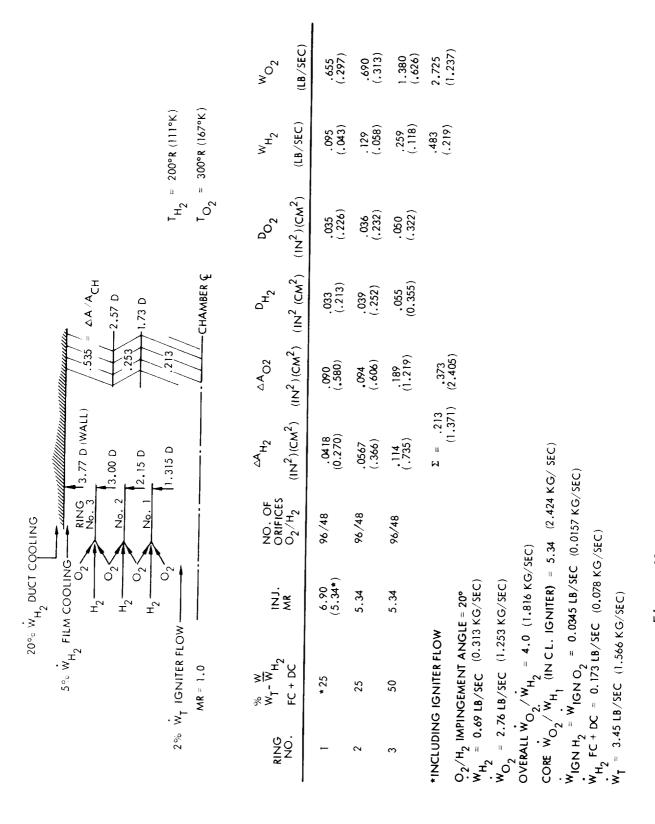


Figure 43. Triplet Injection Parameters (COLD) X408170

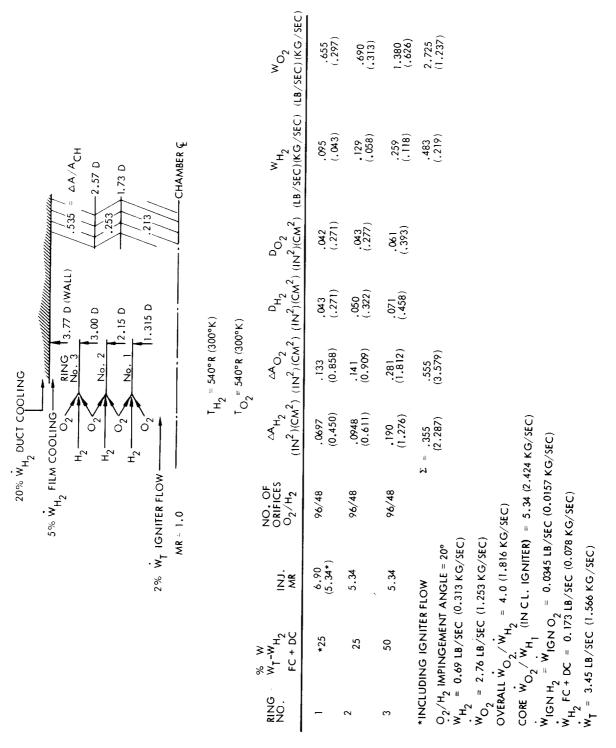


Figure 44. Triplet Injection Parameters (AMBIENT) X408170

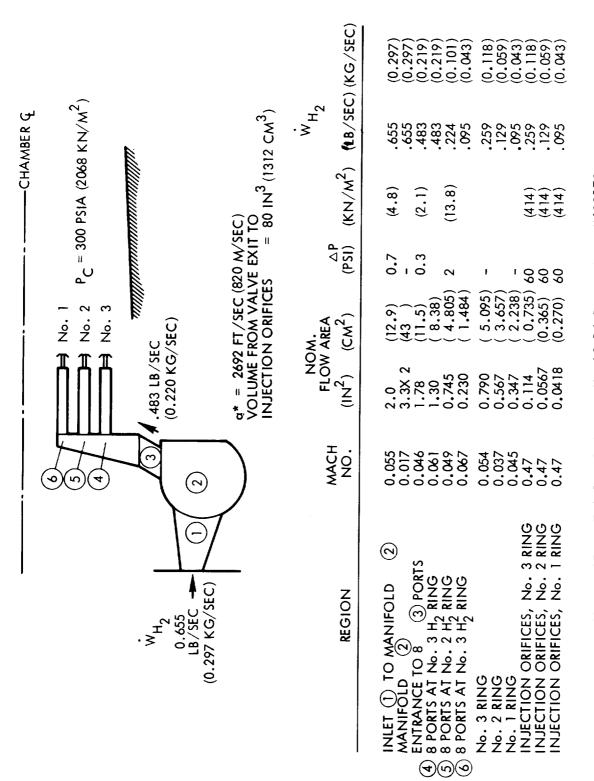
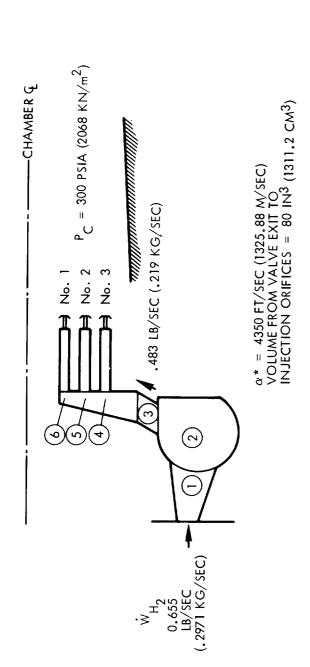


Figure 45a. Triplet Injector Manifold Parameters X408170 Hydrogen Side, TH $_2$  = 200 $^{
m O}$ R (111 $^{
m O}$ K)

$P_{c} = 300 \text{ PSIA}$ (2068 KN/M <sup>2</sup> )		; O <sub>2</sub> (LB/SEC) (KG/SEC)	2.726 (1.238) 2.726 (1.238) 2.726 (1.238) 2.726 (1.238) 0.691 (0.314) 1.032 (0.468) 0.674 (0.306) 0.328 (0.149) 0.691 (0.314)	
—— CHAMBER € No. 1 RING No. 2 (20 No. 3		(KN/M <sup>2</sup> )	(11.7) (11.7) (18.6)	(310) (310) (310)
	;) 20 IN <sup>3</sup> (328 CM <sup>3</sup> )	△P (PSI)	2.1	455 1 1 45 1 1
	SEC) IT = 20 IN <sup>3</sup>	NOM. FLOW AREA $N^2$ ) (CM $^2$ )	(9.7) (12.9) (5.59) (4.00) (2.90) (7.87) (5.68)	(1.38) (0.580) (0.606) (0.897)
4	C (232 M/: VALVE EX ORIFICES	FLOW (IN <sup>2</sup> )	1.5 1.0 × 2 2.0 0.867 0.62 0.45 0.232 1.22 0.88	0.214 0.090 0.094 0.139
© ©	a* = 762 FT/SEC (232 M/SEC) VOLUME FROM VALVE EXIT TO INJECTION ORIFICES = 2	MACH No.	0.09 0.07 0.07 0.085 0.075 0.028	0.082 0.41 0.41
	= *p VOLI TO 11		@	Z Z Z Z Z Z S C C C C C
w <sub>O<sub>2</sub></sub> 2.727 LB/SEC (1.238 KG/SEC)	_	REGION	INLET (1) TO MANIFOLD MANIFOLD (2) DISTRIBUTION PLATE (3) MANIFOLD (4) 8 PORTS (5) 8 PORTS (7) 8 PORTS (9) 18 ORIFICES (10)	INJECTION ORIFICES No. 3 R INJECTION ORIFICES No. 2 R INJECTION ORIFICES No. 1 R

Figure 45b. Triplet Injector Manifold Parameters X408170 0xygen Side,  $To_2 = 300^{9} \mathrm{R} \; (167^{6} \mathrm{K})$ 



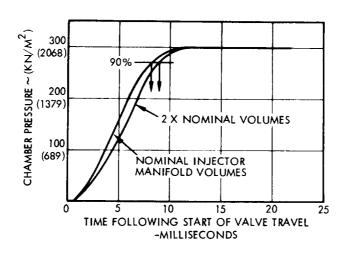
			NOW.		• • • • • • • • • • • • • • • • • • • •
		MACH	FLOW AREA	<u>م</u> <	, T
	REGION	2	$(IN^2)$ $(CM^2)$	(PSI) $(KN/m^2)$	$^2$ ) (LB/SEC) (KG/SEC)
	INLET (1) TO MANIFOLD (2)	0.000	2.0 (12.90)	2 (13.79	655 ( 29711)
	MANIFOLD @	0.028	$3.3 \times 2 (21.29) \times 2$		. 655 (. 29711)
(	ENTRANCE TO 8 ③ PORTS	0.075	1.78 (11.48)	0.7 (4.83)	.483 (.2191)
<del>9</del> (	8 PORTS AT No. 3 H, RING	0.10	1.30 (8.39)	,	.483 (.2191)
(O	8 PORTS AT No. 2 H2 RING	0.081	0.745 (4.81)	5 (34,48)	, 224 (, 1016)
9	8 PORTS AT No. 3 H2 RING	0.11	0.230 (1.48)		.095 (.0431)
	No. 3 RING	0.088	0 790 (5 10)		250 / 1175)
	No. 2 RING	0.081	0.567 (3.66)		129 (10585)
	No. 1 RING	0.073	0.347 (2.24)	ı	. 095 (.0431)
	INJECTION ORIFICES, No. 3 RING	0.47	0.190 (1.23)	60 (413,7)	. 259 (1175)
	INJECTION ORIFICES, No. 2 RING	0.47	0.0948 (.6116)	60 (413.7)	, 129 (, 0585)
	INJECTION ORIFICES, No. 1 RING	0.47	0.0967 (.6239)	60 (413.7)	.095 (.0431)

Figure 46a. Triplet Injector Manifold Parameters X408170 Hydrogen Side, TH $_2$  = 540°R (300.02°K)

——————————————————————————————————————		
•	<sup>₩</sup> O <sub>2</sub> (LB/SEC) (KG/SEC)	-

	.≽	727 IR/SEC	(1.237 KG/SEC)	2.726 (1.238)	2,726 (1,238)	2,726 (1,238)	2,726 (1,238)	૽ૺ૽	1.032 (.468)	0.674 (.306)	0.328 (.149)	ٺ	1.032 (.468)	0.674 (,306)	0,328 (,149)	1.381 (.627)	0.691 (.314)	0.655 (.297)
( ) ( ) ( ) ( ) ( ) ( )		ΔP	(PSI) $(KN/m^2)$	3.7 (25.51)	1	3,1 (21,37)	•		5 (34.43)		_	1	1	•	•	45 (310)	45 (310)	45 (310)
(10)	NOM.	FLOW AREA	$(IN^2)$ $(CM^2)$	1.5 (9.7)	$1.0 \times 2 (2.54) \times 2$	2.0 (12.9)		0.867 (5.59)	0.62 (4.00)	0,45 (2,90)	0,232 (1,50)	1,22 (7,87)	0.88 (5.68)	0.55 (3.55)	0.214 (1.38)	0,281 (1,81)	0,141 (.9097	0.133 (.858)
		MACH	NO.	0.12	80	60.0		0.053	0.114	0.102	0.10	0.038	0.081	0.084	0.11	0.41	0.4	14.0
			REGION	INIET (1) TO MANIFOLD (2)		DISTRIBUTION PLATE (3)	MANIEOI D 4	8 PORTS	8 PORTS	8 PORTS	18 ORIFICES				,	INJECTION ORIFICES No. 3 RING	INJECTION ORIFICES No. 2 RING	NJECTION ORIFICES No. 1 KING
								(F)	O		@	6	9		3			

Figure 46b. Triplet Injector Manifold Parameters X4081700xygen Side,  $T0_2 = 540^{\circ} R$  (300.02°K)



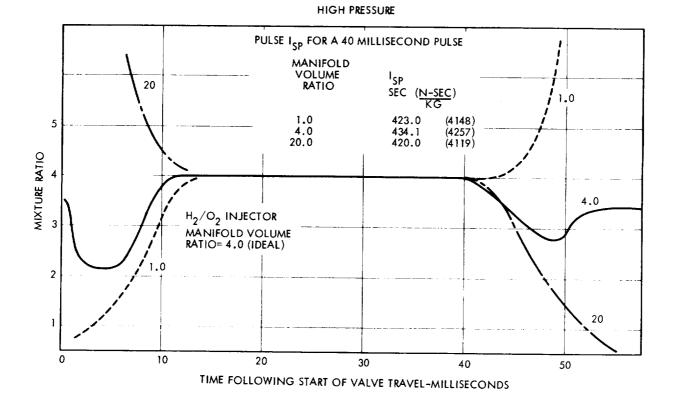


Figure 47. Variation in Pulse Mode Mixture Ratio with Injector Manifold Volume Ratio High Chamber Pressure Thruster

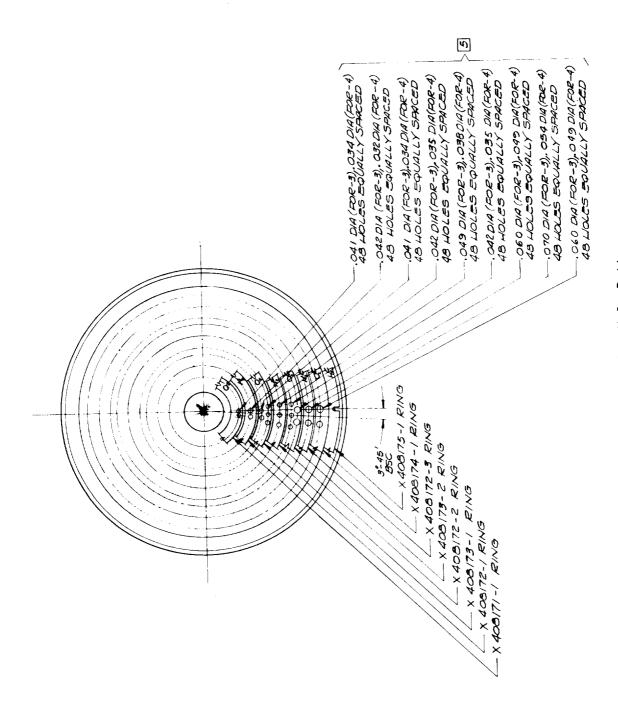


Figure 48. Triplet Hole Pattern

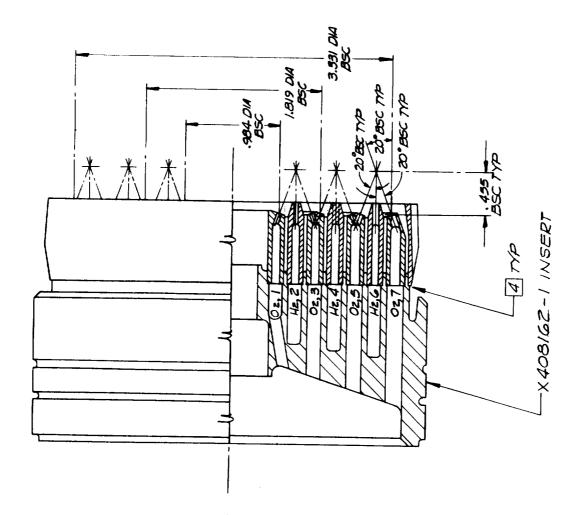
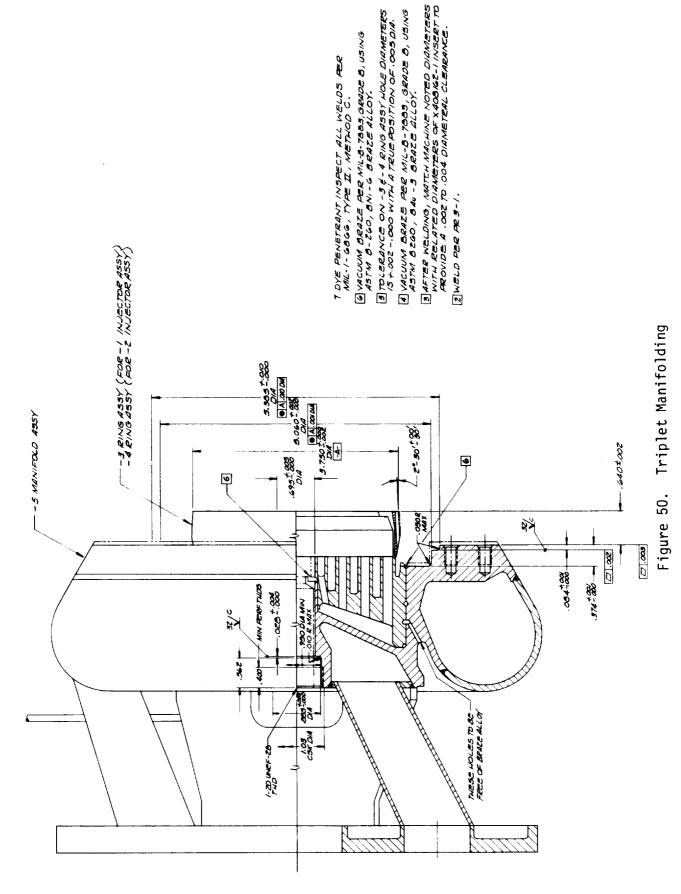


Figure 49. Triplet Body Insert With Rings



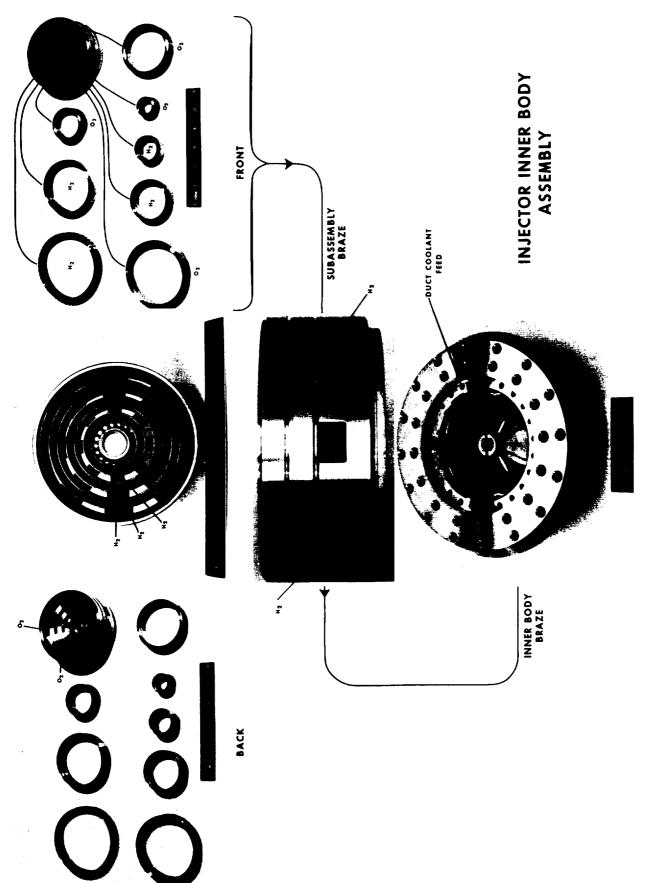


Figure 51. Injector Center Body Assembly

ALL PARTS EB WELDED IN PLACE

Figure 52. Injector Manifold Assembly

Figure 53. Triplet Injector Assembly

#### 3.5.3 Duct Design and Fabrication

The duct is a single piece Berylco-10 configuration with 90 channels The design detail is shown in Figure 54 and the finished product is shown in Figure 55.

### 3.5.4 <u>Nozzle Design and Fabrication</u>

The nozzle is a single piece A-286, 40:1 nozzle for the altitude firings. Its design is shown in Figure 56. The nozzle is an 80% bell contour. The chamber has a nearly 3° taper in it to provide a guiding surface for the duct. The Section B is at an area ratio of 12:1. Short S.S. nozzles of 12:1 were used for sea level checkout firings. The nozzles were machined from pierced billets. A finished nozzle is shown in Figure 57.

#### 3.5.5 <u>Catalytic Igniter</u>

The high pressure catalytic igniter selection was based upon design guidelines established during the igniter scaling analysis described in Volume I of this contract report. The downstream  $0_2$  injection technique was incorporated to minimize overall igniter response time, as experimentally determined during the response enhancement investigation (Section 3.2.2., Volume I).

The high Pc igniter design is shown schematically in Figure 58. The volumes and flow resistances were selected to allow  $0_2$  to pulse through the system and diffuse backwards through the downstream end of the catalyst bed, this  $0_2$  being intercepted by the low MR (less than 1:1)  $H_2$ - $0_2$  mixture from the upstream end. The pneumatic design of the unit was investigated by analog modeling of the results from the igniter scaling analysis. Overall mixture ratio is 1:1, with 90 percent of the total  $H_2$  flow utilized for cooling the reactor combustion chamber.

Ten percent of the igniter  $H_2$  and  $O_2$  passes through the catalyst bed at a MR of 1:1 or lower. The remaining 90 percent of the  $O_2$  is injected downstream of the catalyst bed to provide high response ignition and to raise the local MR to 10 O/F to provide a high temperature effluent for reliable main thruster ignition.

## 3.5.6 Overall Thruster Assembly

The overall assembly drawing is shown in Figure 59. This figure gives all major dimensions. The actual thruster assembly is shown in Figure 60.

## 3.5.7 Thruster Material Tradeoff Summary

The material selection philosophy for the thruster assembly nozzle was based on a philosophy which would not allow any new material development requirements. The pro and con factors considered are given in Table 8. Effects of  $\rm H_2$  are shown in Figure 61. From these results the primary material selection were taken as A-286 for the nozzle and Berylco-10 for the duct.

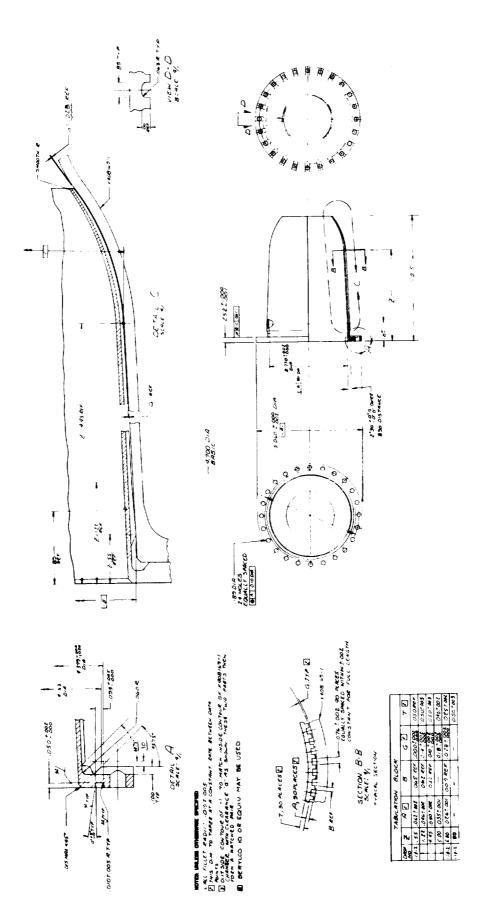


Figure 54. Duct Design Details

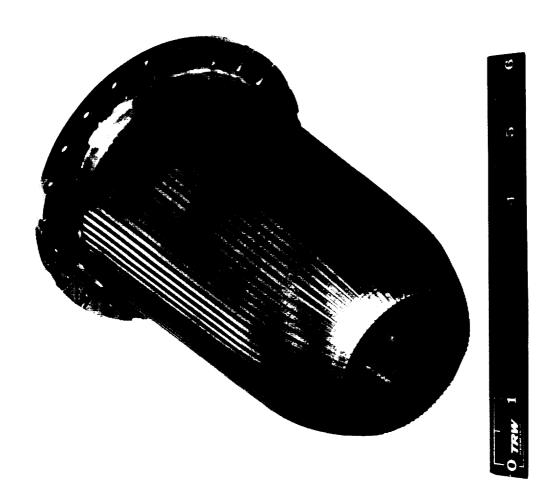


Figure 55. Finished Berylco-10 Duct (Weight 1.90 lb)

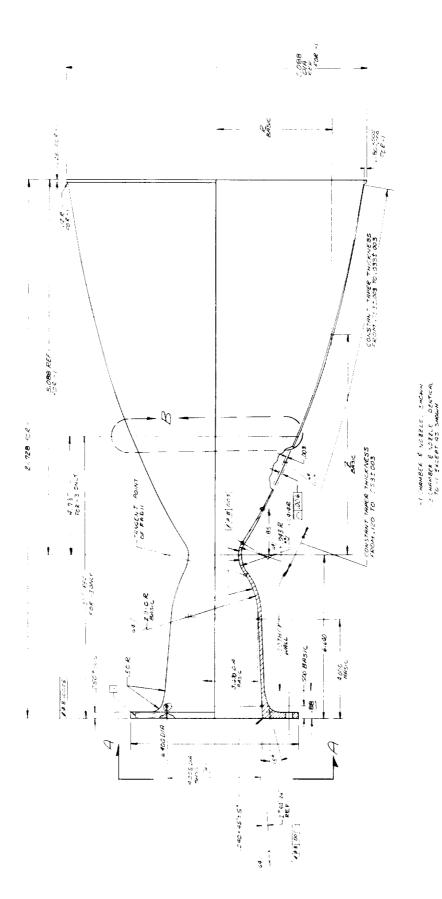


Figure 56. 1500 lb $_{
m f}$  (6672 N) Nozzle and Chamber Design

Chamber



• FLIGHT ENGINE NOZZLE  $\Delta X_{thrust} = 0.050$ " (.127 cm)  $\Delta X_{exit} = 0.020$ " (.0508 cm) - 0.030" (.0762 cm)

WEIGHT 9.4 lbs (41.81 N)

Figure 57. Finished A-286 Single Piece Nozzle

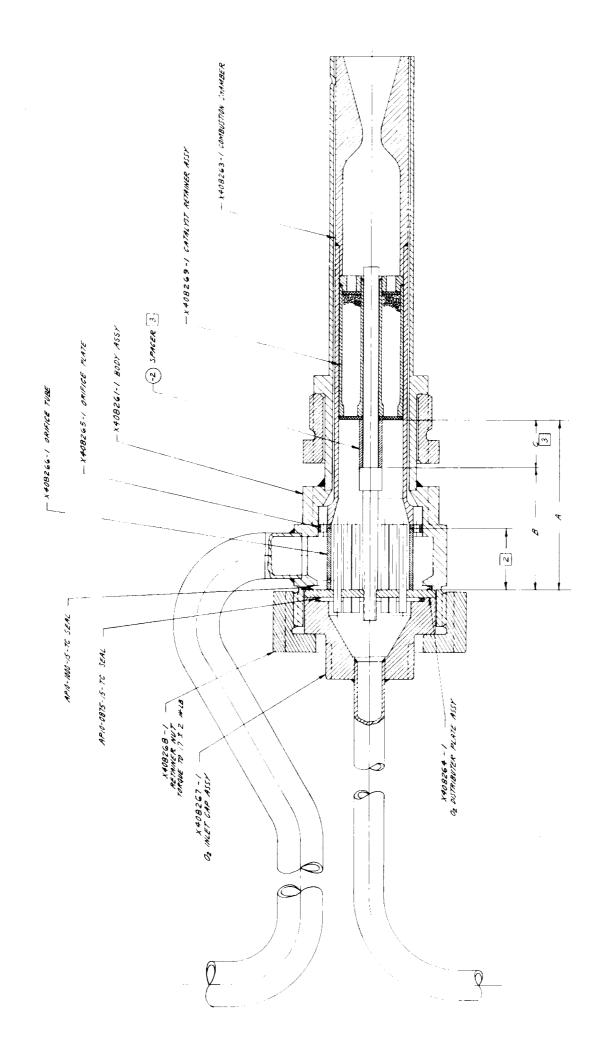


Figure 58. Catalytic Reactor - Downstream  $\mathbf{0}_2$  Injection

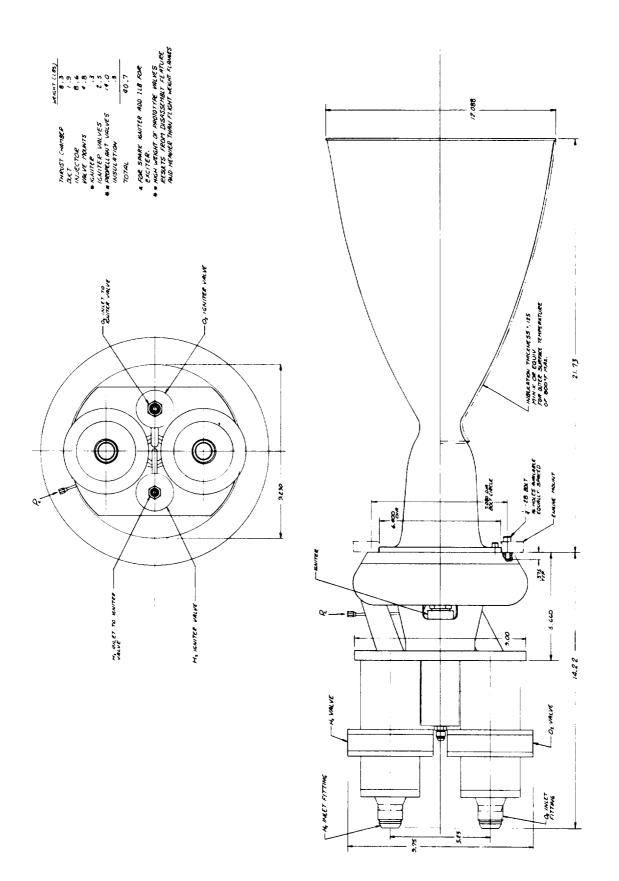


Figure 59. Overall Experimental TCA, With the Experimental Weights Included.

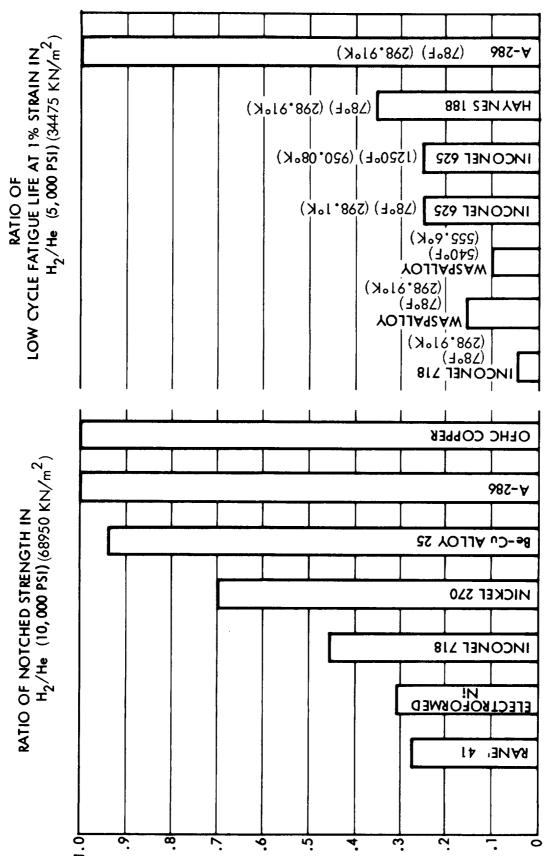


# Table 8. Thruster Material Tradeoff Summary

PRO

CON

A-286	READILY FORMABLE AND WELDABLE  NOT SENSITIVE TO 02, H2  RESISTANT TO CORROSION  EXCELLENT MECH. PROP. TO 1350°F(1005.64°K)  FTY 60 KSI (413.7 KN/m²)  READILY AVAILABLE IN BAR, SHEET, WIRE, PLATE	STRENGTH IS REDUCED WITH USE ABOVE 1350°F (1005.64°K) THERMAL CONDUCTIVITY IS ONLY FAIR
	THERMAL-PHYSICAL PROPERTIES EXTENSIVELY STUDIED CONSIDERABLE CASE STUDY INFORMATION	
r-605	READILY FORMABLE AND WELDABLE  RESISTANT TO CORROSION  GOOD HIGH TEMP. PROP. 30 KSI  (206.85 KN/m²) AT 1350°F(1005.64°K)  READILY AVAILABLE IN SHEET, BAR, WIRE THERMAL PHYSICAL PROP. WELL CHARACTERIZED  WIDE USAGE IN HIGH TEMP. STRUCTURES	SCANT DATA ON EFFECT OF H <sub>2</sub> ENVIRONMENTS, PARTICULARLY IN WELDS AND <sup>2</sup> HAZ FATIGUE STRENGTH IS REDUCED IN H <sub>2</sub> ENVIRONMENT, NOT SENSITIVITY PROBABLY INCREASED THERMAL CONDUCTIVITY IS ONLY FAIR
ELECTROFORMED NICKEL	• CORROSION RESISTANT • FORM COMPLEX STRUCTURES READILY • GOOD THERMAL CONDUCTIVITY • EXCELLENT DUCTILITY (IN AIR)	POOR HIGH TEMP. STRENGTH 5 KSI (34.48 KN/m <sup>2</sup> ) AT 1350°F (1005.64°K) SENSITIVE TO H <sub>2</sub> ENVIRONMENT PROCESS SENSITIVE, WIDE SPREAD IN MECH. PROPERTIES LITTLE HISTORICAL DATA IN CRITICAL APPLICATIONS
TD NICKEL	• CORROSION RESISTANT • GOOD HIGH TEMP. PROP. 32 KSI (220.64 KN/m²) AT 1350°F (1005.64°K) • GOOD THERMAL CONDUCTIVITY • APPEARS INSENSITIVE TO H2	PROPERTIES AND USAGE NOT WELL CHARACTERIZED LIMITED AVAILABILITY JOINING PROBLEMS



Effect of Hydrogen on Selected Properties of Some Candidate Thruster Materials 6]. Figure

# 3.5.8 Low Pressure Thruster Design

The low pressure thruster design cross-section is included here for reference purposes. It is shown in Figure 62.

# 3.6 THRUSTER/SYSTEM INTERACTION ANALYSIS

Analysis of the pulse mode operation of the 1500 lbf (6672 N) thrusters was performed for each set of operating conditions previously specified in Table 1. The minimum impulse bit (MIB) capability of the complete thruster/igniter assembly (including igniter-only operation) was determined analytically, and the environmental effects on MIB and steady-state performance were also evaluated, as described in the following sections.

# 3.6.1 Thruster/Igniter MIB Analysis

MIB analyses were conducted using lumped parameter dynamic computer models developed to simulate flow rates, pressures, and propellant reactions throughout the actual thruster, igniter, and propellant feed system volumes. Comparison of these model outputs with test firing data has shown good correlation.

Figure 63 presents typical computer data comparing igniter-only and full thruster pulse-mode operation. Response times of 10 milliseconds for the igniter valves and 25 milliseconds for thruster valves were selected for this particular computer run. Data from a simulated full thruster firing without and with feed systems are presented in Figures 64 and 65, respectively, indicating the increase in MIB caused by feed line volumes.

The results of the full thruster MIB analyses are summarized in Figure 66 for the high pressure, 1500 lbf (6672 N) thruster. Although propellant supply pressure and temperatures have some effect, thruster MIB is mainly a function of valve response time, as indicated in Figure 66.

Igniter-only MIB analyses are presented in Figure 67. MIB is obviously a direct function of igniter flow rate, expressed as a function of nominal overall thruster flow rate in Figure 67. Igniter valve response effects are indicated in Figure 67. The results in Figure 67 indicate that manifold volumes have little effect on MIB. Igniter-only MIB is affected mainly by valve response time and, of course, by igniter flow rate.

MIB analyses were also performed for the 1500 lbf (6672 N), 15 psia (103 kN/m $^2$ ) thruster. Design and analysis only were conducted for this thruster; no hardware fabrication or testing was performed during this program.

The results of the MIB analyses for the high thrust, low Pc thruster are summarized in Figure 68. Again, one of the major factors affecting thruster and igniter MIB is valve response time, as shown in Figure 68.

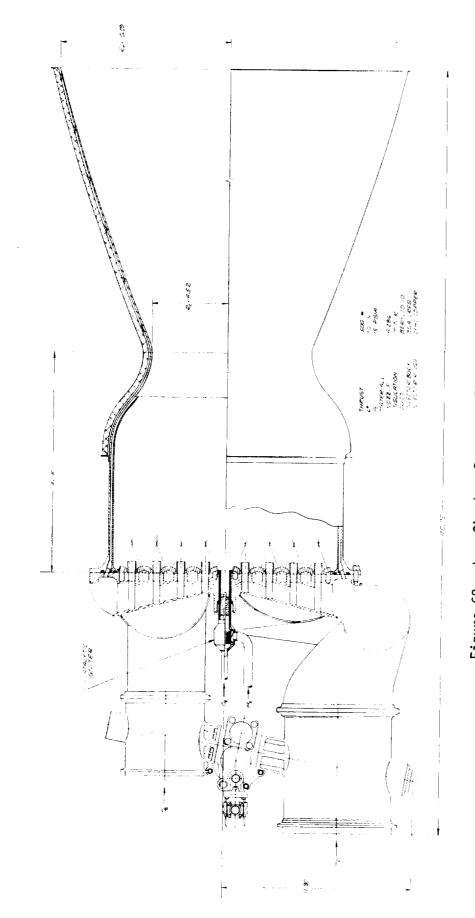


Figure 62. Low Chamber Pressure Thruster Design

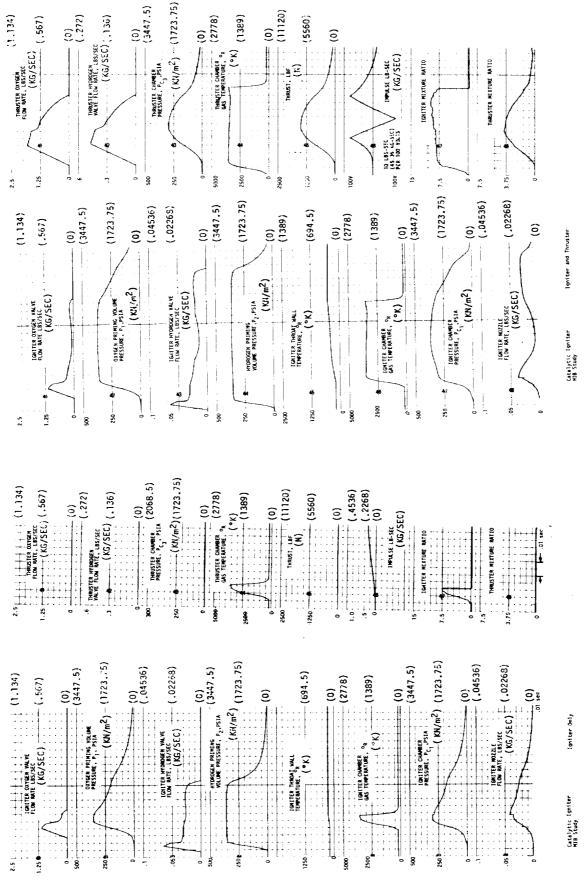
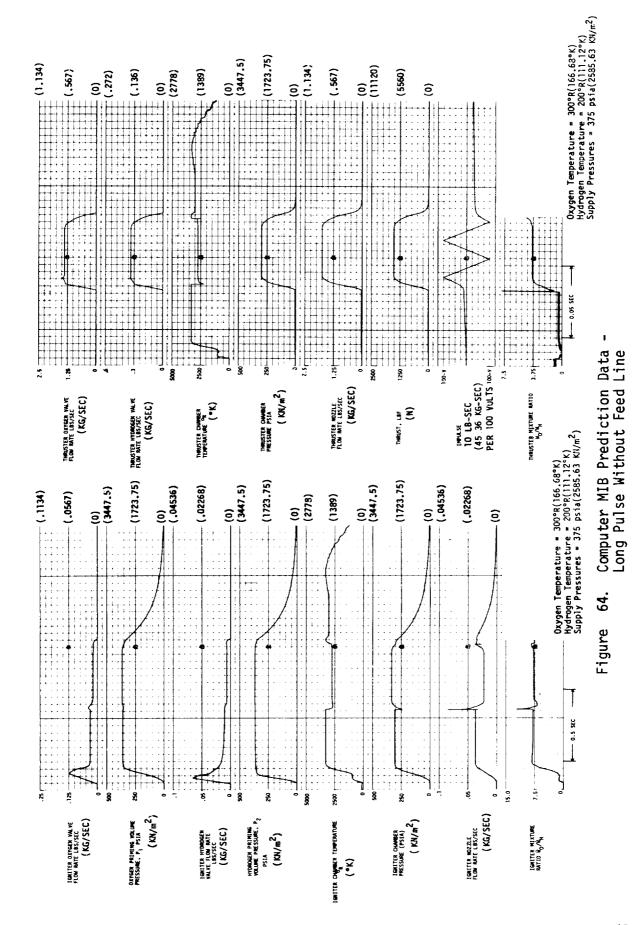
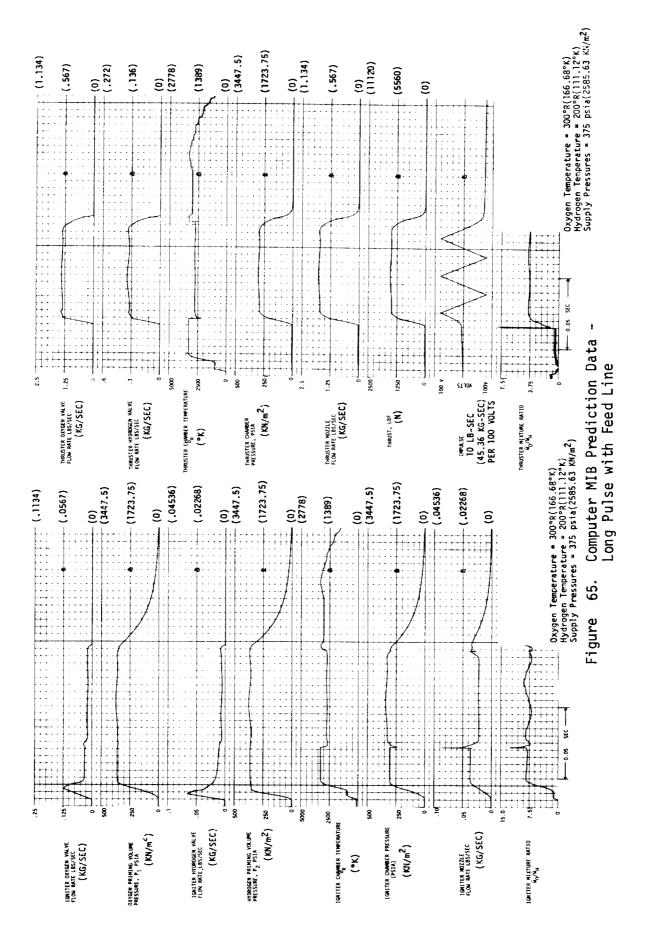
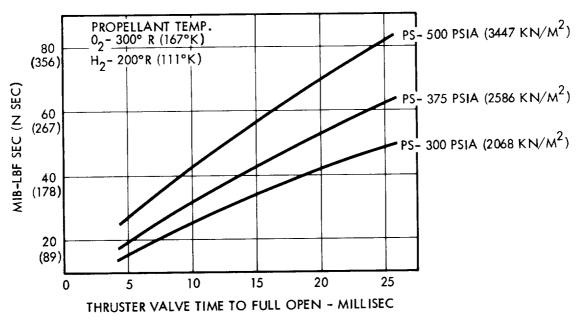


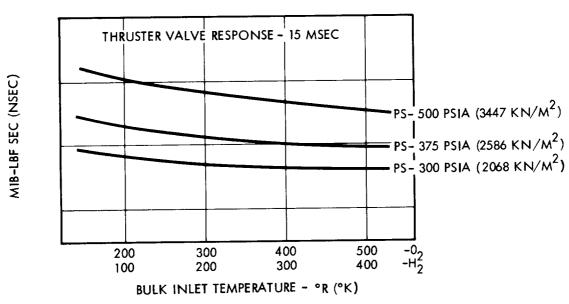
Figure 63. Computer MIB Prediction - Igniter-Only . Operation Compared to Full Thruster





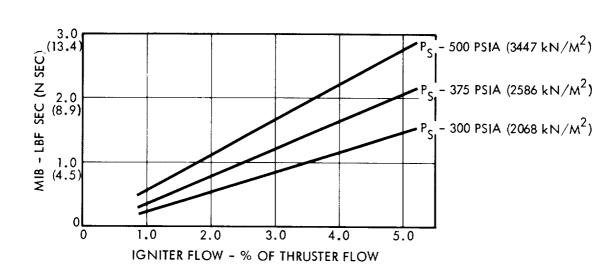


(a) THRUSTER MIB VERSUS VALVE RESPONSE AND SUPPLY PRESSURE

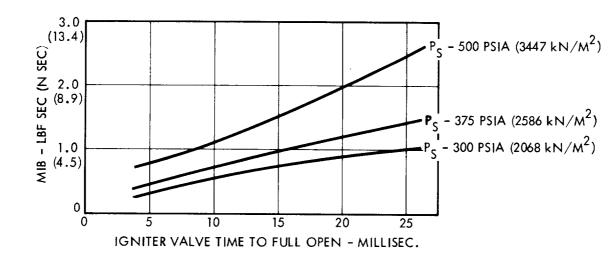


(b) THRUSTER MIB VERSUS PROPELLANT TEMPERATURE AND SUPPLY PRESSURE

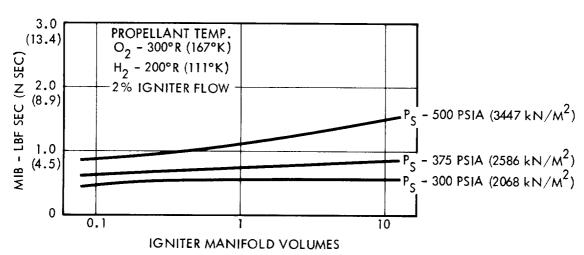
Figure 66. Thruster MIB Analysis - High Pc



(a) IGNITER MIB VERSUS FLOW RATE AND SUPPLY PRESSURE

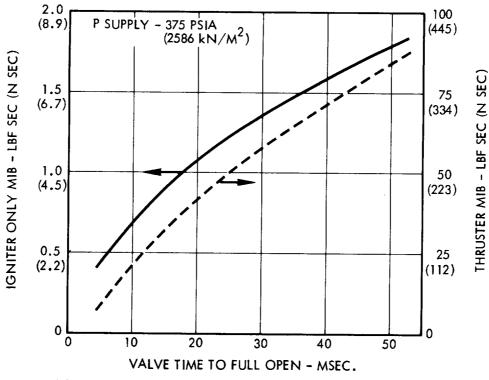


(b) IGNITER MIB VERSUS VALVE RESPONSE AND SUPPLY PRESSURE

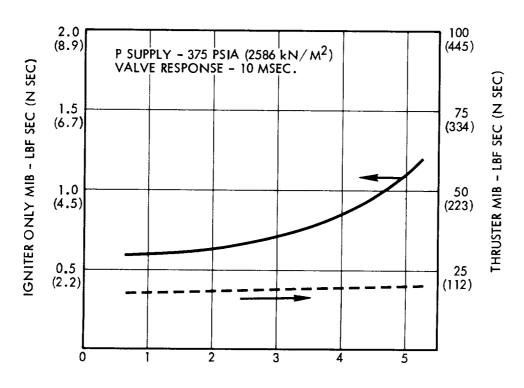


(c) IGNITOR MIB VERSUS MANIFOLD VOLUMES AND SUPPLY PRESSURE

Figure 67. Igniter-Only MIB Analysis - High Pc



(a) THRUSTER AND IGNITER MIB VERSUS VALVE RESPONSE



(b) THRUSTER AND IGNITER MIB VERSUS IGNITER FLOW RATE

Figure 68. Thruster/Igniter MIB Analysis - Low Pc

Figure 68 indicates that igniter flow rates have a significant effect on igniter-only MIB but have a negligible effect on MIB if the full thruster is fired, as expected.

The overall thruster/igniter MIB analysis results indicate that thruster or igniter MIB is largely a function of valve response time. Extremely low MIB values for thrusters of this size (less than 1.0 lbf sec [4.448 N/sec]) were shown to be attainable by firing the igniter only. These MIB predictions were later verified by full thruster and igniter-only test firings, as described in Section 4.

4.	HIGH	PRESSURE	THRUSTER	EVALUATION	TESTS	

# 4. HIGH PRESSURE THRUSTER EVALUATION TESTS

The following test series were conducted.

- Injector performance tests with a copper heat sink chamber, with no duct installed, at sea level conditions to document core c\* performance and heat transfer to the chamber walls over the mixture ratio range with ambient temperature propellants.
- Combustion performance tests with a duct cooled stainless steel chamber at sea level conditions to document c\* performance at different duct flow levels over the mixture ratio range.
- Altitude tests with ambient temperature propellants and with cold propellants to document specific impulse performance and thrust chamber temperatures at different duct cooling flow levels over the mixture ratio range.
- Pulse mode tests to document minimum impulse bit performance and duct cooling characteristics.
- Igniter-only tests to document minimum impulse bit performance and cooling requirements.

Test description and test results are presented in the following sections.

### 4.1 PERFORMANCE TESTS

4.1.1 Injector Performance and Uncooled Thrust Chamber Heat Flux Determination

# 4.1.1.1 Test Description

Tests were conducted in the TRW Vertical Engine Test Stand VAl, which was used for both sea level and altitude tests (Figure 69). The engine installation with the heat sink chamber is shown in Figure 70. Engine instrumentation is shown in Figure 71. The primary flow measurement consisted of sonic orifices, with close coupled upstream pressure and temperature measurement. The sonic orifices were calibrated in the test stand using an NBS calibrated Quantum Dynamics flowmeter with the pressure and temperature instrumentation used for the engine tests. Heat flux was determined using temperature measurement of the external surface of thermal isolation plugs. Heat flux measurement location and Photocon installation location are indicated in Figure 72. Igniter flow and main propellant flow were controlled by Circle Seal valves and Flowdyne valves, respectively. A photograph of the valves mounted on the injector is presented in Figure 73. A spark igniter installation, used for the injector performance tests, is shown in the photograph. Close-up photographs of the igniter and main propellant valves are shown in Figure 74.

Figure 69. Overall View of High Thrust Engine Stand

Figure 70. Test Installation of Heat Sink Thrust Chamber

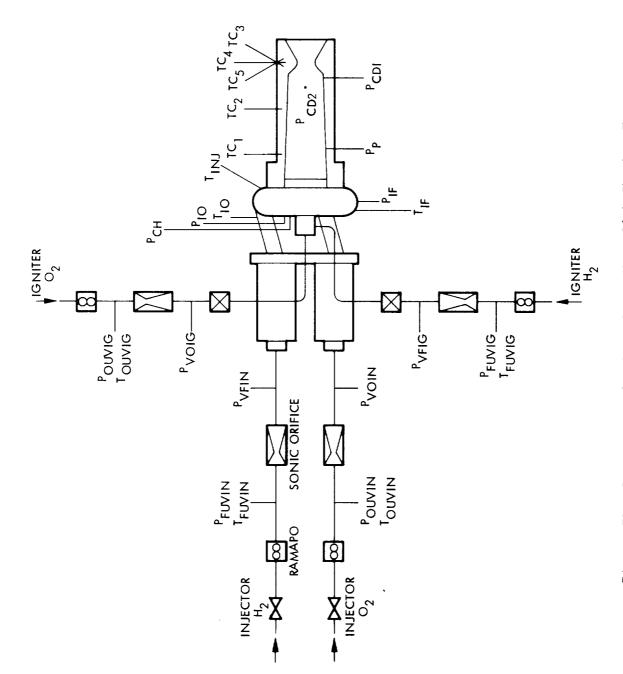
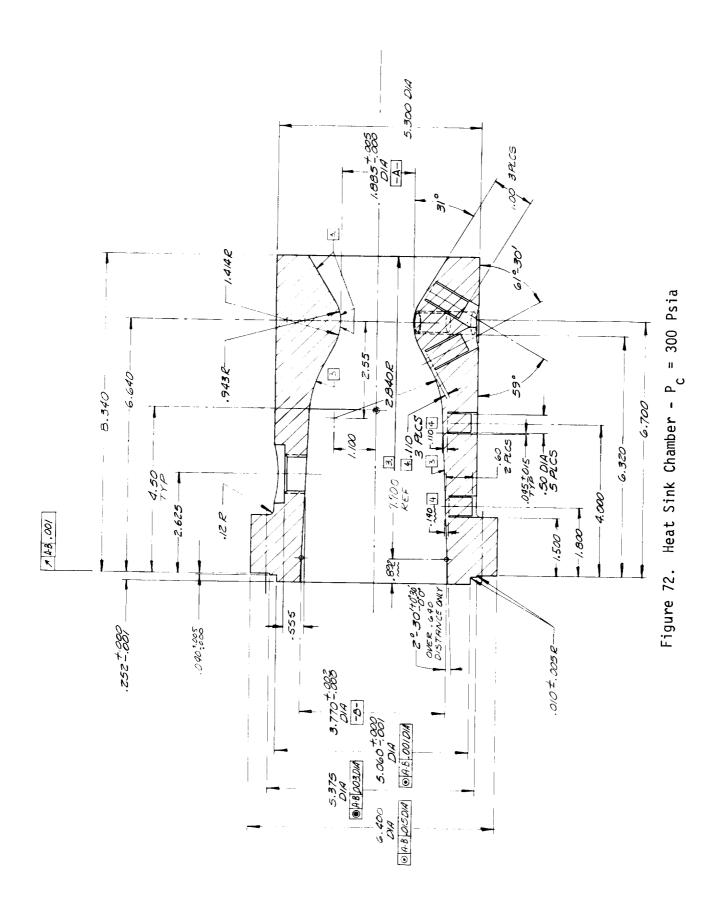


Figure 71. Instrumentation Location - Heat Sink Chamber Tests



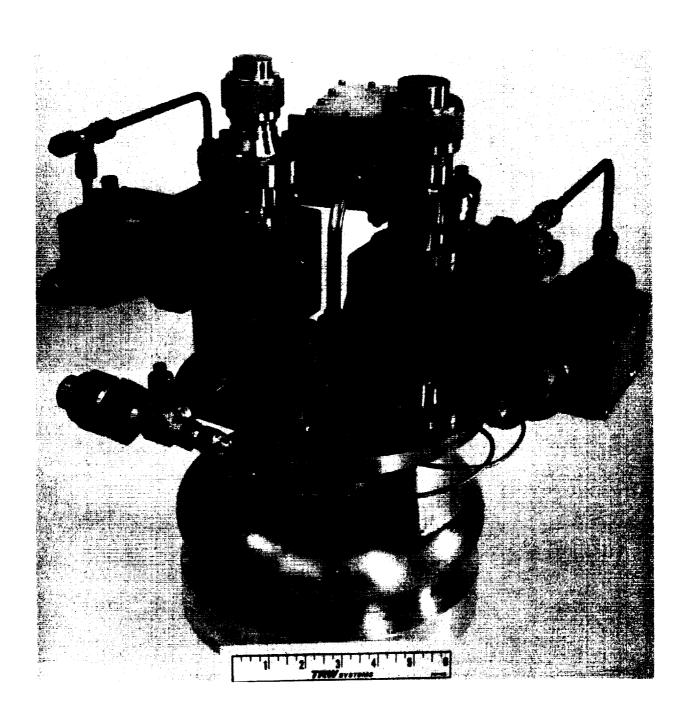


Figure 73. Head End Assembly, Electrical Ignition (TRW Supplied)

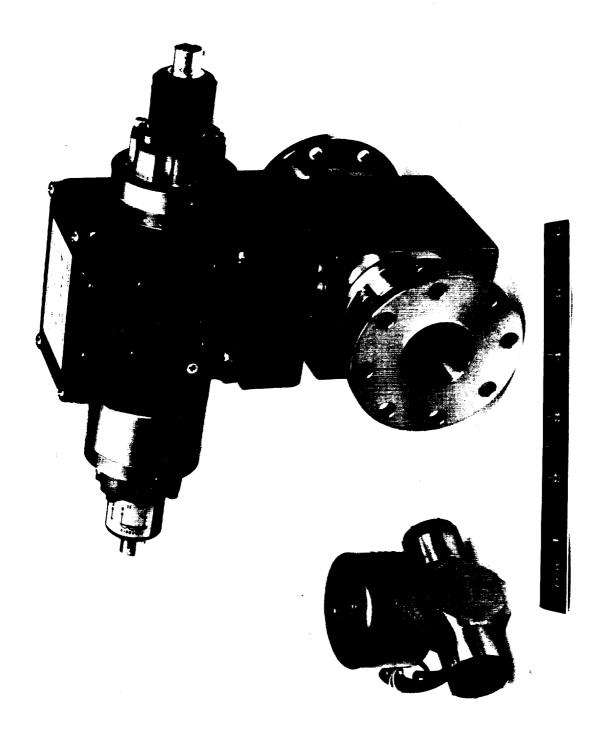


Figure 74. Flodyne and Circle Seal Valves

Prior to the hot firing tests the injector was water flowed to determine if internal manifolding was free of burrs, chips or braze material. A photograph of the water flow indicating the uniformity of the flow pattern is presented in Figure 75.

### 4.1.1.2 Test Summary

A summary of the performance data is presented in Table 9. Core mixture ratio was varied from 3.8 to 6.0 which would correspond to an overall mixture ratio of 2.9 to 4.5 with 25 percent duct cooling flow. Corrected core c\* efficiency is essentially 100 percent over the mixture ratio range (Figure 76). (The calculation procedure is presented in Appendix A.) The chamber heat flux determined from the temperature measurements was consistent with chamber design values (Figure 77). The igniter flow was approximately 2 percent of the total propellant flow. A hydrogen flow of approximately 5 percent of the total hydrogen flow was injected along the chamber walls.

The rapid dynamic response of the thruster during start transients is indicated by the oscillograph traces presented in Figure 78. Chamber pressure overshoot was minimal as indicated by the photocon traces.

The short duration tests with the triplet injector indicated a recirculation of gas from the igniter occurred at the inner oxidizer ring. The recirculation was reduced by extending the steel sleeve of the igniter so that the tip extended 1/8 inch (.3175 cm) beyond the face ( $H_2$  rings) of the injector. This reduced the hot gas recirculation, but the inner oxidizer ring required a clean up cut of 0.033 inches (.0838 cm) after the test series. A redesign was proven to be satisfactory in the longer firing duration test series with the duct cooled chamber, as described in the following section.

### 4.1.2 Cooled Thruster Performance Tests

# 4.1.2.1 Steady State Firings Installation

Sea level tests were conducted with a duct cooled stainless steel thrust chamber ( $\epsilon$  = 12) to determine C\* and thermal performance. The configuration allowed rapid visual inspection of the installed duct and injector face between firings. A photograph of the engine installation is presented in Figure 79. Flow and injector pressure instrumentation was essentially the same as for tests with the heat sink chamber. The hydrogen flow to the duct was calculated from measurement of the pressure differential across the replaceable duct flow control orifice ring. Chamber pressure was measured at the injector face.

# 4.1.2.2 Test Summary

The test results are summarized in Table 10. The C\* efficiency is 94 to 95 percent at a ratio of duct cooling flow of 27 percent of the total hydrogen flow at a mixture ratio of 4.2 (Figure 80). The C\* performance increases to 98 percent as duct flow is decreased to 16 percent of the total hydrogen flow. Once confidence had been established in the firing procedure, the catalytic igniter was substituted for the electrical igniter as indicated in Table 10.

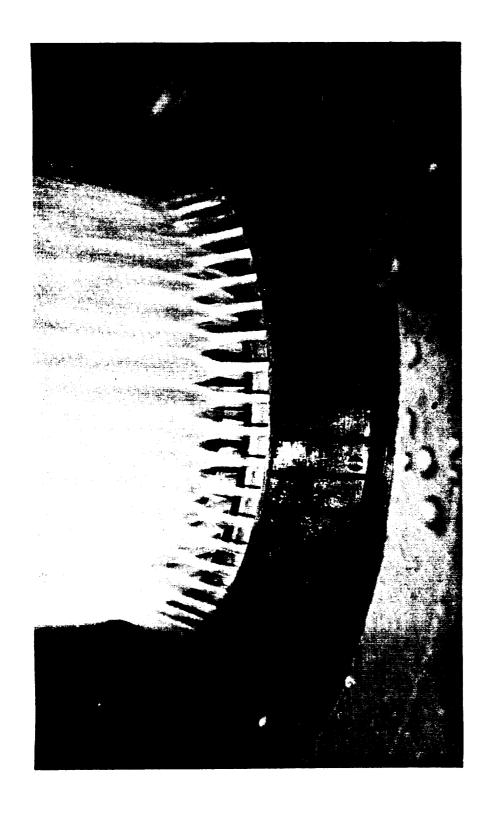


Figure 75. Water Flow of Injector - Oxidizer Side

Table 9. Triplet Injector Performance

	Comments	igniter H2 coolant recirculation burned igniter tip, extended igniter coolant tip 1/8" after Test 821	- Firex triggered, no igniter operation because of wet stand	Snark ning changed after Test 830			No digital tape											Recirculation at onner oxidizer ring required cleanup out of 0.033 inch after Test 848	
ا م	(3)	0.60		99.41	99.38	99.42	· ·	99, 57	100,1	1 00.1	1000.0	99. 7	6.66	100.0	1 00 1	100.9	100.6	100.6	100.8
	m/sec	2482.8		2473.7	2471.1	2470.9	0.7.0	2422, 1	2403.0	2401.2	2396.0	2380.2	2347.9	2343.9	2331.0	2535.6	2520.7	2520.9	2519.2
Carrected	it/sec	8145. 6		8115.7	8107.2	8106.7	7 .70 10	7946.7	7884.0	7877.8	7860.9	7809.0	7713.1	7690.1	7647.8	8319.6	8270.1	870.8	8265, 2
Mixture	(O/F)	3.837		4.149	4.173	4.190	4. 220	4.839	5, 203	5.214	5.258	5,345	5,652	5, 799	5, 967	3, 791	3, 807	3, 921	3, 991
	kg/se.			. 015	. 015	. 015	610.	. 015	. 015	. 015	. 015	. 015	. 01.9 	. 015	. 015	. 015	. 015	. 015	. 015
Flow H	lb/sec	. 032		. 032	. 032	. 032	750.	, 032	. 032	. 032	. 032	. 032	250.	. 032	. 032	. 032	. 032	. 032	. 032
igniter Flow	kg/se∈	. 014		410.	. 014	. 014	<b>*</b>	. 014	. 014	. 014	. 014	. 014	010	. 015	. 015	. 015	. 014	. 014	. 014
Ó	lb/sec	. 031		. 031	. 031	. 031	160.	. 031	. 031	. 031	. 031	. 031	. 032	. 032	. 032	. 032	. 031	. 031	. 031
	kg/aec	. 266		. 238	. 235	. 233	. 231	. 206	961.	. 195	. 192	. 188	781.	176	. 172	. 264	. 262	. 256	. 249
r Flow H	lb/sec	. 586		. 524	. 517	. 514	604.	455	. 431	. 429	. 424	. 415	397	388	. 380	582	. 578	. 565	. 550
Injector Flow	2 kg/sec	1.061		1. 031	1.024	1.024	1.022	1.055	1.078	1.076	1.074	1.071	1.099	1.090	1.101	1.941	1.039	1.040	1.040
С	lb/sec	2.340	į	2, 273	2, 258	2.258	4, 254	2, 325	2.376	2, 373	2,367	2.360	2.422	2.4	2.427	2. 294	2, 291	2, 293	2, 292
)	kN/m <sup>2</sup>	1857.5		1808.6	1793.4	1791.3	1784.4	1761.0	1763.7	1759.6	1748.6	1727. 2	1740.3	1711.3	1711.3	1905, 8	1900.3	1883.0	1871.3
PCD	1bf/in <sup>2</sup>	269.4		262.3	260.1	259.8	258.8	255.4	255.8	255.2	253.6	250.5	252.4	248 2	248.2	276.4	275. 6	273.1	271.4
	Duration (sec)	0.9	0,35	0.7	0.7	_				-					-	1.0	: -		-
	Test No.	821	826	828	831	832	833	835	837	838	839	840	841	247	844	845	44.6	74.8	848

(1) Tests for which data are not shown did not yield reliable steady state data Test Configuration: X408170-3 Triplet Injector (ambient temperature) X408258 Heat Sink Chamber Flodyne Propellant Valves Circle Seal Igniter Valves X408322 Spark Igniter

Propellant Temperature: 80°F (300°K)

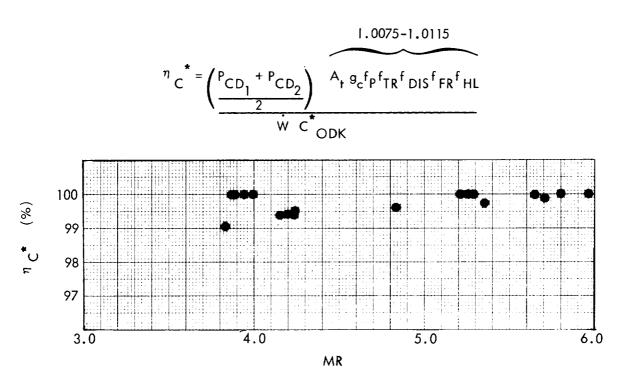


Figure 76. Triplet Injector Performance Summary

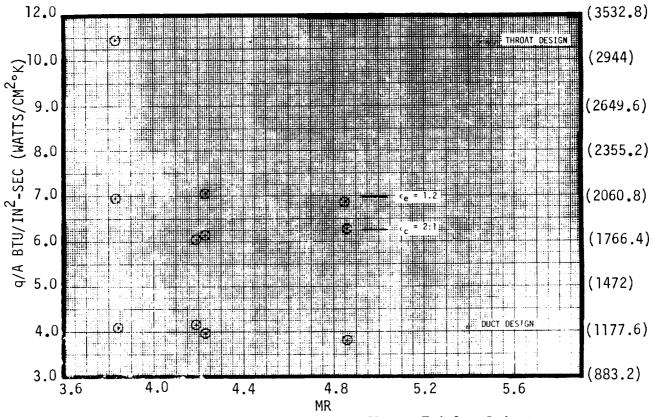


Figure 77. Preliminary Chamber Heat Flux - Triplet Injector

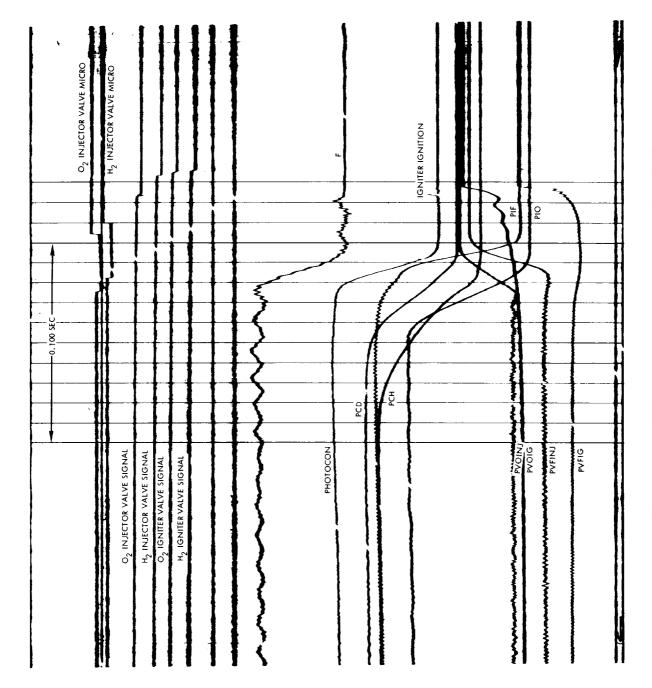


Figure 78, All Starts Were Virtually Identical

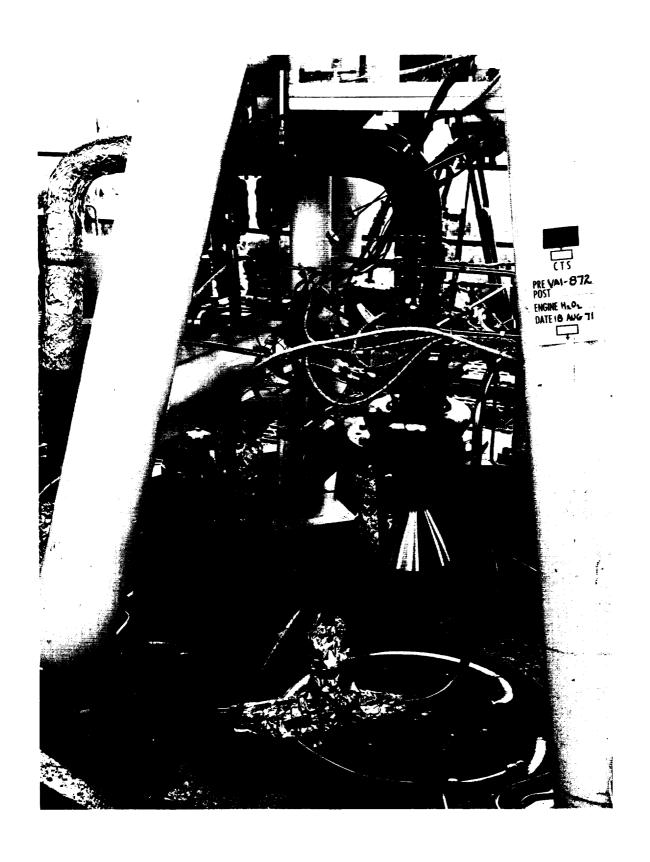


Figure 79. Duct Cooled Engine Installation with 12:1 Exit Area Nozzle

Combustion Performance with Duct Cooled Chamber (Sea Level) Table 10.

		Comments	repaired for Test 880	Copper duct collapsed on shutdown					
Duct Flow	Ordice Dia, in	9, 656, 10, 244,		-		0.0784	0, 125 (0, 317;		<del></del>
	9	Spark				<b>j</b> —			
	Duct	1.00.1		-1:Be-101		~	•		
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	Overall)	28.2	8. 3. 8. 8. 8. 8.	8 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	07.7		e. 16	9. 9. 9. 2. 3. 4. 9. 3. 4. 9.	9 - c c 8 8 8 8
	all) ni/sec	2526. 3 2543. 2	2486. b 2476. a	2478. 4 2466. 0 2462. 3	2462. 2 2458. 5		2344. 4	2365. 0 2348. 4 2352. 7	2343.7 2300.7 2352.4
	Overall) H/sec 117564	8288.1 8343.1	8158, 2 8126, 3	8042, 4 8078, 4	8078, 1 8066, 1		7641. 6	7759, 7 7704, 8 7718, 8	7689, 2 7577, 9 7717, 9
	MR (O/F)	2. 700	3, 574 3, 659	3,580	3, 665		4, 245	4, 194 4, 224 4, 221	4, 261 4, 580 4, 280 4, 280
Duct Coolant W	"DC X100 WH2	2	•	- <del>5</del>			- 15		
	R/sec	9. 015 0. 015	0, 014	0.015	0, 015		0.0059	0, 0059 0, 0059 0, 0059	0, 0059 0, 0059 0, 0059 0, 0059
Flow	ii 1b/sec k	0, 032	0.031	0.032	0.032		0. 013	0, 013 0, 013 0, 013	0, 013 0, 013 0, 013 0, 013
Igniter Flow	kk/sec	0, 014 0, 014	0, 014	0.014	0.014		0.0059	0, 0059 0, 0059 0, 0059	0, 0059 0, 0059 0, 0059 0, 0059
	0 <sub>2</sub>	0. 031 0. 031	0, 031 0, 031 0, 031	0, 031	0.031		0.013	0, 013 0, 013 0, 013	0, 013 0, 013 0, 013 0, 013
(2)		0, 354	0.286 0.284 0.286	0, 289	0, 287	leted)	0, 280	0. 285 0. 286 0. 285	0, 283 0, 263 0, 282 0, 285
finector Flow	H <sub>2</sub>	0.789 0.786	0.631 0.625 0.631	0.636 0.639 0.640	0.632	(To Be Completed)	0.617	0.629 0.630 0.629	0, 623 0, 580 0, 621 0, 628
	0 <sub>2</sub> kg/sec	0,980	1, 060 1, 076 1, 071	1, 070 1, 076 1, 103	1. 103	-	1, 208	1, 216	1, 223 1, 227 1, 227 1, 231
(	lb/sec	2,160	2. 336 2. 372 2. 362	2.358 2.372 2,431	2.431		2. 663	2. 681 2. 705 2. 700	2. 697 2. 706 2. 706 2. 714
H	lb <sub>f</sub> /sec kN/m <sup>2</sup>	1916.1	1903. 0 1914. 7 1913. 4	1905. 1 1912. 7 1949. 9	1941.6		1982. 3	2018. 2 2018. 9 2018. 9	2005, 8 1954, 0 2012, 0 2013, 3
d HD		277.9	276. 0 277. 7 277. 5	276.3 277.4 282.8	281.6		287.5	292. 7 292. 8 292. 8	290. 9 283. 4 291. 8 292. 0
7,000	Duration (sec)	0,15	2,00,11,00,00,00,00,00,00,00,00,00,00,00,	3.000	0.00	7 5 0 5	0.50	2. 0 3. 0 0. 15	-, 7, 4, 6, 0, 0 0 0 0 0 2,
	Test No.	871 872 873	8881 883 883 883	98 98 98 9 98 94 98 9	989				

(1) Tests for which data not shown did not yield reliable steady state performance data

Test Configuration: X408170 Triplet Injector (-3 ambient propellant temperature design, or -4 cold propellant temperature design, as noted X40816-3 Sanites State [Chamber A.J.A. -12 X40816 Duct (-1 ambient propellant temperature design or -2 cold propellant temperature design or -2 cold Flodyne Propellant temperature design or -2 cold Flodyne Propellant temperature design. as noted; Flodyne Propellant Valves Circle Saal (gatter Valves X408260 Catalytic (gatter, as noted.

(2) Includes duct flow.

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Figure 80. C\* Efficiency with Duct Cooled Chamber

A duct fabricated from OFHC copper was used for the initial tests because the copper material was readily available for duct fabrication. The long lead time for delivery of the Berylco-10 material, for which the duct was designed, delayed fabrication of the design duct configuration. The much lower structural strength of the OFHC copper was evidenced by collapse of the duct during shutdown transients. The OFHC copper duct, reaching approximately 1000°F (811.18°K) within 1-2 seconds of engine operation, collapsed inwards because the chamber pressure decay was more rapid than the pressure decay in the duct. The copper duct was prepaired by rolling out the collapsed region and was used in subsequent tests. No structural problems resulting from pressure differentials were encountered with the higher strength Berylco-10 duct.

Hot gas recirculation across the face of the inner oxidizer ring during initial tests at sea level resulted in some melting in this region. The recirculation was eliminated by replacing the inner oxidizer ring with an oxidizer ring with the impingement angle, with respect to the axial  $\rm H_2$  orifices, reduced from 20° to 10°, setting back the face of the inner oxidizer ring by 0.050 inches (.127 cm) and decreasing the igniter flow from 2 percent to 0.8 percent.

# 4.1.2.3 Altitude Firing Installation

With the catalytic igniter, injector, and duct fully checked out, the testing activity turned to the altitude firings. Both ambient and reduced temperature propellants were used in their tests. A photograph of the test installation is presented in Figure 81. The thin steel wall chamber was instrumented with thermocouples fore and aft as well as three locations circumferentially.

# 4.1.2.4 Test Summary

The test results are summarized in Table 11. The experimentally determined specific impulse, corrected to vacuum, is 432 seconds at a mixture ratio of 4.0 with a duct coolant flow of 32 percent for ambient temperature propellants. The specific impulse and C\* performance over a mixture ratio range from 3.0 to 4.8 is presented in Figure 82. The range of chamber pressures tested was limited because the initial test results at the nominal level of 300 psia ( $2068.5 \text{ KN/m}^2$ ) indicated that hot gas recirculation patterns were occurring at the outer rings of the injector causing higher heat transfer levels than anticipated. (The fixes described previously to the inner oxidizer ring resulted in no further overheating in this region.) During the latter tests, it was discovered that the protruding OFHC copper injection rings were imploded, resulting in some mixture ratio maldistribution. A transient negative pressure differential apparently occurred during startup across the injection rings for some of the ignition sequences which were used. Distortions of the internal flow passages may have been responsible for higher heat flux which occurred in local regions.

Numerous firings were conducted with the Berylco-10 duct with no difficulty. Figure 83 shows the duct after a 60-second firing and a total of 40 runs indicating its excellent condition. The corresponding nozzle temperature data are shown in Figure 84. All of this thermal data indicated strongly that excellent circumferential thermal control was being achieved



Figure 81. Test Installation of Altitude Thrust Chamber

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. . . . .

		P	C .		Injector	Flow			Igniter F
	Test <sup>(1)</sup>			C	) <sub>2</sub>	Н	1 <sub>2</sub>	0	2
Test No.	Duration (sec)	lb <sub>f</sub> /in <sup>2</sup>	kN/m <sup>2</sup>	lb/sec	kg/sec	lb/sec	kg/sec	lb/sec	kg/sec
944	0. 2	222.1	(1520)	1.823	(.8269)	0. 576	(, 261)	.0095	(. 0043)
945 946	0.5 no ign	223.1	(1538)	1.063	(.8207)	0.510	(.201)		,
947	1.0	213.9	(1474.84	) 1.687	(. 7652)	0.574	(.260)	. 0085	(.0039)
948	no ign		,						
949	0. 2				( 0004)	0.510	(.231)	. 0069	(.0031)
950	1.0	216.5	(1492.77		(.8904) (.8260)	0.452	(, 205)	.0076	(.0034)
951	1.0	201.3	(1387.96	) 1.021	(.0200)	0.432	(. 203)	•	
965 966	0.15 1.0	291.7	(2011.27	1) 2 728	(1.237)			0.013	(. 0059)
967	6. 0	295. 0	(2034, 03		(1, 244)	0.640	(.290)	1	
969	1.0	2,5,0	(2001, 00	,	,				
970	6.0	291.1	(2007.13	3) 2.771	(1.257)	0.638	(. 289)	ı	
971	0.15							1	
972	0.15	224 2			(1 100)	0.635	(, 288)	Į.	
974	2.0	286.0	(1971.97		(1.199)	0.558	(.253)	Į.	
975	1.0	277. <b>1</b> 279. 7	(1910.60 (1928.53		(1.234) (1.204)	0.592	(.269)	0. 014	(.0064)
976	2.0	292.6	(2017, 48	2) 2. 413	(1.095)	0.803	(.364)	1	
978	2.0	300.6	(2072.64	1) 2, 671	(1.212)	0.718	(.326)		
979	1.0	277.7	(1914. 74	1) 2. 711	(1.230)	0.571	(.259)	1	
980	2.0	290.6	(2003. 69		(1.203)	0.664	(.301)	İ	
983	0, 10		(======================================	,					
984	0.075							1	
985	0.050								
986	0.050								
987	0.050	305 7	(2038. 8	51 2 04 2	(1.300)	0.670	(, 304)	1	
988	2.0 2.0	295.7 307.6	(2120.9)		(1.224)	0. 732	(.332)	İ	
1000	10.0	304.1	(2096. 7		(1.223)	0.724	(.328)	1	
1001	290	301. 1	(==,,	., 2. 0 , .					
1	at 1.3	309.9	(2136.70	6) 2. 727	(1.237)	0.737	(.334)		
	10	306.9	(2116.0		(1, 238)	0.731	(.332)	ŀ	
	48	306.4	(2112,6		(1.242)	0.728	(.330)		
<b> </b>	100	310.5	(2140.9		(1, 282)	0.724	(, 328) (, 327)	1	
}	150	313.5		8) 2. 896	(1.314) (1.333)	0.720 0.724	(.328)		
1 1	200	316.5		7) 2. 938	(1.333)	0.724	(.331)		
	250 259	318.8 319.0		3) 2. 969 1) 2. 975	(1.347)	0.712	(. 323)	1	
	269 269	319.6		4) 2, 978	(1.351)	0.697	(.316)	1	
	279	320.3		7) 2. 984	(1.354)	0.682	(.309)	l	
1	287	320.8		2) 2. 990	(1, 356)	0.670	(.304)	7	

<sup>(1)</sup> Tests for which data not shown did not yield reliable steady state performance

cold propellant) X408163 Duct (-1 ambient propellant temperature design; -2 cold propellant

used for tests with cold propellant Flodyne Propellant Valves Circle Seal Igniter Valves

X408260 Catalytic Igniter

<sup>(2)</sup> Includes duct flow

<sup>(3)</sup> Specific impulse extrapolated to A<sub>C</sub>/A<sub>t</sub> = 40 for cold propellant tests with nozzle A<sub>C</sub>/A<sub>t</sub> = 12

Test Configuration: X408170 Triplet Injector (-3 ambient propellant temperature design; -4 cold temperature design used for tests with cold propellant

X408169 Thrust Chamber (A<sub>C</sub>/A<sub>t</sub> = 40, except A<sub>C</sub>/A<sub>t</sub> = 12 nozzle used for tests with cold propellant



Table 11. Altitude Test Summary

ow H	(2) 2	Duct Coolant WDC WXX	MR	C* (Over		F <sub>Vac</sub>	cuum	I sp Vacuum lb <sub>f</sub> sec	Ns <b>e</b> c	η <sub>ς</sub> * (Overall)	Comments
b/sec	kg/sec	$\frac{DC}{W_{H_2}}$ X100	O/F	ft/sec	,	lb <sub>f</sub>	N	lb <sub>m</sub>	kg		
0165	(. 0075)	26	3, 095	8237. 7	(2511)	990.2	(4404)	443. 8 <sup>(3</sup>	) (4352)	98.4	Cold Propellant
. 0161	(. 0073)	25	2,873	8378, 2	(2554)	954.3	(4245)	453.7	(4449)	99.7	
	,										
0183	(.0083)	25	3.729	7796.2	(2376)	1006.6	(4477)	435.8	(4273)	94.3	Į.
0156	(.0071)	30	3.914	7850.3	(2393)	919.3	(4089)	435.2	(4268)	95.6	<b>Y</b>
. 014	(.0064)	37	4.308	7641.0	(2329)	1425.7	(6342)	422.2	(4140)	94.0	Ambient Temp.
			4.219	7657.3	(2334)	1453.7	(6466)	426.6	(4182)	94.0	
- [		ļ	4.271	7494.2	(2284)	1470.3	(6540)	427.9	(4196)	92. 1	
		32									
			4.10	7654.5	(2333)	1424.5	(6336)	431.0	(4226)	93.6	
1		ļ	4.781	7414.7	(2260)	1382.7	(6150)	418.2	(4101)	92.7	
			4.409	7560.5	(2304)	1383.1	(6152)	422.6	(4144)	93. 4	
i i			2. 973	7980.5	(2432)	1416.0	(6298)	436.6	(4281)	95.4	
			3. 668	7782.4	(2372)	1478. 1	(6575)	432.6	(4281)	94.2	
ł			4.659	7422.6	(2262)	1381.4	(6144)	417.4	(4093)	92.4	
			3. 934	7689.9	(2344)	1459. 1	(6490)	436.5	(4280)	93. 6	Pulse Tests
				-							
		ļ									+
			4.194	7368.3	(2246)	1478.7	(6577)	416.5	(4084)	90.4	
-			3.619	7901.8	(2408)	1499.6	(6670)	435.4	(4270)	95. 5	
		1	3.655	7833. 1	(2388)	1496.6	(6657)	435.7	(4272)	94.7	
1		•	3.631	7882.8	(2403)	1508.2	(6708)	433.6	(4252)	95.3	
			3.644	7816.5	(2383)	1518.8	(6756)	437.2	(4287)	94.6	
- {			3.690	7791.4	(2375)	1531.5	(6812)	440.2	(4317)	94, 3	
l			3.828	7708. 2	(2349)	1562.5	(6950)	438.4	(4299)	93. 6	
l			3. 944	7641.2	(2329)	1587.6	(7062)	437.4	(4289)	93. 1	
1			3. 976	7616.7	(2322)	1603.6	(7133)	436. 2	(4277)	<b>92.</b> 9	
1			3. 987	7594.4	(2315)	1621.7	(7213)	436.7	(4282)	92. 6	
			4. 092	7623.0	(2324)	1613.7	(7178)	435.9	(4274)	93. 2	
ļ			4.185	7662.6	(2336)	1606.7	(7147)	435.4	(4270)	94. 0	
ţ			4. 285	7699.3	(2347)	1597.7	(7107)	434.7	(4263)	94. 7 95. 2	
•			4.368	7724.0	(2354)	1565, 8	(6965)	426.1	(4178)	70.4	

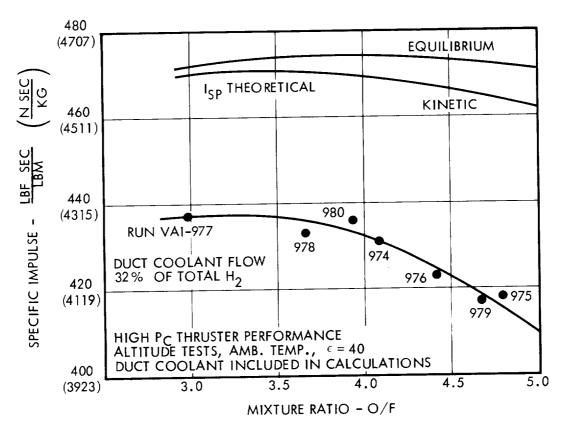
ld propellant

ests with

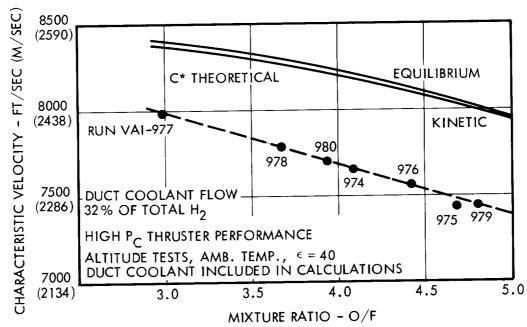
temperature design



1		



High Pressure Duct Cooled Thruster Performance - Specific Impulse



High Pressure Duct Cooled Thruster Performance - Characteristic Velocity

Figure 82. Altitude Engine Performance

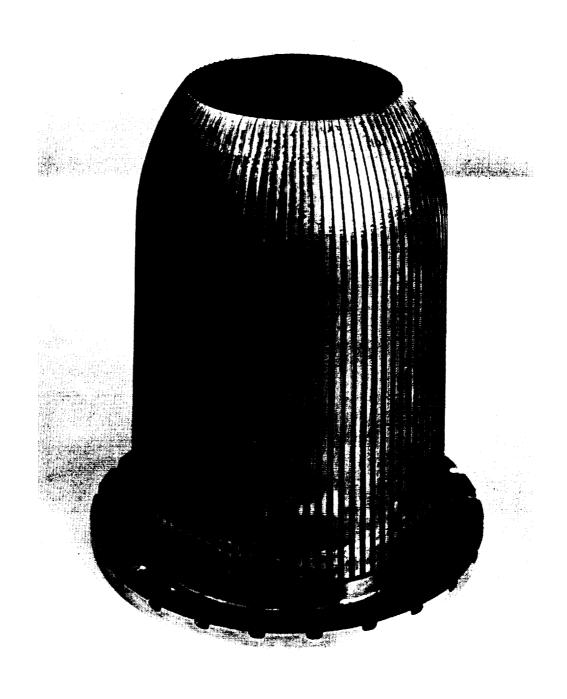


Figure 83. High Thrust Berylco-10 Duct After 60 Second Firings (40 Firings Total)

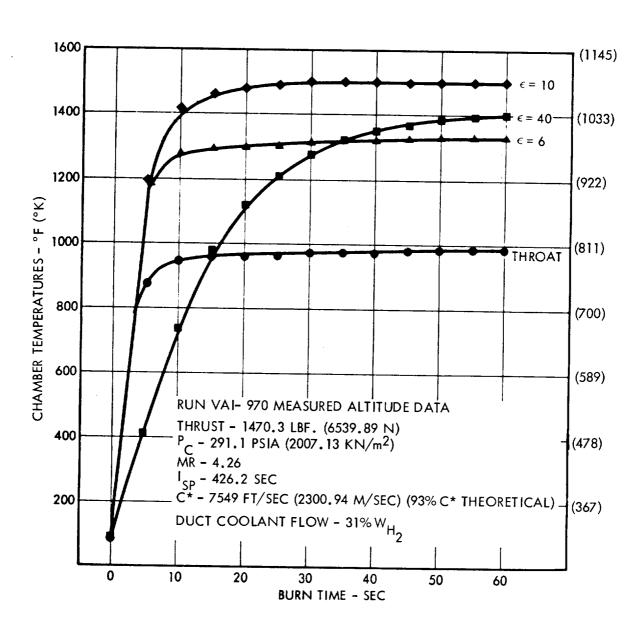


Figure 84. Thrust Chamber Temperatures for 60 Second Firing

with the duct concept. Following the 60 second data run, the analysis of the data indicated that the thruster could be run for the final duration run of the series, a 500 second test.

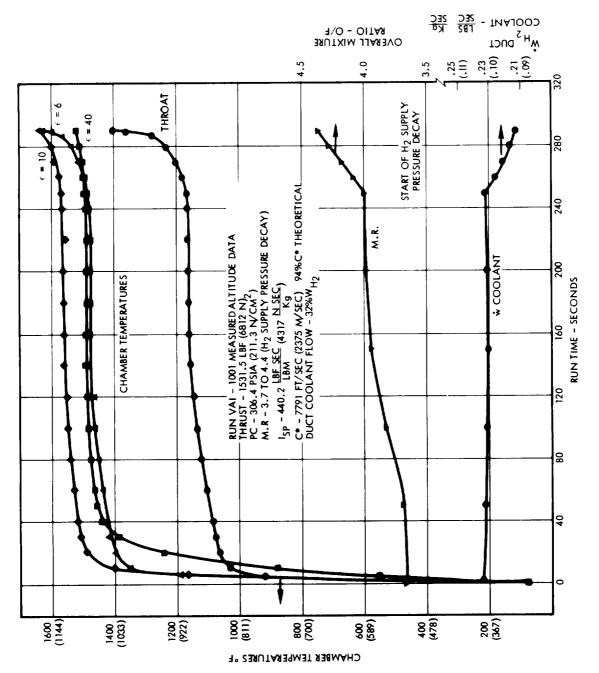
The 500 second duration test was to be made with ambient temperature propellants. Thrust chamber temperatures measured during the 290 second duration test with ambient temperature propellants are shown graphically in Figure 85. The highest temperatures of  $1570^{\circ}F(1128^{\circ}K)$  occur at a nozzle area ratio of 10. Steady state throat wall temperatures are  $1160^{\circ}F(900^{\circ}K)$ . These data are significant in that they do prove the viability of the concept. The run was started purposely at a reduced mixture ratio. At 40 seconds the decision was made to increase the mixture ratio to its nominal value. At 250 seconds the automatic H<sub>2</sub> pressure control regulator began to bottom out because of an apparent loss of supply pressure. The mixture ratio suddenly increased and the run was terminated. Except for the unfortunate sudden rise in mixture ratio, there is every indication that the 500 seconds would have been completed.

The long duration test, scheduled for 500 seconds duration was terminated after 290 seconds because of H2 supply pressure decay. The valve to the second H2 trailer, required for the second half of the test was found to have been locked in the closed position. At the onset of rapid H2 pressure decay and a nozzle throat temperature sudden rise in temperature the test was terminated. Examination of the hardware revealed localized surface melting damage to the outer oxidizer ring and a corresponding melt through of the duct.

The injector is shown in Figure 86 and 87 after the termination of the test. As is observed the injector proper is in very good condition. Damage occurred only on the outer oxidizer ring and fuel film coolant metering ring. The inner rings were in excellent condition as indicated.

The sudden rise in mixture ratio resulted in irreparable damage to the Berylco-10 duct, since the H<sub>2</sub> flow to the duct suddenly dropped by at least 25 percent. The duct is shown in Figures 88 and 89 post test the 290 second firing.

Experimental performance with cold propellants, with the X408170-4 triplet injector designed for cold propellants, compares closely with performance with ambient temperature propellants. The temperature conditions for the cold propellant tests are listed in Table 12. Propellant temperatures at the engine inlet were as low as  $380^{\circ}R$  (211°K) for the  $0_2$  and  $280^{\circ}R$ ((155°K) for the H<sub>2</sub>. Hydrogen flow to the chamber cooling duct ranged from 25-30 percent. The X408163-2 duct, designed with smaller groove flow areas, was used for the tests with cold propellant. Characteristic velocity and specific impulse performance for the cold propellant tests are presented in Figure 90. The measured characteristic velocity is somewhat higher than for the cold propellants. The nozzle area ratio for the cold propellant tests was 12. The measured specific impulse, corrected to vacuum, was extrapolated to that for a nozzle area of 40 by multiplying by the ratio of theoretical vacuum thrust coefficients and the nozzle divergence corrections for comparison to the ambient propellant temperature test results. (These corrections are 1.0517 and 1.0335, respectively.) The specific impulse compares closely



High Chamber Pressure Thruster Long Duration Test Data Figure 85.

Figure 86. Injector Face After 290 Second Firing



Figure 87. View of Injector After 290 Second Firing Showing Local Melting Adjacent to Outer Oxidizer Ring

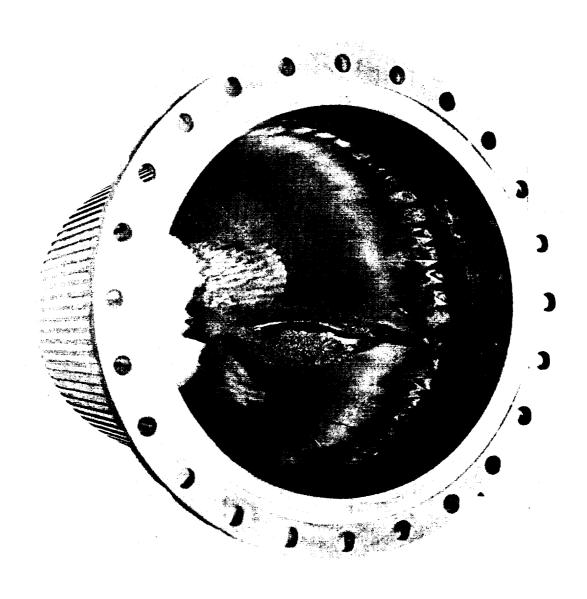
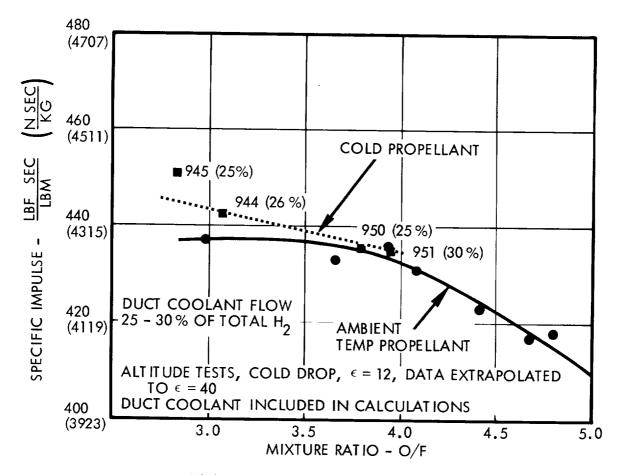


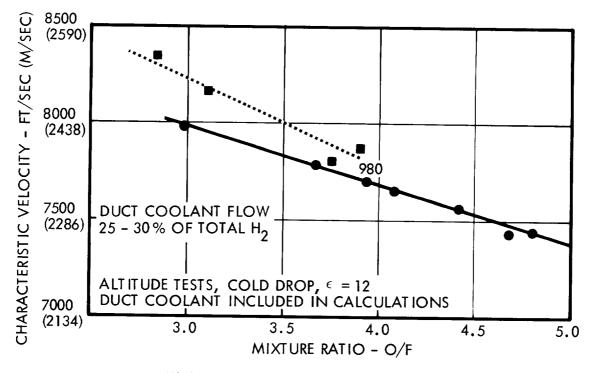
Figure 88. Internal View of Duct After 290 Second Firing



Figure 89. External View of Duct After 290 Second Firing



High Pressure Duct Cooled Thruster Performance - Specific Impulse



High Pressure Duct Cooled Thruster Performance - Characteristic Velocity

Figure 90. Comparison of Altitude Performance With Cold Propellant and With Ambient Temperature Propellant

Table 12. Cold Propellant Test Temperature (°R)(°K)

	Injec	tor <sup>(1)</sup>	Igni	ter
Test	02	H <sub>2</sub>	02	H <sub>2</sub>
945	380 (211)	280 (156)	380 (211)	280 (156)
947	360 (200)	310 (172)	492 (273)	280 (156)
950	460 (256)	300 (167)	450 (250)	320 (178)
951	370 (206)	320 (178)	498 (277)	420 (233)

<sup>(1)</sup> Measured at inlet to subsonic flow control orifices

for the different temperature conditions, with the cold propellant data being slightly higher because of the lower percent duct cooling flow. Experimental specific impulse at an overall mixture ratio of 3.9 is 435 seconds with 320°R (178°K)  $\rm H_2$  and 370°R (206°K)  $\rm O_2$  injection propellant temperature.

In all of the tests, except for two no ignition tests, the high response catalytic igniter functioned quite well. The two no ignitions may have been due to sequencing valve delays. The exact cause is uncertain at this time. Figure 91 shows a typical start ignition transient. Following the igniter flow variables and signals, it is seen that the igniter response is extremely fast. The igniter ignition flow was programmed ahead of the main thruster valves by  $\sim 25~\rm ms$ . It would appear that the igniter could safely be overlapped into the starting sequence without difficulty.

#### 4.1.2.5 Correlation with Predicted Modelling

The thermal data indicated, as a whole, a slightly higher temperature than was predicted by the design model. The overall trends were in near perfect agreement with the model. The minor discrepancies are felt to be associated with the duct gap not exactly matching the design valves.

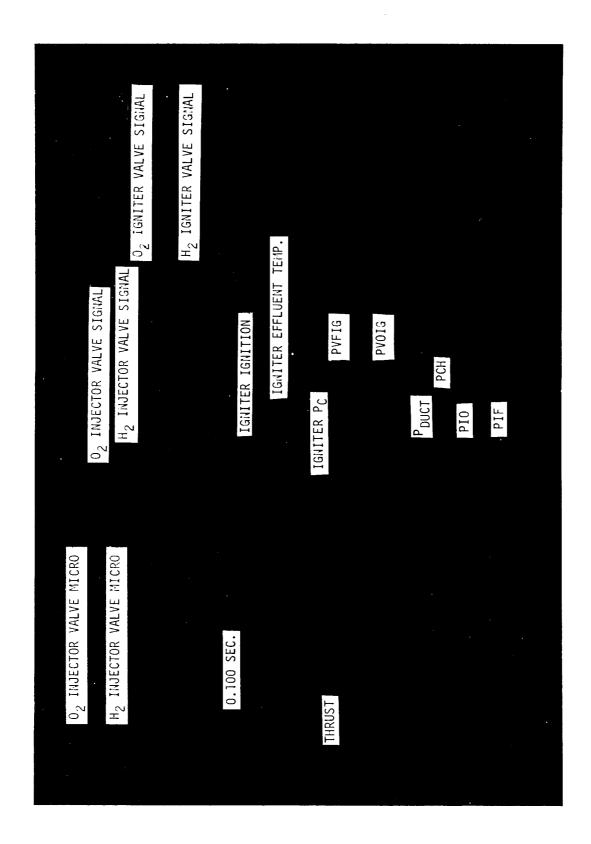


Figure 91. High Thrust Triplet Injector/Catalytic Igniter, Run VA1-961, MR4 - Start Transient

The performance correlation to the model proved to be outstanding as illustrated in Figure 92. The 32 percent coolant point is overlayed on the graph as originally derived from the MSFC data for the smaller TRW thruster. The 32 percent data from this program actually exceed the model predictions as shown. Of interest also is the fact that the original predicted roll-off at high MR did not actually turn out to be as severe.

## 4.1.3 Pulse Mode Tests

Runs 981 through 987 were conducted to partially investigate the pulse capability of the duct cooled thruster design. It should be kept in mind here that this program did not develop optimum valves for the thruster. The valves were TRW supplied and no effort was made to push these valves to their design limit. The igniter timing sequence was also not changed for these tests. The details of the pulse tests are summarized in Table 13.

Table 13. Pulse Mode Investigations

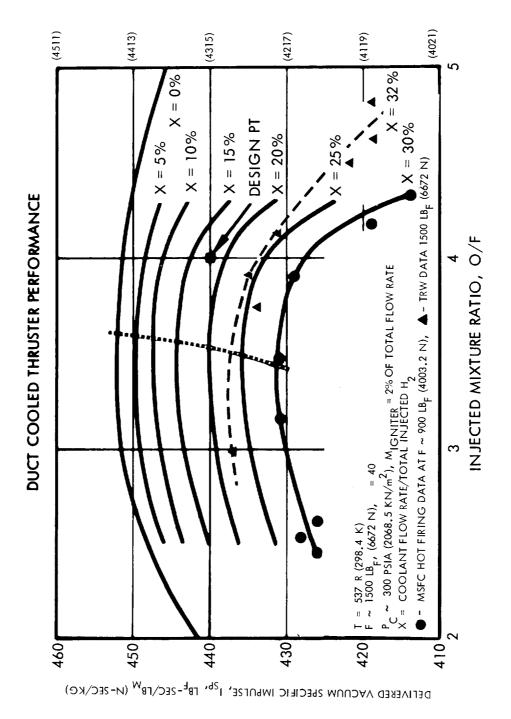
	Test Number	No. of <u>Pulses</u>	On-Time/ Off-Time	(lb-sec) (KG-sec) <u>Impulse</u>
VA1 Sea Level	981	1	0.100ms/ -	56.8 (2576.4)
VA1 Sea Level	982	1	0.100ms/ -	63.0 (2857.7)
Altitude	983	5	0.100ms/0.200ms	93.2 (lst pulse)(4227.6)
	984	5	0.075ms/0.100ms	68.4 (1st pulse)(3102.6)
	985	5	0.050ms/0.200ms	32.4,40.4,39.6,42.3,41.4(1469.7) (1832.5),(1796.3),(1918.7),(1877.9)
	986	5	0.050ms/0.100ms	35.3 (1st pulse) (1601.2)
	987	5	0.050ms/0.100ms	30.3 (1st pulse) (1374.4)

The approximate specific impulse for the initial 0.050 ms pulse of a pulse series, defined as the integrated thrust/integrated flow over the pulse, is approximately 328 seconds. During the 0.050 ms pulses, flow across the injection orifices is sonic. Following the  $P_{\rm C}$ -c\* dependance empirical data for both the low and high pressure performance data gives an average specific impulse result of 395 to 400 seconds for the 50 ms results.

Test number 988 was a 2.0 second test after the series of tests. This test showed a significant shift in overall fuel  $\triangle P$ . Physical examination of the injector showed the fuel rings to be collapsed in several locations.

Figure 93 shows the beginning of the pulse train for run VA1-983 with nominally 100ms pulses. The pulses were quite reproducible as seen by visual comparison and F-t integrations. The long igniter  $P_{\rm C}$  decay and rise times were found to be caused by a partially blocked instrumentation tube. The igniter temperature response is seen to be excellent on each startup.

Figure 94 shows typical data for the 50 ms pulse train of test VA1-985. The integrated impulse results show excellent reproducibility beyond the first pulse.



Performance Map - High Pressure Thruster Data Correlation With Original Model Predictions Figure 92.

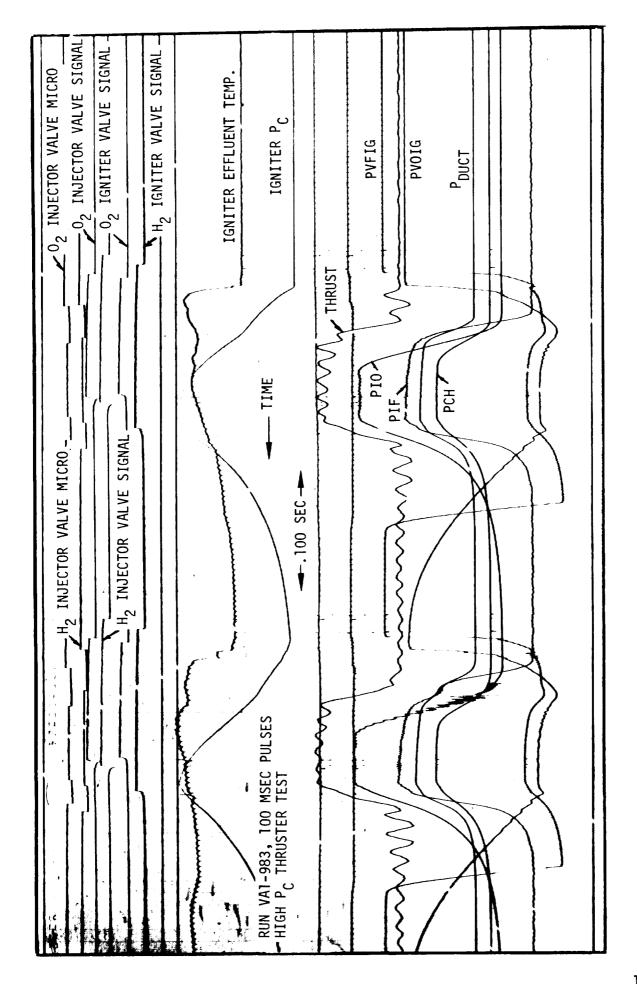


Figure 93. High Thrust Pulse Test VAl-983, 100 msec Pulses

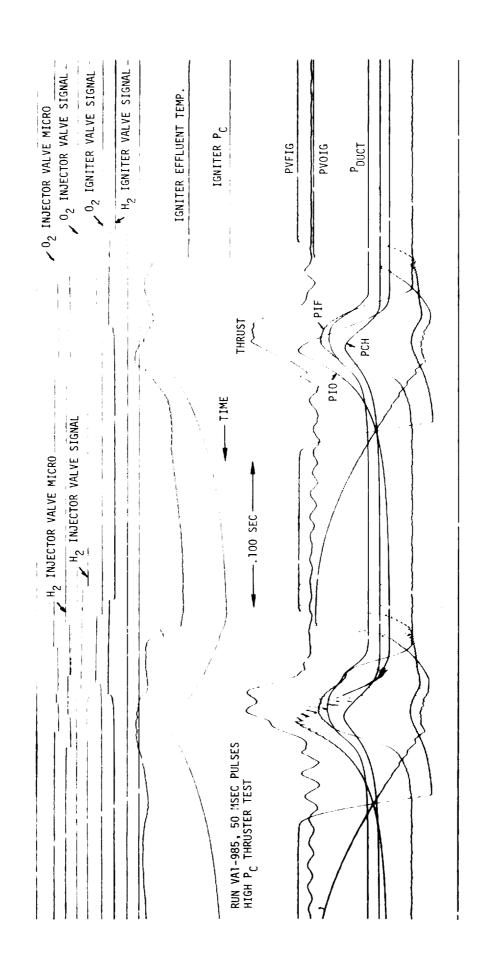


Figure 94. High Thrust Pulse Test VAl-985, 50 msec Pulses

Closer examination of Figures 93 and 94 provide a strong indication that the problem with the  $\rm H_2$  ring collapse is associated with a phenomena associated with pulsing. Note the pressure ringing in the second pulse on both sets of tests. Since it was not possible to place high frequency pressure response instrumentation in the thruster, the low frequency instrument outputs can only be received as an indicator of some difficulty. It appears that some reaction was taking place within the  $\rm O_2$  manifolding. It is speculated that there may have been popping occurring in the internal ring cavities also.

No damage was incurred in the  $0_2$  side. The  $0_2$  rings are shorter, stiff cylinders. The longer  $H_2$  cylinders did not apparently have sufficient wall thickness to length ratio to withstand the magnitude of this hypothesized popping effect.

## 4.2 IGNITER-ONLY OPERATION

Firing tests of a catalytic pilot bed igniter mounted in a complete 1500 lbf (6672 N) thruster assembly were performed to determine the MlB capability and cooling requirements, if any, for igniter-only mode of operation. The previous results of the thruster/system MlB analysis (Section 3.4.1) had indicated the significant advantages of igniter-only operation in achieving extremely low MlB for precise attitude control with minimum propellant usage. These tests were conducted to identify any problems caused by firing of the igniter in a full thruster assembly without main propellant flow.

# 4.2.1 <u>Test Description</u>

The thruster igniter-only evaluation firings were performed with a full scale cooling duct and thrust chamber, except that a nozzle with an area ratio of 12:1 rather than a 40:1 nozzle was used to allow the use of the HA5 altitude test stand. This test position was used for the low pressure thruster tests (described in Volume I), and was equipped with a thrust measurement stand of suitable range for igniter-only firing of the high pressure thruster. The same catalytic igniter used for all of the full thruster tests was installed in a copper dummy high Pc injector instrumented for chamber pressure and face temperature measurements. The igniter, dummy injector, duct, and thrust chamber are shown in Figure 95. Installation of the thruster assembly on the HA5 thrust mount is shown in Figure 96. This photograph was taken during chilldown for an igniter-only test with the thruster at initial temperatures below 300°R (167°K).

# 4.2.2 Summary of Results

The data resulting from the high pressure thruster igniter-only tests are presented in Table 14 and summarized as follows:

Test 365T: A 1/2-second checkout firing was conducted to verify ignition with ambient temperature propellants.

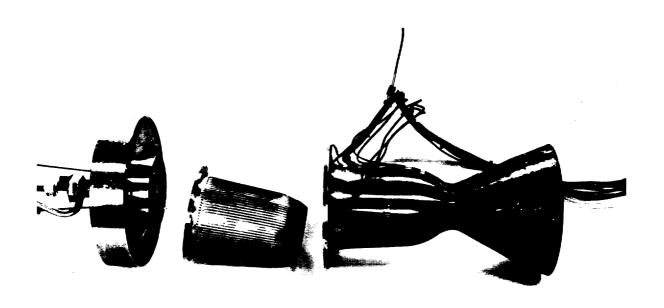


Figure 95. Igniter, Dummy Injector, Duct and Thrust Chamber



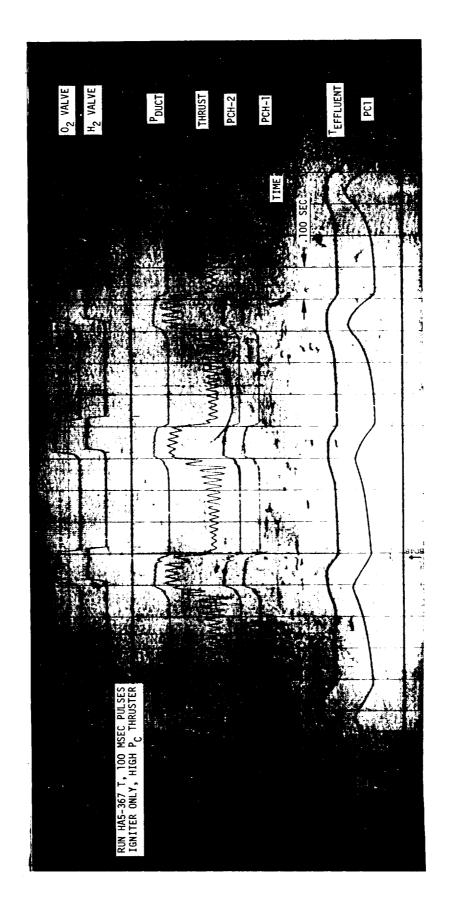
Figure 96. Installation of Thruster Assembly on HA5 Thrust Mount

Table 14. Igniter Only Pulse Tests

	Comments	Ambient temperature checkout	Extended duration test with duct coolant flow	Evaluation test with no duct coolant flow	Reduced igniter flow rates	Five pulses fired in succession at reduced igniter flow	Low temperature propellant ignition test	Increased igniter flows with low temperature propellants	Three pulses with low temperature propellants	Five pulses of 100 msec duration each -
Run	Duration (sec)	0.5	30	09	0.1	0.5	0.5	1.0	0.5	0.1
	(N)	120	4.76	101	36.9	33.8	80.1	100	108	86.3
Ц	Vaccum (1bf) (N)	26.9	21.9	22.8	8.29	7.59	18.0	22.5	24.2	19.4 86.3
	erature (°K)	767	288	286	167	288	83.9	79.4	79.4	91.1
Н2	Temperature (°R) (°K)	530	518	515	523	519	151	143	143	164
	Temperature (°R) (°K)	767	292	262	292	167	202	186	183	182
02	Temper (°R)	529	525	526	525	523	363	335	330	328
	H (kN/m <sup>2</sup> )	18.5	14.1	13.9	4.55	3.93	12.3	18.4	21.4	16.5
۵	(15f/in <sup>2</sup> ) (kN/m <sup>2</sup> )	2.69	2.04	2.02	0.66	0.57	1.78	2.67	3.10	2.39
					366к					

- Test 365Z: Duct coolant flow equal to one-half the total igniter flow rate was maintained during a 30-second duration igniter-only firing. Maximum chamber temperature was 446°F (503°K) measured at the thruster throat.
- Test 365Z16: A 60-second igniter firing was performed without any duct coolant flow. Maximum chamber temperature was 482°F (523°K) and stabilized, indicating that continuous igniter firings of 60 seconds or longer could be made without duct flow or other cooling of the main thruster being required.
- Test 366K: Igniter flow rates were reduced for this firing, resulting in a thrust level less than 10 lbf (44.5 N) and a thruster chamber pressure of less than 1 psia  $(6.9 \text{ kN/m}^2)$ .
- Test 366Z7: A series of five pulses was conducted at reduced igniter flows to verify repeatability of ignitions.
- Test 367K, R: Low temperature propellant tests were successfully performed with chilled thruster hardware. Igniter flows were increased for test 367R.
- Test 367S: Low temperature pulses of 500 msec duration each were conducted with low temperature propellants and thruster hardware.
- Test 367T: A series of five pulses was made with firing durations reduced to 100 msec on time. An oscillograph recording of this firing is reproduced in Figure 97, and indicates the repeatability of the pulses. No duct coolant was employed for any of the tests after 365Z, and the thruster and igniter hardware remained in excellent condition, as shown in Figures 98 and 99.

The results of the igniter-only high pressure thruster evaluations verified the previous analyses indicating that extremely low MIB values are feasible with this mode of operation. Test firings of up to 60-seconds steady-state operation without any chamber coolant required have indicated that igniter-only operation has many advantages for a reaction control thruster.



High Pc Thruster Test HA5-367T Igniter-Only Firings, 100 msec Pulses Figure 97.

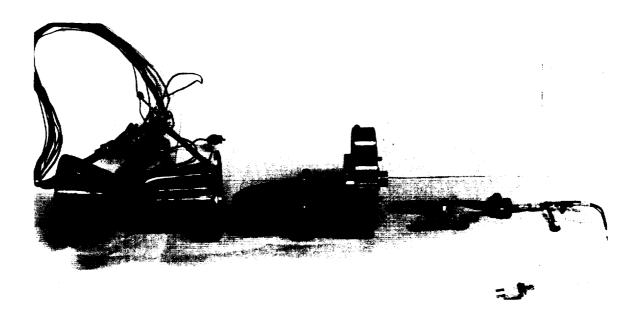


Figure 98. Thruster Hardware After Test



Figure 99. Igniter Hardware After Test 365Z

#### 4.3 ENVIRONMENTAL EFFECTS ON THRUSTER OPERATION

A series of firing tests of the high pressure thruster/igniter assembly was conducted to evaluate the effects on thruster operation of potentially degrading environmental exposures. For these tests, the analytical and experimental results of catalytic igniter environmental effects investigations (described in Volume I of this contract report) were utilized in selecting test conditions. The environmental conditions evaluated during this task were saturated air exposure at thruster hardware temperatures as low as 200°R (111°K).

### 4.3.1 Test Description

The environmental effects evaluation tests were performed with the 40:1 exit area ratio, altitude thrust chamber and the high pressure triplet injector. The thruster assembly was installed in the altitude test cell and chilled with liquid nitrogen. Exposure of the subzero temperature thruster hardware to saturated air (100 percent relative humidity) resulted in significant frost formation on the thruster surfaces, as shown in Figure 100.

The photograph shown in Figure 101 was taken during saturated air soaking of the chilled thruster prior to test firing. Steam from the altitude system was utilized as a source of saturated air for these tests. The effects of soak times of up to 30 minutes in saturated air with thruster temperatures as low as 200°R (111°K) were evaluated during this test series.

## 4.3.2 Summary of Results

Data from the thruster environmental effects tests are presented in Table 15 and the results are summarized as follows:

Tests 989 - 990: Initial checkout tests were conducted with only ambient air, normal humidity exposure of the thruster to establish baseline ignition response before environmental soak exposure.

Test 991: The first exposure was 6 minutes saturated air at reduced propellant and initial thruster temperatures, as indicated in Table 4-VII. A trickle purge of gaseous nitrogen was maintained through the igniter to prevent catalyst bed icing. The igniter fired satisfactorily on this test, but main thruster valves did not open. Inspection of the test installation revealed that liquid nitrogen from the conditioning system was dripping on the main thruster valves, causing ice formation which physically prevented external movement of the valve. The thruster valves were shielded from LN<sub>2</sub> exposure for all subsequent tests.

Test 992: A 30-minute saturated air soak at initial thruster temperatures similar to test 991 resulted in normal main thruster ignition with the  ${\rm GN}_2$  igniter trickle purge on.



Figure 100. Chilled High  $P_c$  Thruster - Environmental Effects Tests

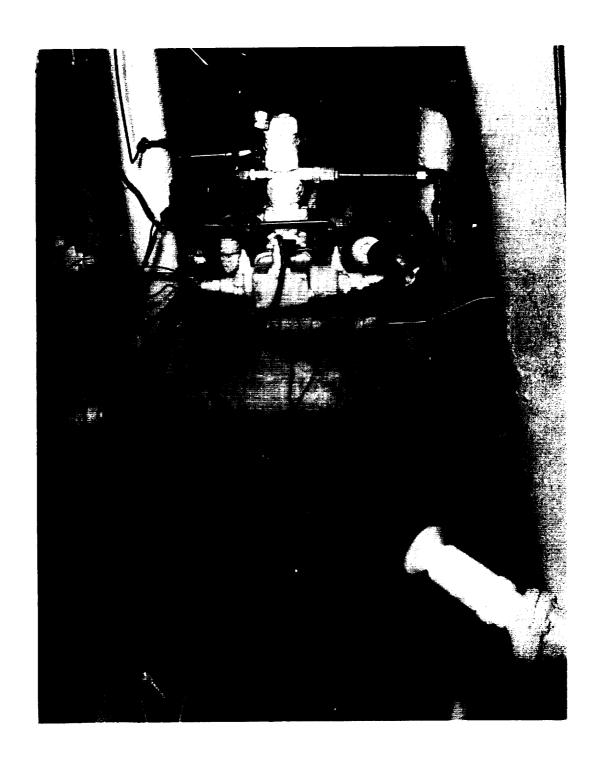


Figure 101. Saturated Air Soaking of Chilled Thruster Before Test Firing

Table 15. Environmental Effects on Thruster

Comments	initial checkout, good ignition	repeat to verify ignition before cold propellant tests	igniter fired, but main thruster valves did not open	main thruster ignition satisfactory after long soak	igniter fired, but main $0_2$ valve did not open $(2)$	checkout firing, good ignition	main thruster ignition, but no $Pch(3)$	no reaction in igniter, catalyst bed iced
Environmental Exposure	ambient air, normal humidity	ambient air, normal humidity	6 minutes saturated air, ${\sf GN}_2$ trickle purge on igniter	$30\ \text{minutes}$ saturated air. $\text{GN}_2$ trickle purge on igniter	<pre>15 minutes saturated air, without igniter purge</pre>	ambient air, normal humidity	28 minutes saturated air, ${\sf GN}_2$ trickle purge on igniter	27.5 minutes saturated air, without igniter purge
Thruster Initial mperature R) (°K)	283	291	198	202	113	289	121	112
Thruster Initial Temperature (°R) (°K)	509	523	356	364	203	521	217	201
H <sub>2</sub> Temperature (°R) (°K)	282	287	146	164	117	288	287	286
H2 Temper	508	516	262	295	210	518	517	514
0 <sub>2</sub> Temperature (°R) (°K)	284	288	158	204	166	289	288	287
Tempe (°R)	512	518	285	368	298	521	519	516
P <sub>CH</sub> (1bf/in <sup>2</sup> ) (kN/m <sup>2</sup> )	1891	1742	no main ignition	250.3 1726	no main ignition	203.6 1404	,	no igniter ignition
P <sub>C</sub> (1bf/in <sup>2</sup> )	243.8	252.7	no main	250.3	no main	203.6	(3)	no ignit
Test No. (VAI-)	989	066	166	992	993	766	995	966

(1) Liquid nitrogen from the thruster conditioning system was dripping on the main fire valves, resulting in icing which physically prevented valve movement; the valves were shielded for subsequent runs.

<sup>(2)</sup> Icing again physically prevented valve movement; this time only the main  $\mathbf{0}_2$  valve.

Main thruster ignition was attained, but Pch lines were frozen, verifying saturated air exposure at low hardware temperatures. (3)

Test 993: Saturated air soak of 15 minutes duration without GN<sub>2</sub> igniter purge did not affect the operation of the catalytic igniter; however, main thruster ignition was not achieved because of ice formation externally preventing opening of the main oxygen valve.

Test 994: A checkout test was again performed at ambient temperatures and normal humidity to verify baseline ignition characteristics.

Test 995: Main thruster ignition was achieved after 28 minutes of saturated air soak at a thruster initial temperature of 217°R (121°K). No chamber pressure rise was measured because of ice formation within the Pc lines. The frozen chamber pressure line indicated that saturated air did reach the injector face during the soak period. The igniter purge was maintained for this test firing.

Test 996: Repeating test 995 without the GN<sub>2</sub> trickle purge through the catalyst bed resulted in no igniter reaction, and thus no main thruster ignition would occur. Normal igniter operation was most likely prevented by ice formation within the catalyst bed.

The results of the high pressure thruster environmental tests indicated that frost or ice formation on the thruster surfaces did not affect ignition characteristics, as long as the catalyst bed itself was not allowed to ice up. Purging and/or heating of the catalyst bed is therefore recommended to insure high response reaction after extreme environmental exposures. This same recommendation was also made after analysis of the igniter only environmental effects tests described in Volume I of this contract report.

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5. CONCLUDING REMARKS

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#### 5. CONCLUDING REMARKS

The overall objectives of the high pressure thruster program tasks were to determine the delivered performance, operational requirements, and chamber cooling capability for a high performance, duct cooled, gaseous  $H_2/O_2$  attitude control thruster. Specific task efforts included:

- Design and fabrication of 1500 1bf (6672 N) thruster assemblies, including detailed performance, thermal, stress, and life analyses.
- Demonstration of minimum response catalytic ignition of the thruster.
- Experimental evaluation of thruster altitude performance and minimum impulse bit, overall operating characteristics, and duct cooled thrust chamber cooling capability.
- Determination of environmental effects on thruster operation, including exposure to saturated air at thruster temperatures as low as 200°R (111°K), resulting in significant ice formation.

The major conclusions from the high pressure thruster evaluations were:

- The raised post triplet injector designed for this program delivered an Isp of 432 lbf-sec/lbm (4248 N-sec/kg) with a cooled, flightweight thrust chamber ( $\varepsilon$ = 40).
- Repeatable pulse mode impulse bits as low as 30 lb-sec (1361 kg-sec) were demonstrated.
- Consistent ignitions were attained at propellant temperatures as low as -280°F (156°K).
- A 25 to 40 ms ignition is possible with a catalytic ignitor.
- Cooling capability of the duct cooled thrust chamber was demonstrated by extended duration firings attaining steady-state thruster temperatures.

The experimental results of this program have demonstrated the high performance capability of gaseous hydrogen-oxygen thrusters. The effectiveness of the duct cooling concept was proven for the lightweight thruster design. The analytical cooling design techniques provide a workable, conservative engineering design approach for the concept. The same mixing model approach also predicts the performance with reasonable accuracy.

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#### REFERENCES

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**APPENDICES** 

#### APPENDIX A

#### CALCULATION OF C\* EFFICIENCY

The index of injector performance used in the experimental program was the corrected C\* efficiency. This parameter was calculated by two independent methods, one based on measurement of chamber pressure and the other on measurement of thrust. Details of the computational procedures and of the applied corrections are given in the following sections. The procedures and nomenclature format are essentially those as developed in NASA sponsored programs at Rocketdyne.

### 1.0 CHAMBER PRESSURE TECHNIQUE

Characteristic velocity efficiency based on chamber pressure is defined by the following:

$$\eta_{C^*} = \frac{(P_c)_c (\Lambda_t)_{eff} g_c}{(\mathring{w}_T) (C^*)_{theo}}$$
(A-1)

where

 $(P_c)_0$  = stagnation pressure at the throat

 $(A_t)_{eff}$  = effective thermodynamic throat area

 $g_c$  = conversion factor (32.174 lbm-ft/lbf-sec<sup>2</sup>)

WT = total propellant weight flow rate

(C\*) theo = theoretical characteristic velocity based on shifting equilibrium

Values calculated from Equation (A-1) are referred to as "corrected" C\* efficiencies, because the factors involved are obtained by application of suitable influence factor corrections to measured parameters. Stagnation pressure at the throat is obtained from measured static pressure at start of nozzle convergence by assumption of isentropic expansion, effective throat area is estimated from measured geometric area by allowing for geometrical radius changes during firing and for nonunity discharge coefficient, and chamber pressure is corrected to allow for energy losses from combustion

gases to the chamber wall by heat transfer and friction. Equation (A-1) may therefore be written as follows:

$$\eta_{C^*} = \frac{{}^{P}_{C} {}^{A}_{t} {}^{g}_{c} {}^{f}_{p} {}^{f}_{TR} {}^{f}_{DIS} {}^{f}_{FR} {}^{f}_{HL} {}^{f}_{KE}}{(\dot{w}_{o} + \dot{w}_{f}) {}^{(C^*)}_{theo}}$$
(A-2)

where

P<sub>c</sub> = measured static pressure at start of nozzle convergence, psia

 $A_{t}$  = measured geometric throat area, in<sup>2</sup>

 $g_c = \text{conversion factor } (32.174 \text{ lbm-ft/lbf-sec}^2)$ 

wo = oxidizer weight flow rate, lb/sec

w<sub>f</sub> = fuel weight flow rate, 1b/sec

(C\*) theo = theoretical C\* based on shifting equilibrium calculations, ft/sec

f
p = influence factor correcting observed static
pressure to throat stagnation pressure

f<sub>TR</sub> = influence factor correcting for change in throat radius during firing

f<sub>DIS</sub> = influence factor correcting throat area for effective discharge coefficient

fFR = influence factor correcting measured chamber pressure for frictional drag of combustion gases at chamber wall

f<sub>HL</sub> = influence factor correcting measured chamber pressure for heat losses from combustion gases to chamber wall

fKE = influence factor correcting C\* values to account
for finite chemical reaction rates

## 1.1 PRESSURE INFLUENCE FACTOR (fp)

Measured static pressure at start of nozzle convergence is converted to stagnation pressure at the throat by assumption of effectively no

combustion in the nozzle and application of the isentropic flow equations, with contraction ratio  $(A_c/A_t)$  and shifting-equilibrium specific heat ratios  $(\gamma)$ . Frozen-equilibrium specific heat ratios usually make the influence correction factor about 1/2 percent larger. Hence the value employed with shifting-equilibrium is the more conservative. Figure A-1 shows the influence factor as a function of contraction ratio.

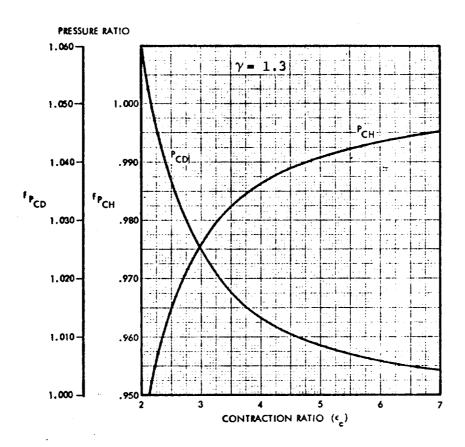


Figure A-1. Momentum Correction

## 1.2 THROAT RADIUS INFLUENCE FACTOR $(f_{TR})$

Temperature gradients produced in the solid metal nozzle wall result in thermal stresses which affect throat radius, with the result that the geometric throat diameter ambient measurement is not the same as that which exists during firing.

In the chamber type employed during the experimental effort (i.e. thin throat wall thickness), the throat area change is computed from the thermal growth of the throat based on temperature changes from ambient temperature. The change in throat area can be written as:

$$A_{th} = \frac{\pi}{4} (2 + \alpha \Delta T) (\alpha \Delta T) D^2$$
 (A-3)

where

 $\Delta A^*$  = change in throat area due to thermal growth

 $\alpha$  = average thermal expansion coefficient

 $\Delta T$  = temperature rise from ambient conditions

D = throat diameter at ambient conditions

The throat area correction factor is as follows:

$$f_{TR} = 1 + \frac{\Delta A_{th}}{A_{th}}$$
$$= [1 + \alpha \Delta T]^{2}$$
(A-4)

The thermal expansion coefficient for copper and CRES is  $\alpha_{\rm Cu} = 9.8 \times 10^{-6}$  in/in-°F, assuming an ambient temperature of 70°F, the throat area correction factor becomes

$$f_{TR} = [1 + 9.8 \times 10^{-6} (T_{th} - 70)]^2$$
 (A-5)

This equation was used to generate the curve in Figure A-2.

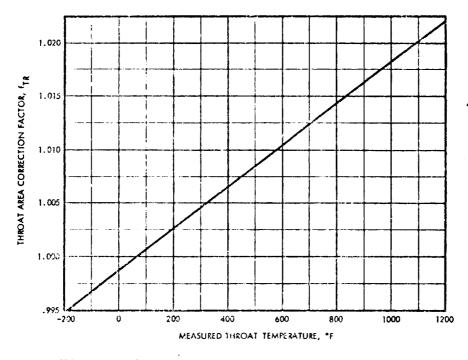


Figure A-2. Throat Area Correction Factor

### 1.3 THROAT DISCHARGE COEFFICIENT INFLUENCE FACTOR (fDIS)

The discharge coefficient is defined as the ratio of actual flow rate through the throat to the theoretical maximum, based on geometric throat area and ideal, uniform, one-dimensional flow with no boundary layer. The discharge influence coefficient may be estimated in two ways: one based on calculations made from a theoretical, inviscid flow model of combustion products, and the other based on a correlation of results obtained in various experimental study results of air flow through nozzles of similar geometry.

### 1.3.1 Theoretical Model

Total mass flow rate is given by

$$\dot{m} = \int_{\Omega}^{A} \rho V dA \qquad (A-6)$$

where:

p = gas density

V = gas velocity

A = cross-sectional area

Theoretical maximum flow rate at the throat is

$$\dot{m}_{\text{max}} = \int_{0}^{A_{\text{t}}} \rho \star V \star dA \qquad (A-7)$$

where:

 $A_{+}$  = geometric area of the throat

P\* = sonic gas density

V\* = sonic gas velocity

For ideal, uniform, parallel flow, Equation (D-7) becomes

$$\dot{m}_{max} = \rho * V * A_{t}$$
 (A-8)

The discharge coefficient is then

$$C_{D} = \frac{\dot{m}}{\dot{m}_{max}} = \int_{0}^{A} \left(\frac{\rho}{\rho \star}\right) \left(\frac{V}{V^{\star}}\right) \left(\frac{dA}{A_{t}}\right)$$
 (A-9)

### 1.3.2 Empirical Value

Experimental conical nozzle discharge coefficients obtained with air by various investigators are plotted in Figure A-3 against the indicated geometric parameter. Data sources also are listed in Figure A-3.

The values obtained by both methods are found to be in excellent agreement.

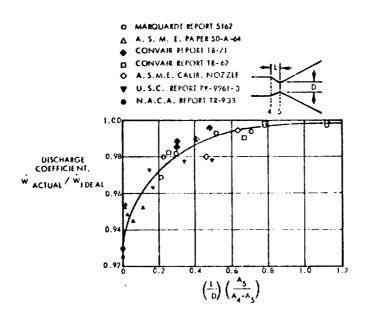


Figure A-3. Conical Nozzle Discharge Coefficient

# 1.4 FRICTIONAL DRAG INFLUENCE FACTOR ( $f_{FR}$ )

Calculations of C\* based on chamber pressure are concerned with chamber phenomena up to the nozzle throat. Drag forces to this point are small enough to be considered negligible, so that the factor  $\mathbf{f}_{FR}$  may be taken to be unity.

# 1.5 ENERGY LOSS INFLUENCE FACTOR ( $\mathbf{f}_{\mathrm{HL}}$ )

Chamber pressure and thrust are decreased by heat transfer from the combustion gases to the walls of a thrust chamber. This enthalpy loss is substantially reduced in ablative chambers and is effectively recovered in a regeneratively cooled chamber.

The effect on C\* of enthalpy loss by heat transfer can be estimated from a loss of chamber enthalpy. This is determined from a two station energy balance, one at the start of nozzle convergence and the other at the throat.

$$1/2 V_c^2 + H_c = 1/2 V_t^2 + H_t + \dot{Q}_{conv}$$
 (A-10)

where:

 $V_c$  = gas velocity at chamber exit

 $V_t$  = gas velocity at nozzle throat

 $H_c = gas enthalpy at chamber exit$ 

H<sub>+</sub> = gas enthalpy at nozzle throat

Q = heat loss in nozzle convergence

Velocity at the throat is given by:

$$V_{t} = [V_{c}^{2} + 2(H_{c} - H_{t} - \dot{Q}_{conv})]^{1/2}$$
 (A-11)

With negligible nozzle inlet velocity

$$V_t = [2(H_c - H_t - \dot{Q}_{conv})]^{1/2}$$
 (A-12)

Logarithmic differentiation of Equation (A-12) gives

$$\frac{dV_{t}}{V_{t}} = 1/2 \frac{d (H_{c} - H_{t} - Q_{conv})}{(H_{c} - H_{t} - Q_{conv})} = 1/2 \left(\frac{d H_{c} - dH_{t}}{H_{c} - H_{t} - Q_{conv}}\right) (A-13)$$

Substitution of enthalpy definition into Equation (A-13) gives:

$$\frac{dV_t}{V_t} = 1/2 \left( \frac{c_{pc} dT_c - c_{pt} dT_t}{H_c - H_t - Q_{conv}} \right)$$
(A-14)

With constant  $C_p$  between the two stations,

$$\frac{dV_{t}}{V_{t}} = 1/2 \left( \frac{c_{p} dT_{c}}{H_{c} - H_{t} - Q_{conv}} \right) \left( 1 - \frac{dT_{t}}{dT_{c}} \right)$$
(A-15)

If the specific heat ratio,  $\gamma$ , is assumed constant,

$$\frac{dT_t}{dT_c} = \frac{T_t}{T_c} \tag{A-16}$$

Substituting Equation (A-16) into Equation (A-15), replacing differentials by incrementals, and noting that C\* is proportional to gas velocity at the throat gives:

$$\frac{\Delta V_{t}}{V_{t}} = \frac{\Delta C^{\star}}{C^{\star}} = 1/2 \left( \frac{c_{p} \Delta T_{c}}{H_{c} - H_{t} - \dot{Q}_{conv}} \right) \left( 1 - \frac{\Delta T_{t}}{T_{c}} \right) \tag{A-17}$$

Total heat loss to the chamber walls, in Btu per pound of propellant, is obtained by summation of observed heat fluxes over the appropriate areas:

Heat loss = 
$$\frac{\Sigma(q/A)}{\hat{w}_T}$$
 (A-18)

where:

q/A = experimentally observed heat flux

A = area applicable to each q/A value

 $\dot{w}_{T}$  = total propellant flow rate

If this heat loss is equated to the change in enthalpy of the gas in the combustion chamber,  $c_p \Delta T_c$ , then substitution in Equation (8.19) gives:

$$\frac{\Delta C^*}{C^*} = 1/2 \left[ \frac{\sum (q/A)A}{\dot{w}_T} \right] \left[ \frac{1 - (T_t/T_c)}{II_c - II_t - Q_{conv}} \right]$$
(A-19)

The applicable influence factor is:

$$f_{HL} = 1 + \frac{\Delta C^*}{C^*} = 1 + 1/2 \left[ \frac{\sum (q/A)A}{\dot{w}_T} \right] \left[ \frac{1 - (T_t/T_c)}{H_c - H_t - Q_{conv}} \right]$$
 (A-20)

An alternate expression can be obtained from the basic C\* definition:

$$C^{\star} = \frac{\sqrt{RT_{C}}}{\Gamma} \tag{A-21}$$

Logarithmic differentiation of this yields:

$$\frac{dc^*}{c^*} = \frac{1}{2} \frac{dT_c}{T_c} \tag{A-22}$$

Substituting incrementals from differentials in Equation (A-22) gives:

$$\frac{\Delta_c^*}{c^*} = \frac{1}{2} \frac{\Delta T_c}{T_c} \tag{A-23}$$

Equating  $\Delta T_c$  with the heat loss from Equation (A-18) results in the following:

$$\frac{\Delta_c^*}{c^*} = \frac{1}{2} \left[ \frac{\Sigma(q/A)A}{\dot{w}_t} \right] \left[ \frac{1}{c_p T_c} \right]$$
 (A-24)

The applicable influence factor is:

$$f_{HL} = 1 + \frac{1}{2} \left[ \frac{\Sigma(q/A)A}{\dot{w}_{+}} \right] \left[ \frac{1}{c_{p} T_{c}} \right]$$
 (A-25)

where

c<sub>p</sub> = specific heat at constant pressure

Although derived independently it can be shown that these two expressions, Equations (A-20) and (A-25), are nearly equivalent.

## 1.6 INFLUENCE FACTOR FOR CHEMICAL KINETICS $(f_{KE})$

The effect of finite chemical reaction rates is to produce a C\* less than the corresponding theoretical equilibrium values. A TRW Systems Group developed one-dimension nonequilibrium reacting gas computer program was employed with reaction rate constants selected for the propellant system. The fluid mechanical and chemical equations were integrated from the inlet section by an implicit technique.

#### 2. CALCULATIONS BASED ON THRUST

The alternate determination of C\* efficiency is based on thrust:

$$\eta_{C^*} = \frac{F_{\text{vac}} g_c}{(C_F)_{\text{vac}} \dot{w}_T C^*_{\text{theo}}}$$
 (A-26)

where:

 $F_{\text{vac}}$  = measured thrust corrected to vacuum conditions by the equation:  $F_{\text{vac}} = F + P_{\text{a}}A_{\text{e}}$ 

F = measured thrust, 1bf

P = ambient pressure, psia

 $A_e$  = area of nozzle exit, in<sup>2</sup>

 $g_c$  = conversion factor (32.174 lbm-ft/lbf-sec<sup>2</sup>)

 $(C_F)_{vac}$  = theoretical shifting thrust coefficient (vacuum)

w<sub>T</sub> = total propellant flow rate, lbm/sec

theo = theoretical shifting-equilibrium characteristic velocity, ft/sec

Values of vacuum thrust are obtained by applying corrections to sea-level measurements. With these values, which include allowances for all important departures from ideality, theoretical thrust coefficients may be used for calculation of  $C^*$ .  $C_F$  efficiency is taken as 100 percent if there is no combustion in the nozzle, if chemical equilibrium is maintained in the nozzle expansion process, and if energy losses from the combustion gases are accounted for.

Applicable influence factors for measured thrust are specified in the following equation:

$$\eta_{C*} = \frac{(F + P_a A_e) g_c \phi_{FR} \phi_{DIV} \phi_{HL} \phi_{KE}}{(C_F)_{theo} (W_o + \dot{w}_f) (C^*)_{theo}}$$
(A-27)

where:

F = measured thrust, 1b<sub>f</sub>

P = ambient pressure, psia

A = area of nozzle exit, in<sup>2</sup>

g<sub>c</sub> = conversion factor (32.174 lbm-ft/lbf-sec<sup>2</sup>)

(C<sub>F</sub>)<sub>theo</sub> = theoretical shifting thrust coefficient (vacuum)

w = oxidizer weight flow rate, 1bm/sec

w<sub>f</sub> = fuel weight flow rate, 1bm/sec

(C\*)theo = theoretical shifting equilibrium characteristic
velocity, ft/sec

 $\phi_{FR}$  = influence for frictional losses

 $\phi_{DIV}$  = influence factor for nozzle divergence

HL = influence factor for heat losses to chamber and nozzle walls

pKE = influence factor correcting C\* and C<sub>F</sub> values to
account for finite chemical reaction rates

The influence factors in Equation (A-27) are applied to vacuum thrust  $(F + P_a A_e)$  instead of to measured site thrust (F) because, for convenience, the factors are readily calculated as changes in efficiency based on theoretical vacuum parameters. The total influence factor is then of the form  $\Delta F/F_{vac}$ .

Implicit in the use of theoretical  $C_F$  values are corrections to geometric throat area and to measured static chamber pressure at start of nozzle convergence. Therefore, calculation of corrected  $C^*$  efficiency from thrust measurement includes all the previously described corrections plus an additional one to account for nonparallel nozzle exit flow. However, because  $(C_F)_{theo}$  is essentially independent of small changes to chamber pressure and contraction ratio which are involved in corrections to  $P_c$  and  $A_t$ , these corrections are of no practical significance in calculation of  $C^*$  from thrust measurements.

# 2.1 INFLUENCE FACTOR FOR FRICTIONAL DRAG ( $\phi_{FR}$ )

This factor corrects for energy losses caused by viscous drag forces on the thrust chamber walls. Its magnitude is estimated by a boundary layer analysis utilizing the integral momentum equation for turbulent flow, which accounts for boundary layer effects from the injector to the nozzle exit by suitable description of the boundary layer profile and local skin friction coefficient. A computer program is used to carry out a numerical

integration of the equation, including effects of pressure gradient, heat transfer, and surface roughness. The program requires a potential nozzle flow solution obtained from variable-property, axisymmetric method of characteristics calculation of the flow field outside the boundary layer; corresponding properties for the subsonic combustion chamber flow field are also calculated.

# 2.2 INFLUENCE FACTOR FOR NOZZLE DIVERGENCE ( $\phi_{ m DIV}$ )

The one-dimensional theoretical performance calculations assume that flow at the nozzle exit is uniform and parallel to the nozzle axis. The influence factor,  $\phi_{\rm DIV}$ , allows for nozzle divergence (i.e., for nonaxial flow) and for nonuniformity across the nozzle exit plane. It is calculated by a computer program which utilizes the axisymmetric method of characteristics for a variable-property gas. Computation begins with a transonic input near Mach 1, providing a characteristic line for use in the analysis of the supersonic portion of the nozzle. The resulting pressures are integrated over the given geometry to give the geometric efficiency.

## 2.3 INFLUENCE FACTOR FOR HEAT LOSS ( $\phi_{ m HL}$ )

To obtain the heat loss influence factor from measured thrust the approach is identical to that taken previously from the pressure measurement, except that the nozzle losses must also be included. With constant specific heat and gamma from start of nozzle convergence to exit, Equation (A-20) becomes

$$\phi_{HL} = 1 + \frac{1}{2} \left[ \frac{\Sigma \left( \frac{G}{A} \right) A}{\dot{w}_{T}} \right] \left[ \frac{1 - T_{e}/T_{c}}{H_{c} - H_{e} - \dot{Q}_{nozzle}} \right]$$
(A-28)

when "e" corresponds to the exit condition, and the summation occurs over the entire combustion.

An alternate can also be derived as in Equation (D-25). This equation becomes

$$\phi_{\text{HL}} = 1 + \frac{1}{2} \left[ \frac{(q/A)A}{\dot{w}_{\text{T}}} \right] \left[ \frac{1}{c_{\text{p}} T_{\text{e}}} \right]$$
(A-29)

## 2.4 INFLUENCE FACTOR FOR CHEMICAL KINETICS ( $\phi_{KE}$ )

The effect of finite chemical reaction rates is to produce a C\* and  $C_{\rm p}$  less than the corresponding theoretical equilibrium values. A TRW Systems Group developed one dimensional nonequilibrium reacting gas computer program was employed with reaction rate constants selected for the FLOX methane-ethane blend propellant system. The fluid mechanical and chemical equations were integrated from the inlet section by an implicit technique.

### APPENDIX B

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